



Bearings for Screw Drives

Axial angular contact ball bearings
Needle roller/axial cylindrical roller bearings
Accessories

added competence

"added competence" for your success

With their forward-looking bearing arrangement solutions for feed spindles, main spindles, rotary tables and linear guidance units, INA and FAG have been at the forefront of the world market for decades. Nevertheless, bearing components alone are often no longer the decisive factor for the success of these machine subsystems.

Indeed, our customers are continuing to benefit directly from significant performance improvements and unique selling points thanks to our "ready-to-fit" products, since these follow the efficient basic concept: unpack, screw mount, use. In order to optimise the entire machine tool system, however, it is also becoming ever more important to integrate important functions such as measurement, sealing, lubrication, braking etc. in the components themselves. This intellectual approach is fulfilled comprehensively by the new concept "added competence" in the Business Unit Production Machinery since it attaches central importance to systems solution thinking for the bearing, bearing position and entire system. This means that you can now access a product range that gives optimum coverage for all your applications in the machine tool.

In addition, there is increasingly frequent usage of direct drives and mechatronic solutions in machine tools. We have therefore incorporated IDAM – INA Drives & Mechatronics – as a further strong partner in our provider network. In this way, we can now supply you from a single source with not only bearing elements but also components precisely matched to the drive system.

This opens up completely new technical and economic design possibilities for your requirements as well as significant advantages in the time and process chain.

In terms of products, we can offer you a comprehensive, precisely balanced range, precision technology and top product quality. In order to match the pulse of your developments as closely as possible, we also have a worldwide network of engineers and service and sales technicians working for you and ensuring that we maintain close contact with you in your own location.

We are therefore confident that we have the right product for you, from a robust individual component right through to the defining high end system solution.

Perhaps you would like to talk to us about this?



Bearings for screw drives

Matrix for preselection of bearings

The application-oriented matrix allows rapid preselection of the suitable bearing.

Axial angular contact ball bearings

6

4

Axial angular contact ball bearings are precision bearings for screw drive bearing arrangements. Depending on the series, they can support radial forces as well as axial forces in one or both directions. Contact seals protect the rolling element system against contamination and moisture. For higher speeds, minimal gap seals can be used.

The bearings are available with and without fixing holes in the outer ring. Bearings with holes are screw mounted directly on the adjacent construction. This solution is particularly economical since there is then no need for the locating bore that would otherwise be required or for the bearing cover with the associated matching work.

For some areas of application, a bearing arrangement of lower precision is often sufficient. For this purpose, bearings with less stringent tolerances are available.

Needle roller/axial cylindrical roller bearings

100

These precision bearings are double direction axial cylindrical roller bearings with a radial bearing component.

The bearings are available with and without fixing holes in the outer ring. Bearings with holes are screw mounted directly on the adjacent construction. The large contact surface and the small pitch of the holes allows a connection to the adjacent construction that is extremely rigid and with low tendency to settling. There is therefore no need for the bearing cover that would otherwise be required to hold the bearing, and the matching work required.

If the axial abutment of the shaft locating washer is not sufficient or a seal raceway is required, bearings with a stepped shaft locating washer extended on one side are suitable.

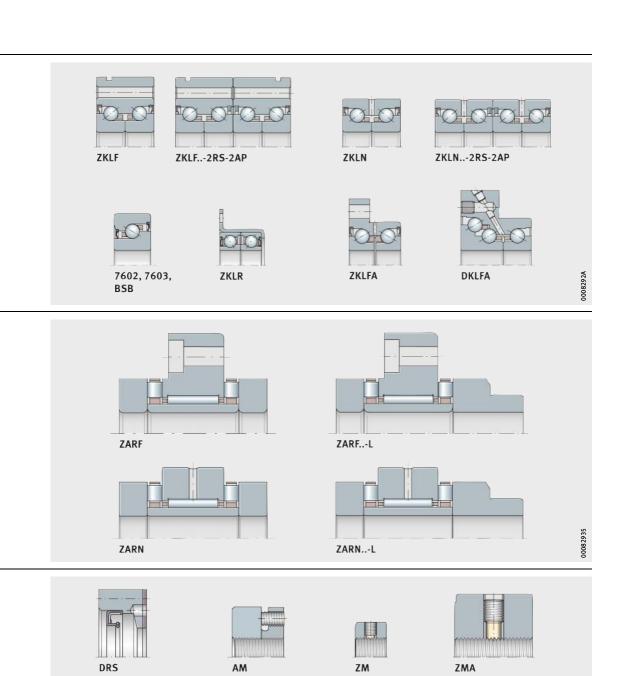
Accessories

134

Seal carrier assemblies Precision locknuts Sockets BEARINX-online Easy BallScrew Seal carrier assemblies DRS are complete sealing sets that are screw mounted on the outer ring of needle roller/axial cylindrical roller bearings ZARF(L). They are precisely centred in this position and seal the bearings against external influences.

Precision locknuts are used to achieve a defined axial preload in bearings for screw drives. They are also used where high axial forces must be supported and high axial runout accuracy and rigidity are required.

Calculations on screw drive bearing arrangements can be carried out free of charge using BEARINX-online Easy BallScrew.



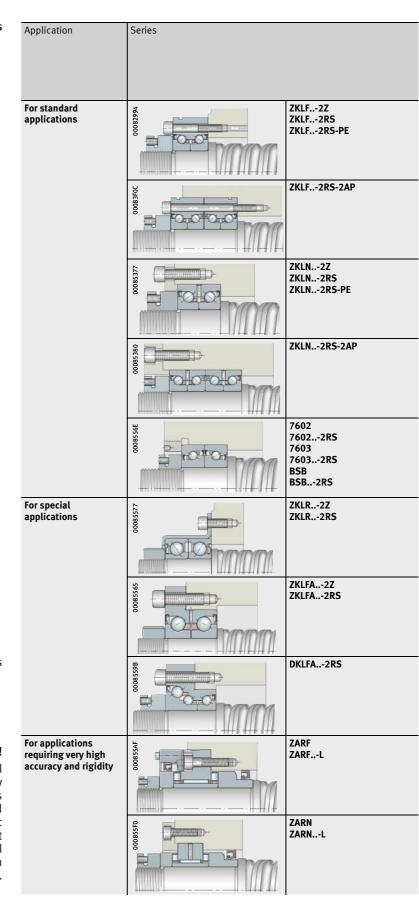
Schaeffler Technologies TPI 123 3

LOCKNUT-SOCKET-KM

AMS

00086074

Matrix for preselection of bearings



Definition of symbols

- +++ Very good
- ++ Good
- + Satisfactory
- Adequate

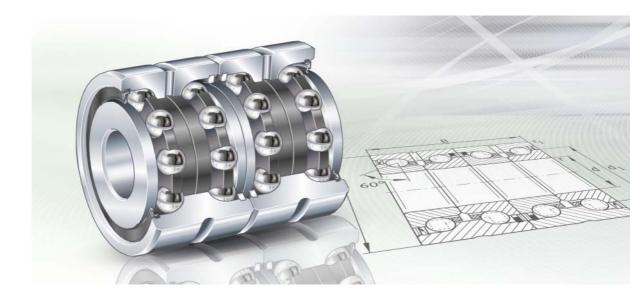
Attention!

This selection matrix is a general overview for preliminary assessment of whether bearings can be considered for the required purpose. For the specific application, the data in the product sections and the technical principles must always be taken into consideration.

Features	Character	istics				
	High radial runout accuracy	High speeds and low friction	High rigidity and load carrying capacity	Work on adjacent construction and mounting	Lowlubricant consumption	Description, page
Outer ring for flange mounting	++	+++	+	+++	++	13
Gap seals or contact seals on both sides	++	++	+	+++	++	
Greased, with relubrication facility Most economical type of screw drive bearing arrangement	•		T	****	111	
As ZKLF, but in matched pair Significantly higher load carrying capacity and rigidity than ZKLF	++	++	++	++	++	15
Gap seals or contact seals on both sides	++	+++	+	+	++	16
Greased, with relubrication facility	++	++	+	+	++	
Easier mounting than single row bearings, e.g. for plummer block housings	T	***	T	*	**	
As ZKLN, but in matched pair	++	++	++	+	++	17
Significantly higher load carrying capacity and rigidity than ZKLN				'		,
Open or sealed on both sides	++	+++	+	-	++	20
Highly suitable for mounting in pairs to form	++	++	+	-	++	
wide range of contact angle combinations	++	+++	+	_	++	
	++	+++	ļ.	_	++	
	++	++	+	-	++	
Particularly economical alternative to solutions comprising individual ball bearings Gap seals or contact seals on both sides For applications with comparatively low loads, but high positional accuracy	+ +	+++	-	+++	++	22
Outer ring for flange mounting Stepped outer ring with flattened areas on both sides Gap seals or contact seals on both sides	++	+++	+ +	+++	++	24
Higher load carrying capacity on one side Outer ring for flange mounting Sealed on both sides For vertical axes For locating bearing arrangements on both	+	++	++	+++	++	24
sides Outer ring for flange mounting Very high accuracy, rigidity and load carrying capacity Bearing prepared for sealing	+++	+	+++	+	-	103
Very high accuracy, rigidity and load carrying capacity Bearing prepared for sealing	+++	+	+++	-	-	104

Schaeffler Technologies TPI 123 5





Axial angular contact ball bearings

		Page
Product overview	Axial angular contact ball bearings	8
Features	Single row and multiple row designs	10
	X-life	12
	Double row design, for screw mounting	13
	Double row design, not for screw mounting	16
	Hybrid design HC	18
	Single row design, suitable for any combination	20
	Angular contact ball bearing unit, for screw mounting	22
	Double row design, with flange, for screw mounting	24
	Triple row design, with flange, for screw mounting	24
	Operating temperature	26
	Cages	26
	Suffixes	26
	Structure and meaning of designations	27
Design and	Basic rating life	29
safety guidelines	Static load safety factor	29
	Resultant and equivalent bearing load	30
	Maximum radial load on screw connections for ZKLF	38
	Permissible static axial load for ZKLF	39
	Design of screw drive bearing arrangements	39
	Design of a locating/locating bearing arrangement	44
	Design of adjacent construction	47
	Speeds	51
	Friction	52
	Lubrication	53
	Mounting guidelines	54
	Calculation example	
	f screw drive bearing ZKLF30100-2Z-XL	58
Dimension tables	Axial angular contact ball bearings	
	For screw mounting	66
	For screw mounting, less stringent tolerances For screw mounting, matched pair	74 76
	Not for screw mounting	78
	Not for screw mounting, less stringent tolerances	82 84
	Not for screw mounting, matched pair	86
	Single direction, sealed on both sides	90
	Angular contact ball bearing units, for screw mounting	92
	Double row axial angular contact ball bearings with flange, for screw mounting	94
	Triple row axial angular contact ball bearings with flange,	
	for screw mounting	98







Product overview Axial angular contact ball bearings

Double row design, for screw mounting

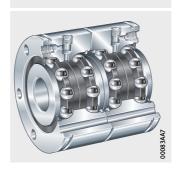
Lip seals or minimal gap seals

ZKLF..-2RS, ZKLF..-2Z, ZKLF..-2RS-PE



Matched pair design

ZKLF..-2RS-2AP



Double row design, not for screw mounting

Lip seals or minimal gap seals

ZKLN..-2RS, ZKLN..-2Z, ZKLN..-2RS-PE



Matched pair design

ZKLN..-2RS-2AP



Single row design, for any combination

With or without seals

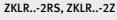
7602, 7603, BSB, BSB..-SU



7602..-2RS, 7603..-2RS, BSB..-2RS, BSB..-2Z-SU



Angular contact ball bearing unit, for screw mounting Lip seals or gap seals





Double row or triple row design, flange with flattened areas, for screw mounting

Double row design, lip seals or minimal gap seals

ZKLFA..-2RS, ZKLFA..-2Z



Triple row design, lip seals

DKLFA..-2RS









Schaeffler Technologies **TPI 123** | 9

Features

Screw drive bearing arrangements are subjected to requirements that often cannot be optimally fulfilled by the design construction of conventional bearings.

For the design of bearing arrangements that have high accuracy, high load carrying capacity, high rigidity, low friction, are easy to fit and maintenance-free or low-maintenance and are suitable for the highly dynamic operating conditions of threaded spindles, there is a wide range of INA and FAG axial angular contact ball bearings. With this product range, solutions can be found to all the technical and economic demands placed on the bearing arrangements of screw drives.

Single row and multiple row designs

Axial angular contact ball bearings are available as single, double or triple row ready-to-fit units. They are self-retaining and comprise thick-walled, geometrically stable outer rings, ball and cage assemblies and one-piece or two-piece inner rings. In several series, the outer ring has through holes for simple flange mounting to the adjacent construction.

The bearing rings are matched to each other such that a defined preload is achieved when the rings are clamped in place using a precision locknut.

Axial and radial load capacity

Due to the 60° contact angle, the bearings can support high axial forces as well as radial forces.

Contact angle arrangement in X and O arrangements

In arrangements of multiple row angular contact ball bearings, a distinction is made between 0, X and tandem arrangements in relation to the arrangement of contact angles. The arrangement of contact angles has an essential influence on the tilting rigidity of the bearing position, *Figure 1*, page 11, and *Figure 2*, page 11.

The characteristics of the O arrangement are:

- high tilting rigidity
- higher critical whirling speed of the threaded spindle due to the high tilting rigidity of the bearing position
- longer rating life under additional radial loads due, for example, to a belt drive

The characteristics of the X arrangement are:

- low tilting rigidity
- lower critical whirling speed of the threaded spindle
- shorter rating life under additional radial loads due, for example, to a belt drive.

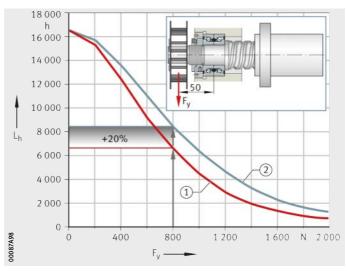
Calculation example with BSB3062-SU $n = 500 \text{ min}^{-1}$

 L_h = basic rating life ψ = misalignment

X arrangement
 O arrangement

Figure 1
Identical rating life of bearings
in X and O arrangement
with misalignments between
bearing seat and threaded spindle

18 000 h 16 000 14 000 12 000 10 000 8 000 (1) 6 000 4 000 2 2 000 0 00087A94 Ó 0,02 0,04 0,06 0,08 mm 0,1



Calculation example with BSB3062-SU $n = 500 \text{ min}^{-1}$

 $L_h = \ basic \ rating \ life$ $F_y = \ additional \ radial \ load \ due \ to \ belt \ drive$

① X arrangement

② O arrangement

Figure 2
Longer rating life of bearings
in O arrangement
with additional radial loads

Schaeffler Technologies TPI 123 | 11









Axial angular contact ball bearings ZKLF and ZKLN are supplied in the X-life design.

As a result of increased raceway accuracy and raceway quality, there is a significant reduction in the stress conditions present on the rolling elements and raceway under the same load, compared to the standard bearings. The improved quality gives reduced friction in the bearing and lower bearing temperatures; running resistance is lower, less strain is placed on the lubricant and the grease operating life and, where applicable, necessary relubrication intervals are extended. As a result of the lower frictional energy, there is a simultaneous increase in the energy efficiency of the bearing arrangement.

Higher basic load ratings and limiting speeds with X-life

The basic dynamic load ratings C_a of axial angular contact ball bearings in the X-life design are 10% higher than those of the previous standard designs. This leads to a longer rating life L_{10} or can be used to allow higher loads on the bearing arrangement while achieving the same rating life.

The lower bearing friction and associated reduction in heat generation in the bearing permit significantly higher limiting speeds $n_{G\ grease}.$

Further information

- Reprint SSD32, Bearings for Screw Drives in X-life Quality
- Download and ordering: http://medien.schaeffler.de.

Double row design, for screw mounting

Axial angular contact ball bearings ZKLF are screw mounted directly on the adjacent construction or in a locating bore, Figure 3 and Figure 4.

When axial angular contact ball bearings ZKLF are mounted without radial centring, *Figure 3*, the bearing is aligned in a radial direction using linear guidance systems and screw drive as a datum.

This prevents misalignments between the screw drive and bearing position. As a result, constraining forces that reduce the rating life cannot occur.

For easier dismounting of the bearings from a locating bore, the outer ring has a circumferential extraction slot, Figure 4.

Relubrication is facilitated by radial and axial threaded connectors with detachable grub screws.



ZKLF..-2RS ZKLF..-2Z ZKLF..-2RS-PE

1) Locknut

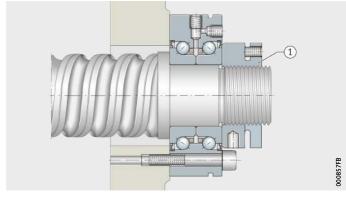
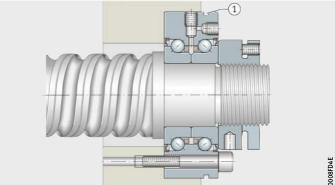


Figure 3 Outer ring screw mounted

on flat adjacent construction, preloaded by locknut AM



ZKLF..-2RS ZKLF..-2Z ZKLF..-2RS-PE

(1) Extraction slot

Figure 4 Outer ring screw mounted in bore, preloaded by locknut AM

With less stringent tolerances

Axial angular contact ball bearings ZKLF..-2RS-PE correspond to series ZKLF but have an axial runout tolerance within tolerance class 5 to ISO 492 (DIN 620-2), less stringent diameter tolerances and the suffix PE. They are used where a lower positional accuracy is acceptable. As a result, the requirement for accuracy of the adjacent construction is lower with these bearings.

Heavy series

Axial angular contact ball bearings ZKLF..-2RS and ZKLF..-2Z are also available in a heavy series. For the same shaft diameter, they have a larger cross-section and thus higher basic load ratings.







Advantages of axial angular contact ball bearings ZKLF compared with solutions based on single bearings The aim of the designer must be to achieve a combination of a drive spindle function with the longest possible operating life and the lowest possible overall costs.

The operating life is dependent on the correct preloading of the bearings and effective sealing. A reduction in the mechanical interfaces (the number of components) also means a reduction in possible mounting errors.

This is made possible by the use of axial angular contact ball bearings ZKLF. The bearings are supplied already correctly matched and sealed. This eliminates the need for mounting of individual bearings with its associated errors. Through flange mounting on a flat surface, alignment of the threaded spindle is a very simple process.

In the analysis of overall costs, it must also be taken into consideration that the use of single bearings involves not only the bearing costs themselves but also the following outlays, *Figure 5*:

- manufacturing costs for additional components (housing, cover)
- costs for seals
- mounting costs for the bearing unit
- costs for matching work in alignment of the spindle
- corresponding stockholding costs and logistics outlay.

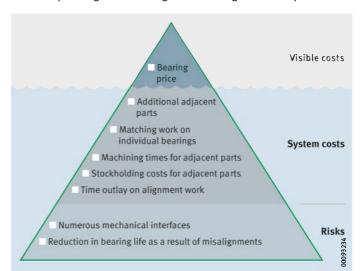
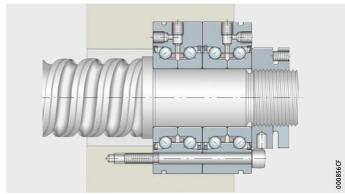


Figure 5
Outlay in single bearing solutions,
presented as an iceberg diagram

Matched pair design

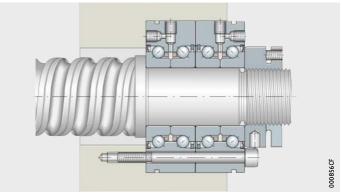
Bearings of series ZKLF..-2RS-2AP comprise a matched pair of bearings ZKLF..-2RS, Figure 6 and Figure 7. The individual bearings are matched to each other.

Matched bearings have an arrow marking on the outside surface of the outer rings. When the bearings are arranged correctly, the lip seals face outwards. During fitting, the hole pattern of the two bearings must match, not the marking.



ZKLF..-2RS-2AP

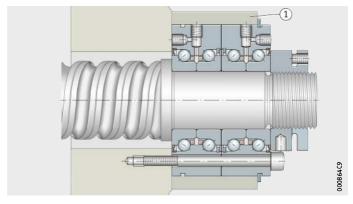
Figure 6 Matched pair, outer ring screw mounted in bore, preloaded by locknut AM



ZKLF..-2RS-2AP

(1) Support ring (not included in scope of delivery)

Figure 7 Matched pair, screw mounted on face of adjacent construction, preloaded by locknut AM



Seals

Bearings with the suffix 2RS have lip seals and highly effective

Bearings with the suffix 2Z are sealed using minimal gap seals and are suitable for higher speeds.

Lubrication

The bearings are greased using a lithium soap grease to GA28 and can be lubricated via the lubrication connectors in the outer ring. For the majority of applications, the initial greasing is sufficient for the whole bearing operating life.

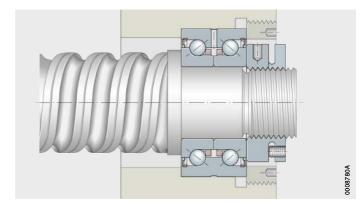






Double row design, not for screw mounting

Axial angular contact ball bearings ZKLN are mounted in a housing bore, *Figure 8* and *Figure 9*. The bearing outer ring is axially located by a ring nut or flange cover.



ZKLN..-2RS ZKLN..-2Z

Figure 8
Outer ring located by ring nut,
preloaded by locknut AM



Figure 9
Outer ring located by flange cover,
preloaded by locknut AM

With less stringent tolerances

Axial angular contact ball bearings ZKLN..-2RS-PE correspond to series ZKLN but have an axial runout tolerance within tolerance class 5 to ISO 492 (DIN 620-2), less stringent diameter tolerances and the suffix PE. They are used where a lower positional accuracy is acceptable. As a result, the requirement for accuracy of the adjacent construction is lower with these bearings.

Heavy series

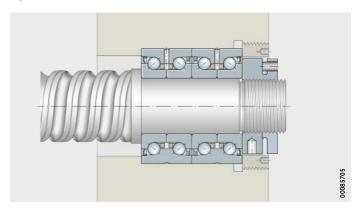
Axial angular contact ball bearings ZKLN..-2RS and ZKLN..-2Z are also available in a heavy series. For the same shaft diameter, they have a larger cross-section and thus higher basic load ratings.

Matched pair design

Bearings of series ZKLN..-2RS-2AP comprise a matched pair of bearings ZKLN..-2RS, *Figure 10*. The individual bearings are matched to each other.

Matched bearings have an arrow marking on the outside surface of the outer rings. When the bearings are arranged correctly, the lip seals face outwards.

The bearing outer ring is axially located by a ring nut or flange cover, *Figure 10*.



ZKLN..-2RS-2AP

Figure 10

Matched pair,
outer ring located by ring nut,
preloaded by locknut AM



Bearings with the suffix 2RS have lip seals and highly effective sealing.

Bearings with the suffix 2Z are sealed using minimal gap seals and are suitable for higher speeds.

Lubrication

The bearings are greased using a lithium soap grease to GA28 and can be lubricated via a lubrication slot and lubrication holes in the outer ring. For the majority of applications, the initial greasing is sufficient for the whole bearing operating life.

The axial angular contact ball bearing ZKLN0619-2Z cannot be relubricated.







Schaeffler Technologies

Hybrid design HC

With the hybrid axial angular contact ball bearings ZKLF...HC and ZKLN...HC, Schaeffler offers reliable solutions to the problem of "false brinelling", known in relation to and occasionally occurring in feed axes with ball screw drives, in which balls are thought to cause indentations in the raceways. Premature bearing failures due to this type of damage, often described as fluting, can thus be avoided.

If operating conditions are present, however, where there is insufficient relative motion between the rolling contact partners, the lubricant film at the rolling contact is interrupted and partial dry running occurs (direct steel/steel contact). Such operating conditions occur, for example, with small swivel angles, highly dynamic positional regulation or vibrations in a stationary state. This can occasionally lead, within a few hours, to premature bearing failure through false brinelling.

Ceramic rolling elements offer the possibility of permanently eliminating this harmful steel/steel contact, *Figure 11*.



Figure 11
Hybrid axial angular contact
ball bearing ZKLF..-HC

Advantages of hybrid bearings

The advantages of hybrid bearings in comparison with steel bearings are:

- reliability, even in short stroke operation for long periods
- a three-fold increase in grease operating life
- higher speed capability.

Other characteristics are as follows:

- They give the same basic rating life
- The fact that the basic static load ratings are approx. 30% lower is irrelevant in many screw drive applications
- The axial rigidity in the raceway system is only 5% lower.

Availability

The following series and sizes are available as standard in the hybrid design:

- ZKLF..-2RS, ZKLF..-2Z, ZKLN..-2RS, ZKLN..-2Z
- bearings with d = 20 mm to 100 mm
- all other series and sizes can be offered by agreement with Triondur coating in order to prevent false brinelling.

Ordering example

Double row axial angular contact ball bearing ZKLF with rings made from rolling bearing steel and balls made from ceramic (Hybrid Ceramic), minimal gap seals

Ordering designation

ZKLF40100-2Z-HC

Further information

- Detailed information on "false brinelling" in feed axes and hybrid designs: see reprint SSD 26, Hybrid Axial Angular Contact Ball Bearings for Screw Drives
- Download and ordering: http://medien.schaeffler.de.



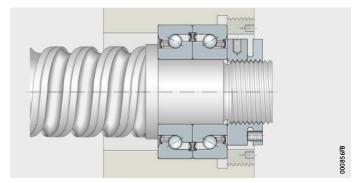




Single row design, suitable for any combination

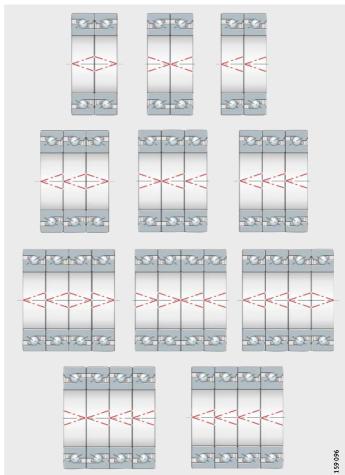
The axial angular contact ball bearings 7602, 7603 and BSB are designed as universal bearings, *Figure 12* and *Figure 13*.

They can be used in sets in any combination for different requirements, *Figure 13*.



7602, 7603 BSB

Figure 12
Single row axial angular
contact ball bearings,
2 bearing set in X arrangement



7602, 7603 BSB

Figure 13
Bearing set combinations

Marking of the mounting position

The bearings have arrow marks on the outside diameter to indicate the contact angle arrangement:

- <> is an O arrangement
- >< is an X arrangement</p>
- << is a tandem arrangement.</p>

In bearings with seals (BSB..-2Z-SU), the contact angle can also be recognised from the seal colour after mounting:

- A brown seal on the outside indicates an O arrangement (side with high inner ring shoulder)
- A red seal on the outside indicates an X arrangement (side with low inner ring shoulder).

Precision matched to requirements

The dimension tables contain the two axial angular contact ball bearings BSB..-T and BSB..-SU with identical dimensions. Both series differ slightly in their rolling element system. The essential difference lies in the precision of the raceways.

Axial angular contact ball bearings BSB..-T and 760 are manufactured to high requirements comparable with those of main spindle bearings. As a result, running noise is extremely low even at high speeds. These bearings are therefore used in high precision screw drive applications and in devices used in low-noise environments such as laboratories.

For applications in general machine building, for example in machine tools and handling systems, axial angular contact ball bearings BSB..-SU have adequate precision.

Seals

Single row axial angular contact ball bearings of the basic design are not sealed.

The most common bearing sizes are also available in sealed versions:

- 760..-2RS and BSB..-2RS contact lip seals 2RS with minimal contact force
- BSB..-2Z-SU minimal gap seals 2Z.

Lubrication

Single row axial angular contact ball bearings of the open basic design are not greased. These bearings are used in preference for applications with oil lubrication.

Alternatively, the bearings can be ordered with the suffix L055. These are greased with the high performance grease Arcanol MULTITOP and can be relubricated via the open raceway gap in the end face.

Single row, sealed axial angular contact ball bearings are greased for life with this grease.

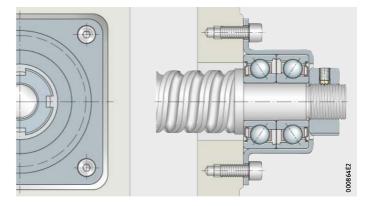






Angular contact ball bearing unit, for screw mounting

Angular contact ball bearing units ZKLR are low-cost bearings and particularly economical. They comprise a deep drawn sheet steel housing with an integral matched pair of angular contact ball bearings in an X arrangement, *Figure 14*. The housing has a Corrotect coating and is thus protected against corrosion. The ball bearings are sealed on both sides by lip seals or gap seals. The bearing assembly is preloaded clearance-free.



ZKLR..-2Z ZKLR..-2RS

Figure 14
Angular contact ball bearing unit

Particularly easy to fit

The units are particularly easy to mount:

- They are directly flange mounted on the, for example, milled face of the adjacent construction. This eliminates the need for precise, costly fits and additional flange covers for seating and axial location of the bearings.
- No additional components are required for location of the bearings in the housing.
- The possible errors in mounting are reduced due to the smaller number of components compared to conventional solutions.
- They align themselves by means of the screw drive nut during mounting. This effectively eliminates any distortion due to misalignment of the bearing seats.
- They have a defined preload. There is thus no need for preloading of the bearings during mounting.
- Axially clearance-free location on the spindle is sufficient.

Seals Bearings with the suffix 2RS have lip seals and highly effective

sealing. Bearings with the suffix 2Z are sealed using gap seals and

are suitable for higher speeds.

Lubrication The bearings are greased for life using a lithium soap grease to

DIN 51825-K2N-40.

Operating limits The units are highly suitable, due to their dimensions and design,

 $for space-saving \ and \ simple \ bearing \ arrangements \ in \ highly \ dynamic$

feed screw drives.

Areas of application include:

- inspection and measuring machines
- small machining equipment
- medical and laboratory equipment
- machines in precision engineering and electronic component manufacture
- simplification of the bearing arrangement through the use of complete units.







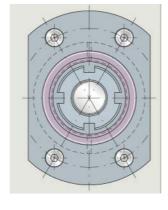
Schaeffler Technologies TPI 123 | 23

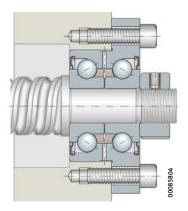
Double row design, with flange, for screw mounting

Series ZKLFA has a stepped outer ring. It can therefore be screw mounted easily on the adjacent construction, *Figure 15*. The flange has flattened areas on both sides, so the radial design envelope in the adjacent construction can be kept very small.

ZKLFA..-2RS ZKLFA..-2Z

Figure 15
Outer ring screw
mounted on adjacent construction,
preloaded by locknut ZM





Seals

Bearings with the suffix 2RS have lip seals and highly effective sealing. Bearings with the suffix 2Z are sealed using gap seals and are suitable for higher speeds.

Lubrication

The bearings are greased using a lithium soap grease to GA28 and can be lubricated via a lubrication slot and lubrication holes in the outer ring. For the majority of applications, the initial greasing is sufficient for the whole bearing operating life.

The axial angular contact ball bearing ZKLFA0630-2Z cannot be relubricated.

Triple row design, with flange, for screw mounting

Axial angular contact ball bearings DKLFA..-2RS have, in addition to two rows of balls with a contact angle of 60° in an O arrangement, a further row of balls, *Figure 16* and *Figure 17*. Due to this additional row, the bearings can support higher axial loads in one direction.

Due to the stepped outer ring, the bearings can be easily flange mounted on the adjacent construction. The flange has flattened areas on both sides. As a result, it requires only a small design envelope in the adjacent construction.



In order to make full use of the load carrying capacity of bearings DKLFA..-2RS, their design requires that they are subjected to continuous load in the main load direction. They are therefore used mainly in screw drives with a locating/locating bearing arrangement and tensioned spindles or in vertically arranged screw drive bearing arrangements.

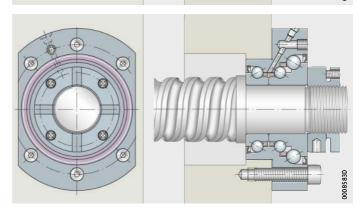
For axial angular contact ball bearing units DKLFA, the locknut tightening torque must be configured in accordance with the load data.

For design of the bearing arrangement, please contact us.

53 M880 C

DKLFA..-2RS

Figure 16
Outer ring screw
mounted on adjacent construction
without radial centring,
preloaded by locknut AM



DKLFA..-2RS

Figure 17
Outer ring screw
mounted on adjacent construction,
preloaded by locknut AM

Seals

Triple row axial angular contact ball bearings have lip seals on both sides.

Lubrication

The bearings are greased using a lithium soap grease to GA28. If necessary, lubrication can be carried out via a threaded connector for central lubrication systems on the flange side.

Schaeffler Technologies TPI 123 | 25

Operating temperature

Axial angular contact ball bearings are suitable for operating temperatures from -30 °C to +120 °C, restricted by the grease, the seal material and the plastic cages in the ball and cage assemblies.



The operating temperature influences the dynamic bearing characteristics. The values given in the dimension table are based on a room temperature of +20 °C.

Cages

The axial angular contact ball bearings have plastic cages. The cage design is not indicated in the suffix.

In axial angular contact ball bearings 7602, 7603 and BSB, the polyamide cages are indicated by the suffix TVP or T.

Suffixes

Suffixes for available designs: see table.

Available designs

Suffix	Description	Design
PE	Less stringent bearing design	Standard
T, TVP	Polyamide cage	
2AP	Axial angular contact ball bearings, matched pair (ZKLN, ZKLF)	
2RS	Contact lip seals on both sides	
2Z	Minimal gap seals on both sides	
L055	Greased with high performance grease Arcanol MULTITOP	
SU	Universal bearing; single bearings can be used in sets in any combination required	
НС	Hybrid Ceramic, rings made from rolling bearing steel, balls made from ceramic	Special design, available by agreement

Structure and meaning of designations

The structure of the designations of axial angular contact ball bearings is specific to the series. The designation of the series is followed by indications of the bearing size and finally indications of the design variants and seals, see tables, Figure 18 and Figure 19, page 28.

Structure of designations of axial angular contact ball bearings

Feature		Indication,	Series ①				
		example	ZKLF	ZKLN	ZKLFA	DKLFA	ZKLR
2	Inside diameter	15	•	•	•	•	•
3	Outside diameter	30	•	•	_	-	_
	Flange outside diameter	63	-	-	•	•	
	Flange height	24			-	-	•
4	Minimal gap seals	2Z	•	•	•	-	•
	Lip seals	2RS	•	•	•	•	•
(5)	Standard design	-	•	•	•	•	•
	Less stringent accuracy	PE	•	•	_	-	_
	Two bearings in matched pair	2AP	•	•			
	With ceramic balls	НС	•	•			

Structure of designations of axial angular contact ball bearings (continued)

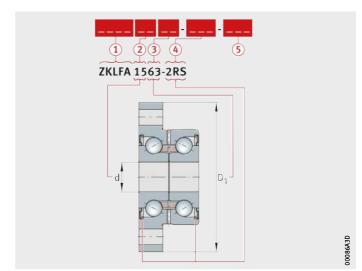
Гол	+*	Indication,	Series (1)			
Feature		example				
		example	76	BSBT	BSBSU	
1	ISO diameter series 2	02	•	_	_	
	ISO diameter series 3	03				
2	Inside diameter	25	•	•	•	
3	Outside diameter	62	-	•	•	
4	Without seals	-	•	•	•	
	Lip seals and greased with Arcanol MULTITOP	2RS	•	•	_	
	Minimal gap seals and greased with Arcanol MULTITOP	2Z	-	_	•	
(5)	Polyamide cage	T, TVP	•	•	_	
6	Universal single bearings suitable for use in sets in any combination required					
	Single bearing	_	•	•	_	
		SU	_	-	•	
	2 bearing set	D	•	•	•	
	3 bearing set	T	•	•	•	
	4 bearing set	Q	•	•	•	
7	Ungreased (bearing without seals)	_	•	•	•	
	Bearing greased with Arcanol MULTITOP	L055	•	•	•	

Indicated in ordering designation.



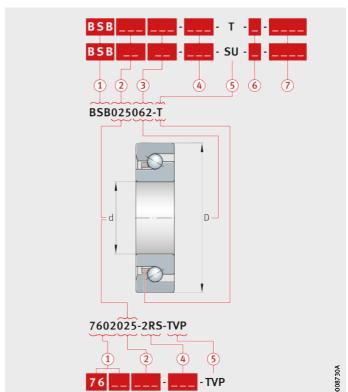






ZKLFA1563-2RS = example, see table, page 27

Figure 18
Structure of ordering designations
of axial angular contact ball
bearings ZKLF, ZKLN, ZKLFA,
DKLFA and ZKLR



BSB025062-T = example, see table, page 27

Figure 19
Structure of ordering designations
of axial angular contact ball
bearings 76, BSB..-T, BSB..-SU

Design and safety guidelines Basic rating life

The decisive factors in determining the bearing size are the basic rating life, the static load safety factor and the axial limiting load.

The rating life L_{10} or L_{10h} is calculated as follows:

$$L_{10} = \left(\frac{C}{P}\right)^{p}$$

$$L_{10h} = \frac{16\,666}{n} \cdot \left(\frac{C}{P}\right)^{p}$$

106 revolutions

Basic rating life in millions of revolutions

 $\begin{array}{c} L_{10h} & \text{ \tiny II} \\ \text{Basic rating life in operating hours} \\ & \text{\tiny NI} \end{array}$

Basic dynamic radial or axial load rating, see dimension table

 C_r is valid for a contact angle $\alpha \le 45^\circ$, C_a is valid for a contact angle $\alpha > 45^{\circ}$

Equivalent dynamic bearing load

Life exponent p = 3

 min^{-1}

Operating speed.

Static load safety factor

The static load safety factor S₀ indicates the security against impermissible permanent deformations in the bearing:

$$S_0 = \frac{C_0}{P_0}$$

Solution Static load safety factor

Basic dynamic radial or axial load rating, see dimension table

 C_{0r} is valid for a contact angle $\alpha \leq 45^{\circ}$, C_{0a} is valid for a contact angle $\alpha > 45^{\circ}$

Maximum static load of bearing.



In machine tools, S_0 should be ≥ 4 .

A value below this limit is only possible after analysis of the individual case. This will include a calculation study of the contact ellipse value k_s. The value describes the position of the contact ellipse in the bearing with complete $(k_s = 1)$ or partial support $(k_s < 1)$.

We recommend that the calculation should be requested from Schaeffler, stating the load and speed duty cycle.







Resultant and equivalent bearing load

The resultant axial bearing load $F_{a\;res}$ is determined from the axial operating load F_{aB} and taking account of the axial preload.

Axial and radial operating loads

Under pure axial load, $P=F_{a\,res}$. If additional radial operating loads are present, P must be calculated as follows:

$$P = X \cdot F_r + Y \cdot F_{a res}$$

Equivalent dynamic bearing load

Radial dynamic bearing load

 $\begin{array}{cc} {\rm F_{a\,res}} & {\rm N} \\ {\rm Resultant\,axial\,dynamic\,bearing\,load} \end{array}$

Radial load factor, axial load factor. The load factors X and Y can be taken from the following table.

Load factors X and Y

Load ratio	Load factors		
	X	Υ	
$\frac{F_{a res}}{F_r} \le 2,17$	1,9	0,55	
$\frac{F_{ares}}{F_r} > 2,17$	0,92	1	

Load varying in steps

In this case, P and n are calculated as follows:

$$\mathsf{P} = \sqrt[p]{\frac{\mathsf{q}_1 \cdot \mathsf{n}_1 \cdot \mathsf{P}_1^{\,\mathsf{p}} + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z \cdot \mathsf{P}_z^{\,\mathsf{p}}}{\mathsf{q}_1 \cdot \mathsf{n}_1 + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z}}$$

$$n = \frac{q_1 \cdot n_1 + \ldots + q_z \cdot n_z}{100}$$

Equivalent dynamic bearing load

Life exponent p = 3

Time proportion

 ${\rm min}^{-1}$

Operating speed.

Static limiting load diagrams for ZKLR

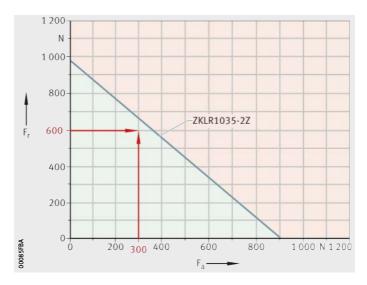
The easy-to-use static limiting load diagrams allow quick verification of the bearing size of angular contact ball bearing units ZKLR, as a function of the axial and radial operating load, *Figure 21*, page 32 and *Figure 22*, page 32. The decisive requirement is that the intersection between the axial and radial load must be below the limit line.

Example, Figure 20: If the operating loads F_a and F_r are below the limit line, the bearing size is suitable for the application.

For some angular contact ball bearing units ZKLR, equations are available instead of the diagrams, see table.

Resultant and equivalent bearing load P for ZKLR

Designation	Equivalent bearing load P
ZKLR0624-2Z	$P = 140 + 0,11 \cdot F_a^{1,45} + 0,012 \cdot F_r^{1,66}$
ZKLR0828-2Z	$P = 215 + 0.21 \cdot F_a^{1.32} + 0.0053 \cdot F_r^{1.74}$
ZKLR1035-2Z	$P = 240 + 0.49 \cdot F_a^{1.18} + 0.016 \cdot F_r^{1.55}$
ZKLR1244-2RS	$P = 720 + 0.34 \cdot F_a^{1.32} + 0.095 \cdot F_r^{1.54}$
ZKLR1547-2RS	$P = 550 + 0,0074 \cdot F_a^{1,55} + 0,045 \cdot F_r^{1,35}$
ZKLR2060-2RS	$P = 930 + 0,0059 \cdot F_a^{1,54} + 0,038 \cdot F_r^{1,35}$

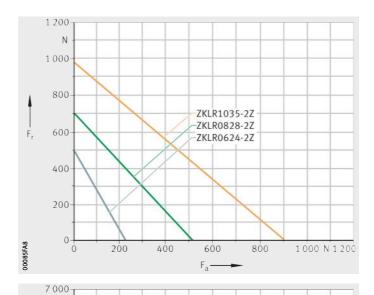


 $F_r = 600 \text{ N}$ $F_a = 300 \text{ N}$

The bearing is suitable in static terms since the intersection F_a/F_r is below the limit line.

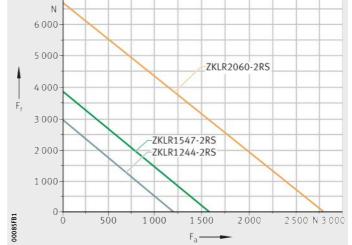
Figure 20
Static limiting load diagram,
example ZKLR1035-2Z

Schaeffler Technologies TPI 123 | 31



 F_r = radial load $F_a = axial load$

Figure 21
Static limiting load diagram for ZKLR..-2Z



 F_r = radial load $F_a = axial load$

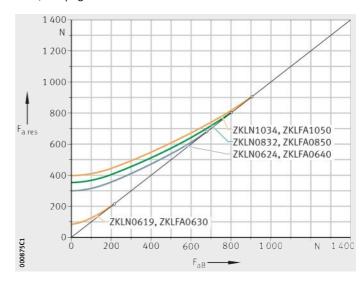
Figure 22 Static limiting load diagram for ZKLR..-2RS

Resultant and equivalent bearing load P for ZKLN, ZKLF, ZKLFA, DKLFA, BSB, 7602, 7603 The bearings are axially preloaded to a defined value if the recommended precision locknuts are used and the correct nut tightening torque is applied. The resultant axial bearing load $F_{a\ res}$ is determined from the axial operating load F_{aB} and taking account of the axial preload, Figure 23 to Figure 34, page 37. For 7602, 7603 and BSB, the diagrams for $F_{a\ res}$ are based on two-bearing sets in an O or X arrangement, see page 20. For other set combinations, please contact us.



A load in excess of the limiting load will cause the rolling element without load to lift off the raceway, in which case the bearing is no longer under preload. As a result, slippage will occur during highly dynamic acceleration events, which will lead to wear. Based on experience, the effect is less pronounced in the case of ball bearings and is therefore non-critical.

For extreme moment loads and statically overdefined systems (locating/locating bearing arrangements), please contact us. The calculation program BEARINX can give a precise design in this case, see page 154.



 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 23
Resultant bearing load
for ZKLN and ZKLFA,
up to d = 10 mm





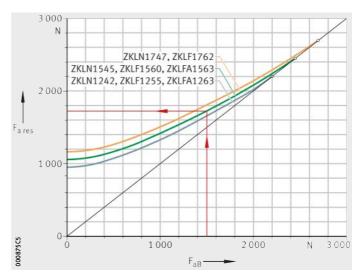


Schaeffler Technologies TPI 123 | 33

Example (red line): Axial angular contact ball bearing ZKLN1545 $F_{aB}=1\,500\text{ N}$ $F_{a\text{ res}}\approx 1750\text{ N}$ Loading by operating load and preload

 F_{aB} = operating load $F_{a \text{ res}}$ = resultant bearing load $^{\circ}$ = limiting load

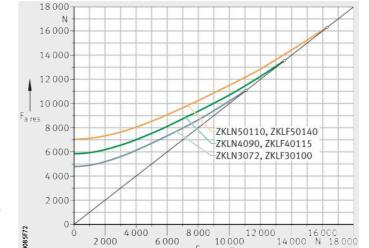
Figure 24
Resultant bearing load
for ZKLN, ZKLF and ZKLFA,
up to d = 17 mm



Ν 8000 ZKLN5090, ZKLF50115 7000-ZKLN4075, ZKLF40100 6000 ZKLN3572, ZKLF3590 5 000 4000 ZKLN3062, ZKLF3080 ZKLN2557, ZKLF2575 3 000-ZKLN2052, ZKLF2068 2 000-1 000 0-5 000 7 000 3 000 1 000

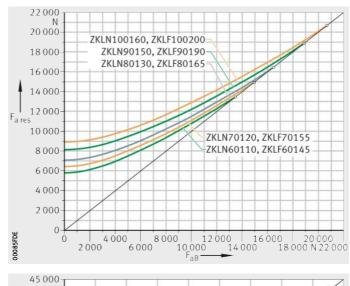
 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 25
Resultant bearing load
for ZKLN, ZKLF,
from d = 20 mm to 50 mm



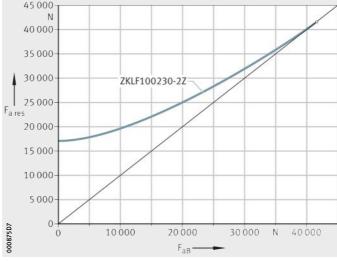
 F_{aB} = operating load $F_{a \text{ res}}$ = resultant bearing load \circ = limiting load

Figure 26
Resultant bearing load
for ZKLN, ZKLF,
heavy series



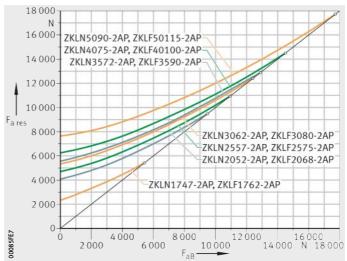
 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 27
Resultant bearing load
for ZKLN, ZKLF,
from d = 60 mm



 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 28
Resultant bearing load for ZKLF, d = 100 mm



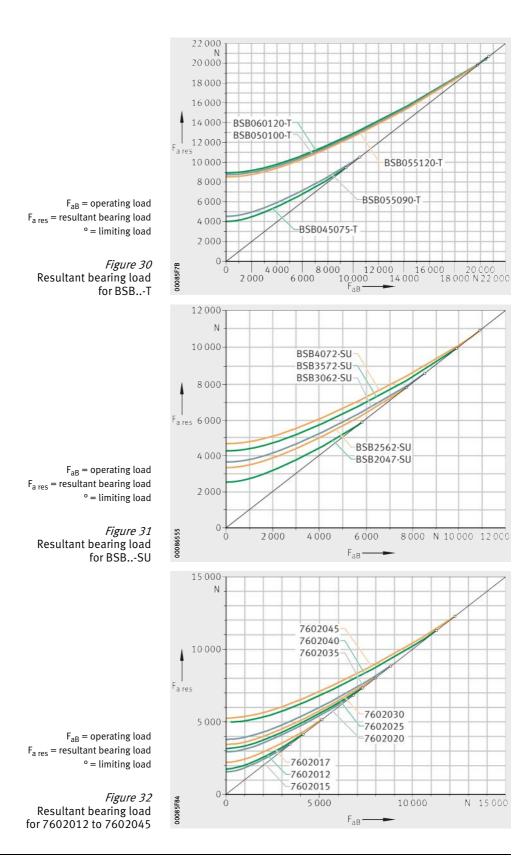
 F_{aB} = operating load $F_{a \text{ res}}$ = resultant bearing load $^{\circ}$ = limiting load

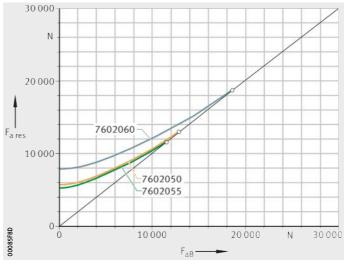
Figure 29
Resultant bearing load for ZKLN..-2AP and ZKLF..-2AP





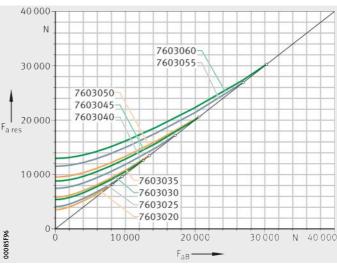






 F_{aB} = operating load $F_{a \text{ res}}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 33
Resultant bearing load for 7602050 to 7602060



 $F_{aB} = \text{operating load}$ $F_{a \text{ res}} = \text{resultant bearing load}$ $\circ = \text{limiting load}$

Figure 34
Resultant bearing load
for 7603020 to 7603065

Maximum radial load on screw connections for ZKLF

If bearings of series ZKLF are screw mounted on a flat adjacent construction, the maximum radial load that can be supported by the screw connections must be taken into consideration.

If the screws used are to DIN EN ISO 4762 as stated in the dimension tables, the maximum radial load that can be supported before the outer ring moves, in the case of dry components, is restricted to $\approx 0.1\times C_{0a}$. If the outer ring is additionally bonded in place using, for example, Loctite 638, the maximum radial load that can be supported increases to approx. $\approx 0.5\times C_{0a}$.

Influence of the rolling element set on permissible radial loads

If the maximum radial loads that can be supported (without any axial load content) are considered in relation to the rolling element set, it can be seen that:

- With a static load safety factor of $S_0 \ge 4$ as required for machine tools, the load that can be supported by the screw connections of the outer ring is in any case greater than the radial load restricted by the rolling element set.
- This means that (at least in theory) radial loads can be supported that, in relation to the rolling element set, will lead to a static load safety factor of $S_0 \le 1$ (incipient plastic deformation of rolling elements), if the outer ring is additionally bonded in place.
- This means that, under normal design criteria and with the decisive static load safety factor for the corresponding areas of application (e.g. S₀ = 4 for machine tools), the rolling element set is always the decisive criterion.

Permissible static axial load for ZKLF

For axial angular contact ball bearings ZKLF, the permissible static axial load in the direction of the screw connections, Figure 35, is defined as follows:

$$P_{0 per} \leq \frac{C_{0a}}{2}$$

P_{O per} ... Permissible static axial load N

 C_{0a}

Basic static axial load rating C_{0a} , see dimension tables.







ZKLF

Figure 35 Static axial load in the direction of the screw connections

Design of screw drive bearing arrangements

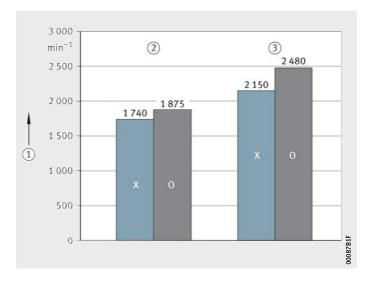
For screw drive bearing arrangements, a distinction is made between two design principles, a screw drive with a rotating spindle and a screw drive with a rotating nut. The type of bearing arrangement has an influence on the critical whirling speed, Figure 36.

Calculation example: diameter of threaded spindle = 40 mm spindle length = 2 000 mm

> X = X arrangement 0 = 0 arrangement

(1) Critical whirling speed, calculated (2) Locating/non-locating bearing arrangement 3 Locating/locating bearing arrangement

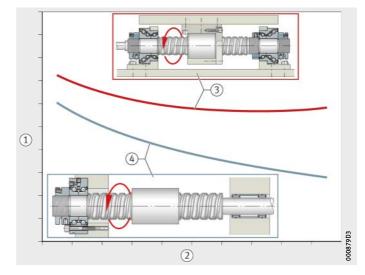
> Figure 36 Effect of bearing arrangement on critical whirling speed



Screw drive rigidity

The axial overall rigidity of the screw drive is of decisive importance for the machining accuracy and performance capability of machine tools. The axial system rigidity of a screw drive is heavily influenced by the design layout.

The highest axial system rigidity is achieved with the design "rotating screw drive nut". In the design "rotating threaded spindle", a significantly higher axial system rigidity is achieved with a locating/locating bearing arrangement on both sides and with increasing spindle length than with a locating/non-locating bearing arrangement, Figure 37.



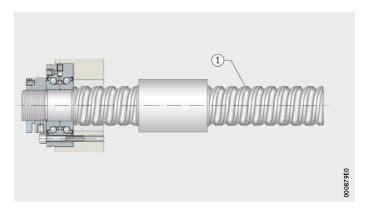
(1) Axial overall rigidity (2) Axial position of screw drive nut 3 Locating/locating bearing arrangement (4) Locating/non-locating bearing arrangement

Figure 37 Axial overall rigidity

Locating bearing arrangement on one side

Locating bearing arrangement on one side for rotating threaded spindle, Figure 38:

- for short threaded spindles
- preconditions for use with longer threaded spindles:
 - low demands on axial system rigidity
 - sufficiently high critical whirling speed.



(1) Rotating threaded spindle

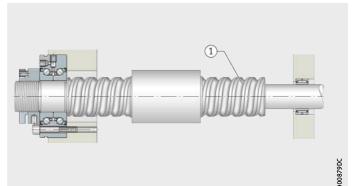
Figure 38 Locating bearing arrangement on one side

Locating/non-locating bearing arrangement

Locating/non-locating bearing arrangement for rotating threaded spindle, *Figure 39*:

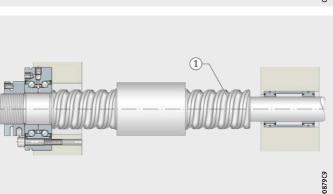
- for longer threaded spindles
- low demands on axial system rigidity
- sufficiently high critical whirling speed of the threaded spindle
- if the tilting rigidity on the locating bearing side is increased by means of a second bearing, an increase in the critical whirling speed of the threaded spindle can be achieved in this bearing arrangement, *Figure 40*. As non-locating bearings, deep groove ball bearings or needle roller bearings are normally used.





1 Rotating threaded spindle

Figure 39
Locating/non-locating
bearing arrangement



 ${\Large \textcircled{\scriptsize 1}} \ {\sf Rotating} \ {\sf threaded} \ {\sf spindle}$

Figure 40

Locating/non-locating bearing arrangement, higher tilting rigidity on non-locating bearing side

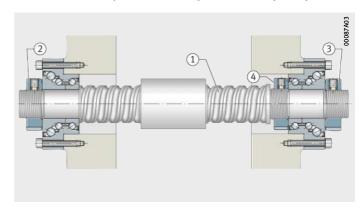
Locating/locating bearing arrangement

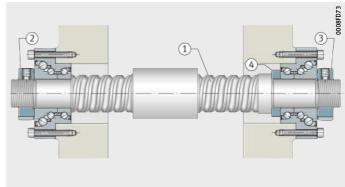
Locating/locating bearing arrangement for rotating threaded spindle, *Figure 41* and *Figure 42*:

- for medium length and long threaded spindles
- high axial system rigidity, *Figure 37*, page 40
- high critical whirling speed of the threaded spindle
- little effect of length changes under heating on the positioning. Spindle firmly clamped on both sides with rotating screw drive nut,

Figure 43 and Figure 44, page 43:

- for long threaded spindles
- for highly dynamic screw drives
- increased spindle tensioning possible
- no influence of spindle tensioning on the bearing rating life.
- ${\Large \textcircled{\scriptsize 1}} \ {\sf Rotating} \ {\sf threaded} \ {\sf spindle}$
 - (2) Locknut for preloading of bearing on left side
 - ③ Locknut for tensioning of threaded spindle
 - 4 Locknut for preloading of bearing on right side
- Figure 41
 Locating/locating
 bearing arrangement,
 tensioned spindle,
 adjustment using locknut
 - Rotating threaded spindle
 Locknut for preloading
 - of bearing on left side
 - 3 Locknut for tensioning of threaded spindle4 Shim
 - Figure 42
 Locating/locating
 bearing arrangement,
 tensioned spindle,
 adjustment using shim





(1) Rotating screw drive nut

Figure 43 Design example, rotating screw drive nut, tensioned spindle



1) Motor with toothed belt drive (2) Hollow shaft motor as direct drive

Figure 44 Application examples, rotating screw drive nut

Advantages of a locating/locating bearing arrangement In addition to axial rigidity, the dynamic characteristics of a screw drive are also important. A locating/locating bearing arrangement can be used to increase the critical whirling speed of the threaded spindle by up to 40% over a comparable locating/non-locating bearing arrangement.

A further aspect is the positional accuracy. If direct linear measuring systems are not used, a locating/locating bearing arrangement can be used to compensate the positional errors arising from spindle heating as long as the threaded spindle is in the tensioned state.

In the design of threaded spindles firmly clamped on both sides, tensioned and with a rotating screw drive nut, increased spindle tensioning is possible without this affecting the bearing rating life. As a result, particularly long threaded spindles with highly dynamic characteristics can be designed. For advice on and the preparation of mounting proposals, please make enquiries with Schaeffler.





Design of a locating/locating bearing arrangement

Where there is a locating bearing arrangement on one side (with or without a non-locating bearing on the opposing side, the threaded spindle may undergo thermal expansion without this changing the load on the bearing position. This does not apply in the case of the locating/locating bearing arrangement.



In order to allow the described advantages of a locating/locating bearing arrangement on both sides, some guidelines on design must be observed. If this is not done, it may lead to a reduced rating life or premature failure of the screw drive bearings.

Principles of a locating/locating bearing arrangement on both sides

A locating/locating bearing arrangement on both sides of a threaded spindle results in a statically overdefined system. In this case, the threaded spindle undergoes heating in operation and expands in length. This increase in length of the threaded spindle is, however, suppressed by the locating/locating bearing arrangement on both sides, with the result that high compressive forces occur in the spindle. These can lead to buckling of the spindle and overloading of the screw drive bearings.

Tensioning of the threaded spindle

In order to avoid overloading of the screw drive bearings, threaded spindles with a locating/locating bearing arrangement on both sides are elastically tensioned while they are cold. This is carried out during mounting by means of either a precision locknut and a calculated tightening torque, *Figure 41*, page 42, or a shim with a calculated adjustment gap, *Figure 42*, page 42.

The dimension for tensioning must be calculated for each application and is based on the expected mean operating temperature of the threaded spindle. The threaded spindle is tensioned while cold to at least the extent that no compressive forces will occur in the spindle when warm from operation. For design, either our calculation service or the free-of-charge calculation tool BEARINX-online accessible via an Internet connection can be used.

Highly dynamic or difficult-to-predict operating conditions

If considerable heating is present or the spindle operating conditions cannot be defined, tempering of the threaded spindle is necessary.

Alternatively, the design principle based on a rotating screw drive nut has proved effective. This is a technically high quality and high performance solution that allows higher tensioning forces to be used, since these do not have an effect on the bearing rating life, *Figure 43*, page 43.

In more economical solutions, the compressive forces occurring in the spindle change the function of the bearing arrangement to a locating/non-locating bearing arrangement. If the losses in terms of axial rigidity and critical whirling speed are accepted, this method allows economical prevention of total failure of the system, *Figure 46* and *Figure 47*, page 46.

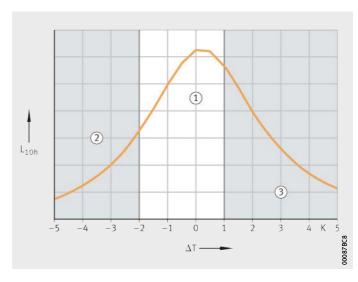
Influence of the design on bearing rating life

Through tensioning of the spindle, the screw drive bearings are subjected to high load on one side while cold. In this case, the triple row axial angular contact ball bearings DKLFA are suitable, since these have particularly high load carrying capacity on one side.

The bearing rating life is directly dependent on the deviation between the assumed design temperature for spindle tensioning and the actual mean operating temperature of the threaded spindle, *Figure 45*.

The following statements can be derived as a result:

- Cold starting of the machine reduces the bearing rating life, since the spindle tensioning forces then apply a high degree of load to the bearing positions.
- If the operating temperature actually present deviates too much from the operating temperature assumed for the design of spindle tensioning, this will reduce the bearing rating life.
- In applications where operating conditions are difficult to predict, unpredictably high heating of the threaded spindle can lead to compressive loading of the spindle with the risk of buckling.



 $\label{eq:L10h} \textit{L}_{10h} = \text{bearing rating life} \\ \Delta \textit{T} = \text{deviation of actual operating temperature from design temperature}$

① Optimum operating range
② Cold, high spindle tensioning forces
③ Compressive load in threaded spindle

Figure 45 Calculation of bearing rating life

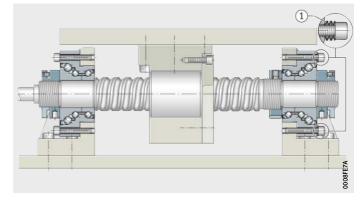






Economical overload protection in O arrangement

At one of the two bearing positions (generally the one opposing the drive), disc springs are added to the fixing screws and lightly preloaded with the screws, *Figure 46*. The screws are secured against loosening by means of suitable adhesive.

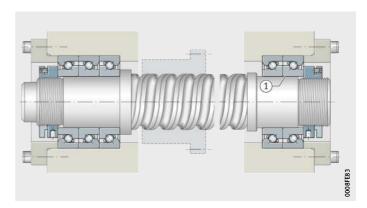


① Disc springs to DIN 2093; the quantity required must be calculated

Figure 46
Overload protection
by means of disc springs
(design example)

Economical overload protection in X arrangement

At one of the two bearing positions, the bearing seat on the shaft is designed as a sliding seat, *Figure 47*. If the spindle operating temperature changes within the scope of the design temperature for spindle tensioning, a locating/locating bearing arrangement is present. If higher temperatures occur, the spindle can expand without constraint, as a result of which the bearing arrangement becomes a locating/non-locating bearing arrangement. In order to prevent fretting corrosion at the sliding seat, suitable coatings such as hard chromium must be used.



(1) Sliding seat of bearing on shaft

Figure 47
Overload protection
by means of sliding seat
of bearing on shaft
(design example)

Design of adjacent construction

The adjacent construction (the shaft and housing) must be designed in accordance with the data in the dimension tables.

The abutment diameters for the shaft and housing shoulders d_a and D_a must be in accordance with the dimension tables.



 ${\rm D_a}$ and ${\rm d_a}$ are recommended minimum abutment diameters. If these values are not used, the rib diameter ${\rm d_1}$ according to the dimension table must be observed.

Pitch and quantity of fixing screws

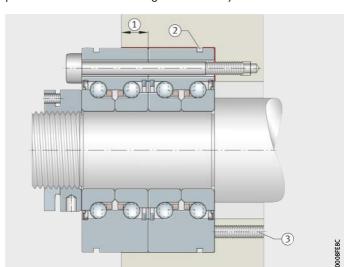
The data in the dimension tables must be observed:

- For the axial angular contact ball bearings ZKLF for screw mounting, the dimension tables give both the quantity and pitch of the fixing screws for the bearing as well as the quantity and pitch of the threaded fixing holes in the adjacent construction.
- In the case of the sizes of axial angular contact ball bearings ZKLF that are also available in the matched pair design 2AP, the individual bearings can have twice as many fixing holes in comparison with the adjacent construction. In these cases, the bearings are mounted leaving every second fixing hole unused.

For matched pairs of bearings

In matched pairs of axial angular contact ball bearings ZKLN and ZKLF, the depth of the housing bore must be defined such that the outer ring of the outermost bearing is radially supported to at least $^{1}/_{4}$ of its width.

The extraction slot for damage-free dismounting of the innermost bearing cannot be used, *Figure 48*, page 47. If the bearing is to be dismounted without damage, threaded extraction holes must be present in the screw mounting face of the adjacent construction.



ZKLF..-2AP

① Radial support
② Extraction slot not available for use
③ Thread optionally available
as threaded extraction hole
for damage-free bearing dismounting

Figure 48
Design example of adjacent construction for ZKLF..-2AP

0.0





Location of outer ring using ring nut for ZKLN, 7602, 7603, BSB

Axial angular contact ball bearings ZKLN, 7602, 7603 and BSB must be located clearance-free in the housing and on the shaft and must be axially preloaded during mounting. The axial preload force must be distributed evenly over the circumference in order to avoid deformation of the raceways.

Outer rings are axially tensioned to the preload force given in the dimension table by means of a ring nut (not included in the scope of delivery), *Figure 49*. The ring nut must be secured against loosening (for example using Loctite 638).

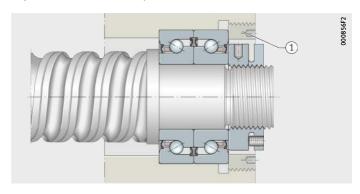
Ring nuts with an axial runout of max. 5 μ m apply the preload force evenly over the bearing rings and should therefore be used in preference to location by means of a cover.

ZKLN, 7602, 7603, BSB

(1) Ring nut

Figure 49
2 bearing set in X arrangement,
locknut AM, ring nut

Location of outer ring using a cover



Clamping of the outer rings using a cover and cap screws leads to deformation of the raceways, *Figure 50*. In order to minimise the deformation and achieve the calculated rating life:

- The cover should be designed with adequate rigidity
- The number of fixing screws should be selected in accordance with the load but at least four should be used
- Tighten the screws in a cross-wise sequence in four stages (finger tight, 40%, 70%, 100% of M_{Δ}).

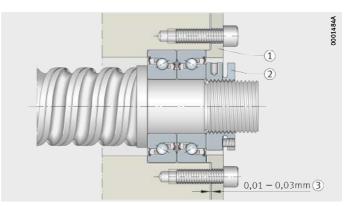


Observe the outer axial preload force in the dimension table. If other values are used, this will influence the bearing preload, bearing friction and heat generation in the bearing position.

ZKLN, 7602, 7603, BSB

① Cover ② Locknut ③ Gap before tightening of the cover fixing screws

Figure 50
2 bearing set in O arrangement,
locknut AM, cover



48 | TPI 123 Schaeffler Technologies

Radial centring of matched pairs of bearings ZKLF..-2AP screw mounted on a flat surface using a centring ring Matched pairs of axial angular contact ball bearings ZKLF..-2AP screw mounted on a flat surface must be radially centred, *Figure 51*.

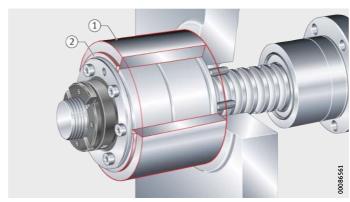
Radial centring of bearings

- ▶ Prepare a centring ring and push it over the joint between the two bearings. The ring must be in contact with the screw mounting face of the adjacent construction.
- ► Secure the ring axially using a commercially available retaining ring in the extraction groove of the bearing.

Dimensions of the ring

Guidelines on the dimensions of the ring, Figure 52:

- The dimensions D and d relate to the outside diameter D of the bearing, see dimension table.
- The width of the ring is the dimension l, see dimension table.



Centring ring
 Retaining ring

Figure 51
Radial centring of matched pair
of axial angular contact
ball bearings ZKLF
during mounting on face



Figure 52 Dimensions of centring ring

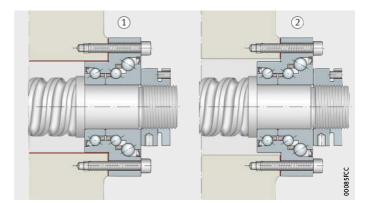






Screw mounting of triple row angular contact ball bearings Triple row angular contact ball bearings DKLFA are located on the adjacent construction as follows, Figure 53:

- in the centring bore
- on the face and thus radially capable of radial alignment using the linear guidance system and screw drive as a datum.



DKLFA

1) Bearing centred in adjacent construction 2 Bearing capable of radial alignment

> Figure 53 Mounting variants of bearing unit DKLFA

Speeds

The dimension tables give two different speed parameters:

- \blacksquare the maximum permissible limiting speed n_G
- \blacksquare the thermally safe operating speed n_{ϑ} for precision applications.

Limiting speed n_G

The limiting speed n_G is based on practical experience and test rig runs. It is the speed limit up to which the bearings can be operated as a maximum without kinematic processes in the rolling bearing leading to premature damage to the bearing. In bearings with contact seal lips 2RS, the speed is restricted by the seal material and its maximum permissible sliding velocity at the seal lip.

Under favourable environmental influences, higher limiting speeds can be achieved by using a bearing with minimal gap seals (suffix 2Z) instead of contact seals.

Thermally safe operating speed no

The limiting speed n_G described allows heating of the bearing, which is generally undesirable for precision applications such as in machine tools. The guide value n_ϑ proven in practice for maximum speeds in machine tool applications is an empirical value at which, for an operating duration of max. 25% and without external operating loads, the bearing temperature is max. 50 °C. In this case, only the bearing preload is present.

If the environmental conditions are known, the equilibrium temperature to be anticipated at the bearing position can be determined by means of a thermal balance calculation.







Friction

In most applications, preloading of bearings by means of the locknut tightening torque gives sufficiently accurate setting values. The reference here is the tightening torque M_A in accordance with the dimension tables in conjunction with an INA precision locknut.

The bearing frictional torque M_{RL} given in the dimension tables is a guide value for design of the drive. The actual values are generally lower. The values for bearing frictional torque relate in the case of axial angular contact ball bearings ZKLN, ZKLF, ZKLFA, DKLFA, 7602..-2RS, 7603..-2RS, BSB..-2RS and ZKLR to grease lubrication and are measured at a speed of n = 5 min⁻¹ and at room temperature (ϑ = 20 °C).

In the case of open axial angular contact ball bearings 7602, 7603 and BSB, the bearing frictional torque M_{RL} relates to lightly oiled raceways.

For dimensioning of the drive, the starting frictional torque and the frictional torque at high speeds in each case of $2\cdot M_{RL}$ to $3\cdot M_{RL}$ must be taken into consideration.

Seal friction

In axial angular contact ball bearings with contact seals (suffix 2RS), the extent of seal friction cannot be disregarded. Sealing influences the limiting speeds and the heating of the bearing.

Frictional energy

In any analysis of the thermal balance, the various operating speeds n_i must be taken into consideration with their time durations q_i .

The frictional energy N_R of the bearings can be calculated as follows:

$$N_{R} = \frac{M_{RL} \cdot n}{9.55}$$

 ${
m N_R}$ W Frictional energy ${
m M_{RL}}$ Nm Bearing frictional torque ${
m n}$ min $^{-1}$ Operating speed.

Lubrication

Bearings for screw drives can be lubricated with grease or oil. In machine tools, however, the mean bearing temperature should not exceed +50 °C. If this is the case, a lubrication method without heat dissipation such as grease lubrication or oil impulse lubrication can be selected.



For higher bearing temperatures and open bearings, recirculating oil lubrication should be used.

Grease lubrication

Bearings with a relubrication facility of series ZKLN, ZKLF, ZKLFA and DKLFA are greased with a lithium soap grease GA28 and are preferably run with grease lubrication. In most cases, the initial greasing is sufficient for the operating life of the bearings.

Under certain application conditions, relubrication may be necessary. In this case, Arcanol MULTITOP is suitable. The initial greasing is compatible with greases with a mineral oil base.

For checking by calculation of the grease operating life, relubrication interval and relubrication quantity on the basis of a duty cycle (speed, load, operating duration, environmental conditions), please contact us.

Relubrication interval

The relubrication intervals cannot be determined precisely in advance. They are essentially dependent on the operating conditions and the environmental influences such as temperature, contamination, dust, water, etc.



Bearings must always be relubricated:

- before and after long stoppage periods
- in conditions of high humidity
- within the defined lubrication intervals as stated in the technical proposal letter.

For the following conditions, please contact us:

- stationary bearings
- vibrations
- very small oscillating movements.

Relubrication quantity

If axial angular contact ball bearings ZKLN, ZKLF, ZKLFA and DKLFA must be relubricated due to the application conditions, please contact us regarding the relubrication quantities.

Oil lubrication

Other lubrication methods such as oil impulse lubrication or recirculating oil lubrication are also possible. Oils CLP to DIN 51517-3 and HLP to DIN 51524-2 of ISO VG 32 to ISO VG 100 have proved effective for oil lubrication.

If oil impulse lubrication is intended for axial angular contact ball bearings ZKLN and ZKLF, gap seals are advantageous. They prevent contamination entering the bearing and allow oil to leave the bearing. This prevents overlubrication.







Mounting guidelines

Axial angular contact ball bearings are self-retaining and the individual bearing components are matched to each other. The inner rings must not be removed from the bearing during mounting and dismounting. If individual bearing components are removed from the bearing, please contact Schaeffler before carrying out any reassembly.

The characteristics of the bearings are only valid when used in combination with INA precision locknuts and the associated tightening torques given in the dimension tables.



During the mounting of bearings, mounting forces should be applied only to the bearing ring to be mounted. Mounting forces must never be directed through the rolling elements or sealing rings.

Bearings should only be mounted and dismounted in accordance with the Mounting and Maintenance Manual TPI 100.

Further information

- TPI 100, Bearings for Screw Drives, Mounting and Maintenance Manual
- Download and ordering: http://medien.schaeffler.de.

Location of inner ring using locknut

Axial angular contact ball bearings must be axially preloaded during mounting by means of a precision locknut.

The performance data given in the dimension tables are only valid if the specified preload forces are observed. The tightening torques for INA precision locknuts required for this purpose are also given in the dimension tables.

The tightening torques for the individual bearing sizes are only valid for the INA precision locknuts listed.

For locating/non-locating bearing arrangements and bearing arrangements for rotating ball screw drive nuts, the tightening torques can be taken directly from the dimension tables.

For locating/locating bearing arrangements, the value in the dimension table can only be used for the locknut ①, *Figure 54*, page 55, and *Figure 55*, page 55. For the other locknuts ② and ③, the tightening torques must be calculated using BEARINX, taking account of the application-specific design criteria.



Calculation can be carried out free of charge using BEARINX-online Easy BallScrew or can be requested. The tightening torques calculated must be observed.

54 | **TPI 123** Schaeffler Technologies

Preventing settling

In order to counteract settling, the following tightening sequence must be observed:

- ► Tighten the precision locknut to twice the tightening torque M_A and then relieve the load again.
- \blacktriangleright Tighten the precision locknut to the stated tightening torque M_A .
- ► Secure the precision locknut against rotation by the torquecontrolled tightening of the grub screws.

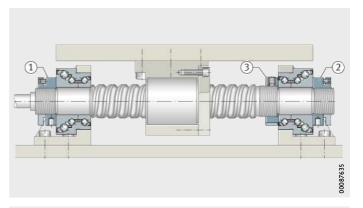


If other suitable locknuts are used, the manufacturer's guidance on calculation of the necessary tightening torque must be observed. Only use locknuts with a minimum radial runout accuracy of the end face to the thread of 5 μ m.



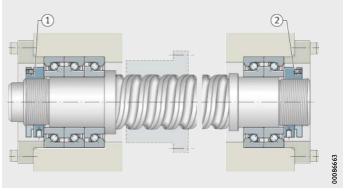
1) MA taken from dimension table (2), (3) (3) (4) M_A calculated using BEARINX

Figure 54 Tightening torques of locknuts in locating/locating bearing arrangement for angular contact ball bearing units



1 M_A taken from dimension table (2) M_A calculated using BEARINX

Figure 55 Tightening torques of locknuts in locating/locating bearing arrangement for angular contact ball bearing units in X arrangement



Schaeffler Technologies TPI 123







Fixing screws

For location of the outer ring, screws of grade 10.9 should be used. Screws should be fully tightened with torque control in accordance with the manufacturer's instructions and in a crosswise sequence.

If the bearing outer ring is supported by an additional housing cover, it must be ensured that the fixing screws are sufficiently well dimensioned.

Radial alignment of bearing position

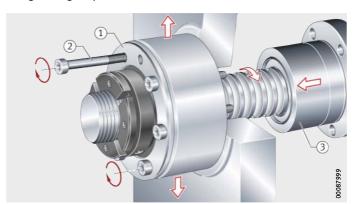
Radial alignment of the bearing position to the threaded spindle with axial angular contact ball bearings ZKLF, ZKLFA, DKLFA and ZKLR, *Figure 56* and *Figure 57*, page 57:

- Screw mount the bearing directly on the adjacent construction. Tighten the screws only loosely, since the bearing must still be capable of radial displacement.
- ▶ Move the screw drive nut towards the bearing position (the position of the screw drive is the datum via the linear guidance system, the nut serves as a functional element for alignment). The bearing will align itself to the optimum radial position (due to the constraining forces exerted by the datum).
- ► Tighten the fixing screws in a crosswise sequence to the specified tightening torque.

① Axial angular contact ball bearing ZKLF ② Fixing screws ③ Screw drive nut

Figure 56

Radial alignment
of the bearing position
to the threaded spindle with axial
angular contact ball bearings
suitable for screw mounting



Mounting of angular contact ball bearing units

Angular contact ball bearing units ZKLR require no additional preload after mounting, see page 56. They can often therefore be located on the threaded spindle simply by means of clearance-free clamping, Figure 57.



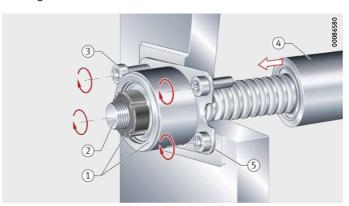
The type of axial location depends on the load to be supported. The adjacent construction can be a milled flat face or, if necessary, even an unmachined screw mounting surface without radial centring.



ZKLR

(1) Bearing unit with locknut (2) Threaded spindle ③ Tightening of fixing screws finger tight 4 Nut of screw drive **5** Tightening of fixing screws

> Figure 57 Mounting of bearing unit ZKLR









Calculation example of screw drive bearing ZKLF30100-2Z-XL

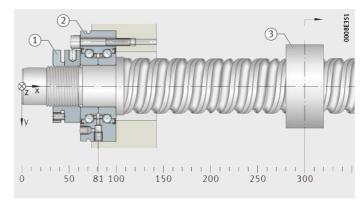
Arrangements of screw drive bearings with a locating bearing arrangement on one side and a locating/non-locating bearing arrangement can be calculated, unless the bearing arrangement is subjected to tilting moment load, using either the following equations or with the calculation software BEARINX-online Easy BallScrew.

If a tilting moment load on the bearing arrangement is to be taken into consideration or statically overdefined systems such as arrangements with locating/locating bearings, calculation must be carried out using the calculation program Bearinx. This also applies if, instead of the basic rating life L_{10h} , the expanded rating life L_{nm} is to be calculated. In this case, please consult Schaeffler or use the freely available program Bearinx-online Easy BallScrew: http://bearinx-online-easy-ballscrew.schaeffler.com.

Given data

The given data for calculation of the rating life are as follows:

- arrangement of screw drive bearings with locating bearing arrangement on one side
- pure axial load.



① Locknut ② Axial angular contact ball bearing ZKLF30100-2Z-XL ③ Nut

Figure 58
Specified screw drive bearing arrangement

Operating parameters

Operating temperature Nominal viscosity at +40 °C Life exponent for ball bearings Contamination factor $\vartheta = +40 \text{ °C}$ $v_{40} = 55 \text{ mm}^2 \cdot \text{s}^{-1}$ p = 3

e_C = 0,5 (normal cleanliness)

= ZKLF30100-2Z-XL

Bearing data Axial angular contact ball bearing

 $\begin{array}{lll} \text{Basic dynamic load rating} & \text{C_a} = 65\,000 \\ \text{Basic static load rating} & \text{C_{0a}} = 108\,000 \\ \text{Fatigue limit load} & \text{C_u} = 6\,400\,\text{N} \end{array}$

Required Expanded rating life.

Permissible limiting load

The checking of limiting loads is carried out with the aid of the diagrams for determining the resultant bearing load $F_{a \text{ res}}$, see page 33.

The limiting load of 11 000 N taken from the diagrams is greater than the maximum operating load occurring $F_{aB\ max}=10\,500$ N, which means that all the rows in the load are under preload at maximum load.

Static load safety factor So

The static load safety factor S_0 of the bearing should be $\geqq 4$ in machine tools. Under pure axial load, P_0 = $F_{a\;res}\approx 10\;500\;N.$

$$S_0 = \frac{C_0}{P_0}$$

$$S_{0a} = \frac{108\,000\,N}{10\,500\,N} = 10,3$$

The result of the calculation shows that the static load safety factor is sufficiently high.

Dynamic forces due to the machining cycle

Calculation of the rating life L_{10h} is carried out using the values from the speed and load duty cycle, see table. The resultant bearing load $F_{a res}$ is determined from the diagram, *Figure 26*, page 34.

Speed and load spectrum for the machining cycle, dynamic values

Load case	Time proportion	Speed	Operating load	Resultant bearing load
i	q _i	n _i	F_{aB}	
	%	min ⁻¹	N	N
1 rapid traverse	15	3 000	1 000	5 000
2 acceleration	15	1 500	10 500	10 500
3 rough machining	55	500	6 000	7 100
4 fine machining	15	100	1 000	5 000

Basic rating life

The basic rating life in millions of revolutions is the fatigue life reached or exceeded by 90% of a sufficiently large group of apparently identical bearings before the first evidence of material fatigue develops.

Equivalent bearing load

The equivalent bearing load for variation in steps is calculated as follows:

$$\mathsf{P} = \mathsf{P} \frac{\mathsf{q}_1 \cdot \mathsf{n}_1 \cdot \mathsf{P}_1^\mathsf{p} + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z \cdot \mathsf{P}_z^\mathsf{p}}{\mathsf{q}_1 \cdot \mathsf{n}_1 + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z}$$

$$P_{a} = \sqrt[3]{ 15 \cdot 3000 \cdot 5000^{3} + 15 \cdot 1500 \cdot 10500^{3} \over 15 \cdot 3000 + 15 \cdot 1500} \dots = 7560 \text{ N}$$

$$15 \cdot 3000 + 15 \cdot 1500 - 1000^{3} + 15 \cdot 100 \cdot 5000^{3}$$

$$15 \cdot 500 \cdot 7100^{3} + 15 \cdot 100 \cdot 5000^{3}$$







Equivalent speed

The equivalent speed for variation in steps is calculated as follows:

$$n = \frac{q_1 \cdot n_1 + q_2 \cdot n_2 + ... + q_z \cdot n_z}{100}$$

$$n = \frac{15 \cdot 3\ 000 + 15 \cdot 1\ 500 + 55 \cdot 500 + 15 \cdot 100}{100} = 965\ min^{-1}$$

Rating life in revolutions

$$L_{10} = \left(\frac{C_a}{P_a}\right)^p$$

$$L_{10} = \left(\frac{65\,000}{7\,560}\right)^3 = 636 \cdot 10^6$$

Rating life in operating hours

$$L_{10h} = \frac{16666}{n} \cdot \left(\frac{C_a}{P_a}\right)^p$$

$$L_{10h} = \frac{16666}{965 \,\text{min}^{-1}} \cdot \left(\frac{65000}{7560}\right)^3 = 10\,975\,\text{h}$$

Expanded rating life calculation

Calculation of the basic rating life L only takes into consideration the influence of load at the rolling contacts. In addition to the load, the lubrication regime and stress increases due to contamination of the lubricant also have a considerable influence on the fatigue life of a rolling bearing.

The method for calculating the expanded rating life was fundamentally changed in 2007 with the revision of ISO 281. The term "expanded rating life" was included for the first time in ISO 281:2010 and replaced the "adjusted rating life".

Expanded rating life

$$\mathsf{L}_{\mathsf{nm}} = \mathsf{a}_1 \cdot \mathsf{a}_{\mathsf{ISO}} \cdot \mathsf{L}_{\mathsf{10}}$$

Life adjustment factor a_{ISO}

The life adjustment factor $a_{\rm ISO}$ takes into consideration the lubrication regime, the contamination of the lubrication and the fatigue limit of the raceway material, *Figure 59*.

$$a_{ISO} = f \left[\frac{e_C \cdot C_u}{P}, \kappa \right]$$

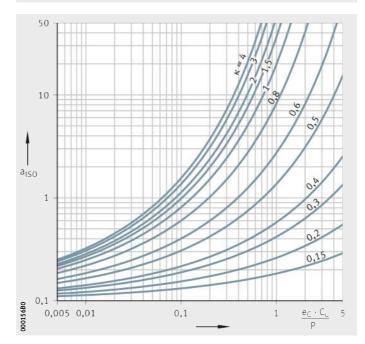


Figure 59 Life adjustment factor a_{ISO} for axial ball bearings







TPI 123 | 61

Calculation of the life adjustment factor a_{ISO} for load case 1

Based on the contamination factor $e_C = 0.5$ for normal cleanliness in accordance with ISO 281, the ratio $(e_C \cdot C_{\mu 1})/P_1$ is determined

Based on the nominal viscosity $v_{40} = 55 \text{ mm}^2 \cdot \text{s}^{-1}$, the operating temperature $\vartheta = 40$ °C and the mean bearing diameter $d_M = d_1$, the operating viscosity ν and the reference viscosity ν_1 are read from the diagrams, Figure 60 and Figure 61, page 63.



If values $\kappa > 4$ are determined, calculation should be continued in accordance with ISO 281 using a viscosity ratio $\kappa = 4$. If $\kappa < 0.1$, the calculation is not valid. In this case, please consult Schaeffler.

Ratio $(e_C \cdot C_u)/P_1$

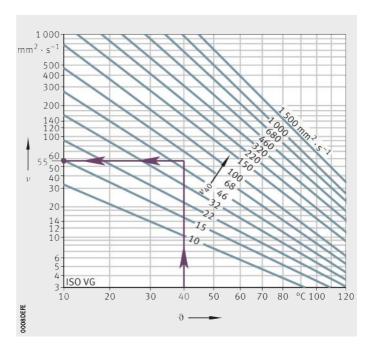
In conjunction with the fatigue limit load $C_{\rm u}$ and the equivalent load in load case 1, this gives:

$$\frac{e_C \cdot C_u}{P_1} = \frac{0.5 \cdot 6400 \text{ N}}{5000 \text{ N}} = 0.64$$

Viscosity ratio κ

The viscosity ratio $\kappa = \nu/\nu_1$ is calculated as follows:

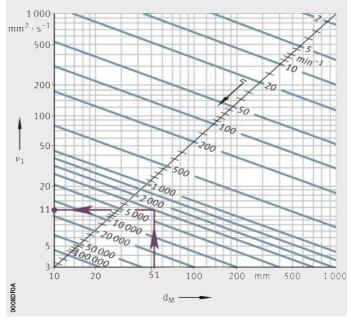
$$\kappa = \frac{55 \text{ mm}^2 \cdot \text{s}^{-1}}{12 \text{ mm}^2 \cdot \text{s}^{-1}} = 4,6$$

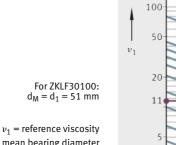


 ν/ϑ diagram for mineral oils

 ν = operating viscosity ϑ = operating temperature v_{40} = viscosity at +40 °C

Figure 60 Calculation of operating viscosity v at operating temperature ϑ





 d_M = mean bearing diameter n = speed

 $\begin{array}{c} \textit{Figure 61} \\ \text{Calculation of reference viscosity } \nu_1 \\ \text{Life adjustment factors a}_{\text{ISO}} \\ \text{for current operating condition} \end{array}$

Calculation of Load case 1 4 2 3 $(e_C \cdot C_u)/P$ 0,64 0,30 0,45 0,64 Operating viscosity ν 55 55 55 55 Reference viscosity v_1 12 16 32 130 Viscosity ratio $\kappa = \nu/\nu_1$ (5) 3,4 1,7 0,4 4 Life adjustment factor a_{ISO} a₁ a₂ a₃ a₄ 35 6,5 8,5 0,5





Calculation of expanded rating life

Calculation of the expanded rating life incorporates not only the life adjustment factor a_{ISO} but also a life adjustment factor for the reliability a₁. A reliability of 90% corresponds here to an achievable rating life with a commonly used, high quality material and under normal operating conditions that have proved sufficient over many years. In accordance with ISO 281, the life adjustment factor in this case is $a_1 = 1$.

Equivalent bearing load

The equivalent bearing load for a variation in steps is calculated, taking account of the life adjustment factors, as follows:

$$P = \sqrt[p]{\frac{1}{a_i} \cdot q_i \cdot n_i \cdot F_i^p + \dots + \frac{1}{a_z} \cdot q_z \cdot n_z \cdot F_z^p}$$
$$q_i \cdot n_i + \dots + q_z \cdot n_z$$

$$P_{a} = \sqrt[3]{\frac{\frac{1}{35} \cdot 15 \cdot 3000 \cdot 5000^{3} + \frac{1}{6,5} \cdot 15 \cdot 1500 \cdot 10500^{3}}{15 \cdot 3000 + 15 \cdot 1500}} \dots + \frac{1}{8,5} \cdot 55 \cdot 500 \cdot 7100^{3} + \frac{1}{0,5} \cdot 15 \cdot 100 \cdot 5000^{3}}{+55 \cdot 500 + 15 \cdot 100}$$

$$= 3.895 \text{ N}$$

Expanded rating life

The rating life is calculated as follows, see page 60 and page 61:

$$L_{10m} = \frac{16666}{965 \,\text{min}^{-1}} \cdot \left(\frac{65000 \,\text{N}}{3895 \,\text{N}}\right)^3 = 80264 \,\text{h}$$

Comparison with BEARINX-online Easy BallScrew

The catalogue calculation method in accordance with ISO 281 does not reflect the influence of bearing clearance, tilting or other influencing factors. As a result, deviations may arise compared to the more detailed calculation method in accordance with ISO/TS 16281, which is used in BEARINX-online.

In this example, calculation using Bearinx-online gives a minimum rating life $L_{h\,min}$ = $64\,729\,h.$

Calculation using Bearinx is not carried out with a static e_C value; the contamination factor is calculated for each load case in accordance with ISO 281. This gives a more accurate result in rating life calculation.



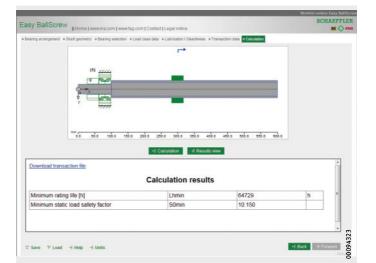
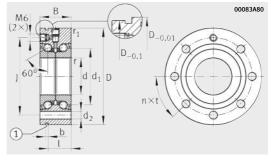


Figure 62
Calculation result using BEARINX-online

000

For screw mounting



ZKLF..-2RS, ZKLF..-2Z (d \leq 50 mm)



$\textbf{Dimension table} \cdot Dimens$	ions i	n mm						
Designation Mass			Dimensions					
	Ę.	m	d	D	В	d_1	r	r ₁
	X-life	≈ kg	-0,005	-0,01	-0,25		min.	min.
ZKLF1255-2RS	XL	0,37	12	55	25	25	0,3	0,6
ZKLF1255-2Z	XL	0,37	12	55	25	25	0,3	0,6
ZKLF1560-2RS	XL	0,43	15	60	25	28	0,3	0,6
ZKLF1560-2Z	XL	0,43	15	60	25	28	0,3	0,6
ZKLF1762-2RS	XL	0,45	17	62	25	30	0,3	0,6
ZKLF1762-2Z	XL	0,45	17	62	25	30	0,3	0,6
ZKLF2068-2RS	XL	0,61	20	68	28	34,5	0,3	0,6
ZKLF2068-2Z	XL	0,61	20	68	28	34,5	0,3	0,6

Performance data Designation Basic load ratings Fatigue Limiting speeds Bearing limit load frictional torque axial dyn. M_{RL} stat. C_{u} n_G grease ո_ϑ C_{a} C_{0a} ${\rm min}^{-1}$ min⁻¹ NmΧL ZKLF1255-2RS 18 600 24 700 9 2 0 0 0,16 1 460 3800 ZKLF1255-2Z ΧL 18600 24700 1460 12000 7600 0,08 ZKLF1560-2RS ΧL 19600 28 000 1650 8 2 0 0 3 5 0 0 0.2 ZKLF1560-2Z XL 19600 28 000 1650 10800 7000 0,1 ZKLF1762-2RS XL 20 700 0,24 31 000 1840 7600 3 3 0 0 20700 6600 ZKLF1762-2Z ΧL 10 100 0,12 31 000 1840 ZKLF2068-2RS ΧL 28 500 47 000 2750 6600 3 000 0,3 ZKLF2068-2Z XL 28 500 47 000 2750 8700 5 400 0,15

① Extraction slot; ② Adjacent construction, bearing screw mounted on flat surface; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition

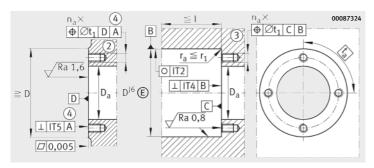
The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a.

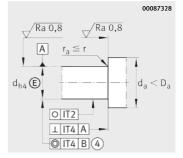
²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁴⁾ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





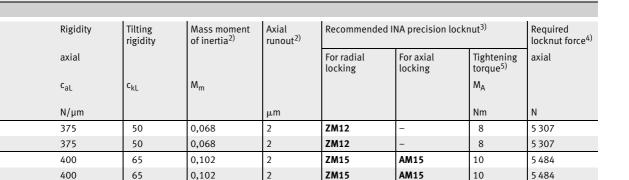
Housing design

Shaft design (threaded spindle)



				Pitch	Mounting o	limensions		Fixing screws ¹⁾ DIN EN ISO 4762 10.9		
J	d_2	b		$n \times t$	D _a	d _a	t_1	Size	$n_a \times t_a$	
					max.	min.				
42	6,8	3	17	3×120°	33	16	0,1	M6	3×120°	
42	6,8	3	17	3×120°	33	16	0,1	M6	3×120°	
46	6,8	3	17	3×120°	35	20	0,1	M6	3×120°	
46	6,8	3	17	3×120°	35	20	0,1	M6	3×120°	
48	6,8	3	17	6×60°	37	23	0,1	M6	3×120°	
48	6,8	3	17	6×60°	37	23	0,1	M6	3×120°	
53	6,8	3	19	8×45°	43	25	0,1	M6	4×90°	
53	6,8	3	19	8×45°	43	25	0,1	M6	4×90°	





ZM17

ZM17

ZM20

ZM20

AM17

AM17

AM20

AM20

15

15

18

18



450

450

650

650

80

80

140

140

0,132

0,132

0,273

0,273

2

2

2

2

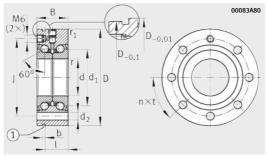
7 5 1 4

7 5 1 4

8 2 5 8

8 2 5 8

For screw mounting







ZKLF..-2RS, ZKLF..-2Z (d \leq 50 mm)

ZKLF30100

Dimension table (continued) ⋅ Dimensions in mm								
Designation Mass			Dimensions					
	Ę.	m	d	D	В	d_1	r	r ₁
	X-life	≈ kg	-0,005	-0,01	-0,25		min.	min.
ZKLF2575-2RS	XL	0,72	25	75	28	40,5	0,3	0,6
ZKLF2575-2Z	XL	0,72	25	75	28	40,5	0,3	0,6
ZKLF3080-2RS	XL	0,78	30	80	28	45,5	0,3	0,6
ZKLF3080-2Z	XL	0,78	30	80	28	45,5	0,3	0,6
ZKLF30100-2RS ²⁾	XL	1,63	30	100	38	51	0,3	0,6
ZKLF30100-2Z ²⁾	XL	1,63	30	100	38	51	0,3	0,6
ZKLF3590-2RS	XL	1,13	35	90	34	52	0,3	0,6
ZKLF3590-2Z	XL	1,13	35	90	34	52	0,3	0,6

Performance data

Designation Basic I axial		·	asic load ratings xial		Limiting speeds		Bearing frictional torque
	X-life	dyn. C _a	stat. C _{0a}	C _u	n _G grease	n_{ϑ} min^{-1}	M _{RL}
ZKLF2575-2RS	XL	30 500	55 000	3 250	5 700	2 600	0,4
ZKLF2575-2Z	XL	30 500	55 000	3 250	7 500	4700	0,2
ZKLF3080-2RS	XL	32 000	64 000	3 7 5 0	5 000	2 200	0,5
ZKLF3080-2Z	XL	32 000	64 000	3750	6700	4 300	0,25
ZKLF30100-2RS ²⁾	XL	65 000	108 000	6 400	4 500	2 100	0,8
ZKLF30100-2Z ²⁾	XL	65 000	108 000	6 400	5 600	4 000	0,4
ZKLF3590-2RS	XL	45 000	89 000	5 200	4 400	2 000	0,6
ZKLF3590-2Z	XL	45 000	89 000	5 200	5 800	3 800	0,3

① Extraction slot; ② Adjacent construction, bearing screw mounted on flat surface; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition

 $^{^{1)}}$ The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a .

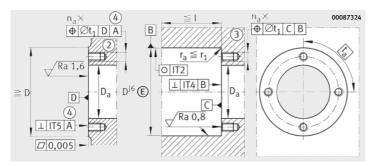
²⁾ Heavy series.

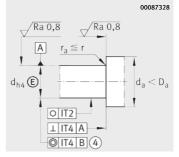
³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁵⁾ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





Housing design

Shaft design (threaded spindle)



				Pitch	Mounting dimensions			Fixing screws ¹⁾ DIN EN ISO 4762 10.9		
J	d_2	b	1	$n \times t$	D _a	d _a	t ₁	Size	$n_a \times t_a$	
					max.	min.				
58	6,8	3	19	8×45°	48	32	0,1	M6	4×90°	
58	6,8	3	19	8×45°	48	32	0,1	M6	4×90°	
63	6,8	3	19	12×30°	53	40	0,1	M6	6×60°	
63	6,8	3	19	12×30°	53	40	0,1	M6	6×60°	
80	8,8	3	30	8×45°	64	47	0,2	M8	8×45°	
80	8,8	3	30	8×45°	64	47	0,2	M8	8×45°	
75	8,8	3	25	8×45°	62	45	0,2	M8	4×90°	
75	8,8	3	25	8×45°	62	45	0,2	M8	4×90°	



	Rigidity	Tilting rigidity	Mass moment of inertia ³⁾	Axial runout ³⁾	Recommended INA	Required locknut force ⁵⁾		
	axial				For radial locking	For axial locking	Tightening torque ⁶⁾	axial
	c _{aL}	c _{kL}	M _m				M _A	
	N/µm			μm			Nm	N
	750	200	0,486	2	ZM25	AM25	25	9 1 2 3
	750	200	0,486	2	ZM25	AM25	25	9 1 2 3
•	850	300	0,73	2,5	ZM30	AM30	32	9 947
	850	300	0,73	2,5	ZM30	AM30	32	9 947

ZMA30/52

ZMA30/52

ZM35

ZM35

AM30

AM30

AM35/58

AM35/58



400

400

400

400

950

950

900

900

1,91

1,91

1,51

1,51

2,5

2,5

2,5

2,5

19 509

19 509

10770

10770

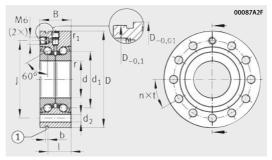
65

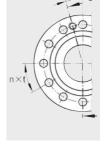
65

40

40

For screw mounting







ZKLF..-2RS, ZKLF..-2Z (d \leq 50 mm)

ZKLF40115 ZKLF50140

Dimension table (continued) · Dimensions in mm Designation Mass Dimensions									
Designation	Dimension	Dimensions							
	و	m	d	D	В	d_1	r	r ₁	
	X-life	≈ kg	-0,005	-0,01	-0,25		min.	min.	
ZKLF40100-2RS	XL	1,46	40	100	34	58	0,3	0,6	
ZKLF40100-2Z	XL	1,46	40	100	34	58	0,3	0,6	
ZKLF40115-2RS ²⁾	XL	2,2	40	115	46	65	0,6	0,6	
ZKLF40115-2Z ²⁾	XL	2,2	40	115	46	65	0,6	0,6	
ZKLF50115-2RS	XL	1,86	50	115	34	72	0,3	0,6	
ZKLF50115-2Z	XL	1,86	50	115	34	72	0,3	0,6	
ZKLF50140-2RS ²⁾	XL	4,7	50	140	54	80	0,6	0,6	
ZKLF50140-2Z ²⁾	XL	4,7	50	140	54	80	0,6	0,6	

Performance data

Designation	Basic load rating		gs	Fatigue limit load	Limiting speed	s	Bearing frictional torque
	X-life	dyn. C _a	C _{0a}	C _u	n _G grease	n _ϑ	M _{RL}
	×	N	N	N	min ⁻¹	min ⁻¹	Nm
ZKLF40100-2RS	XL	47 500	101 000	6 0 0 0	4 000	1 800	0,7
ZKLF40100-2Z	XL	47 500	101 000	6000	5 200	3 300	0,35
ZKLF40115-2RS ²⁾	XL	79 000	149 000	8 800	3 500	1 600	1,3
ZKLF40115-2Z ²⁾	XL	79 000	149 000	8 800	4 400	3 100	0,65
ZKLF50115-2RS	XL	51 000	126 000	7 400	3 200	1 500	0,9
ZKLF50115-2Z	XL	51 000	126 000	7 400	4 200	3 000	0,45
ZKLF50140-2RS ²⁾	XL	125 000	250 000	14800	2 900	1 200	2,6
ZKLF50140-2Z ²⁾	XL	125 000	250 000	14800	3 500	2 500	1,3

 $[\]textcircled{1} \textbf{ Extraction slot; @ Adjacent construction, bearing screw mounted on flat surface; @ Adjacent construction, bearing screw mounted in bore; @ Only valid in assembled condition } \\$

¹⁾ The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a.

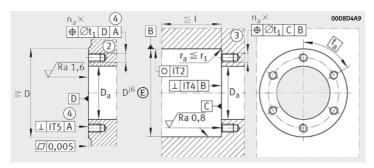
²⁾ Heavy series.

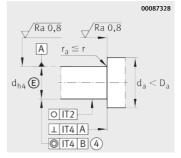
³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁵⁾ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





Housing design

Shaft design (threaded spindle)

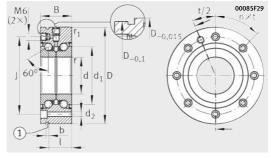


				Pitch	Mounting dimensions			Fixing screws ¹⁾ DIN EN ISO 4762 10.9		
J	d_2	b	1	$n \times t$	D _a	d _a	t ₁	Size	$n_a \times t_a$	
					max.	min.				
80	8,8	3	25	8×45°	67	50	0,2	M8	4×90°	
80	8,8	3	25	8×45°	67	50	0,2	M8	4×90°	
94	8,8	3	36	12×30°	80	56	0,2	M8	12×30°	
94	8,8	3	36	12×30°	80	56	0,2	M8	12×30°	
94	8,8	3	25	12×30°	82	63	0,2	M8	6×60°	
94	8,8	3	25	12×30°	82	63	0,2	M8	6×60°	
113	11	3	45	12×30°	98	63	0,2	M10	12×30°	
113	11	3	45	12×30°	98	63	0,2	M10	12×30°	





F	Rigidity	Tilting rigidity	Mass moment of inertia ³⁾	Axial runout ³⁾	Recommended INA	orecision locknu	t ⁴⁾	Required locknut force ⁵⁾
ć	axial				For radial locking	For axial locking	Tightening torque ⁶⁾	axial
C	aL	c _{kL}	M_{m}				M _A	
١	N/μm			μm			Nm	N
1	1000	550	2,26	2,5	ZM40	AM40	55	13 412
2	1000	550	2,26	2,5	ZM40	AM40	55	13 412
-	1 200	750	5,5	2,5	ZMA40/62	AM40	110	25 185
1	1 200	750	5,5	2,5	ZMA40/62	AM40	110	25 815
1	1 2 5 0	1000	5,24	2,5	ZM50	AM50	85	16 280
1	1250	1000	5,24	2,5	ZM50	AM50	85	16 280
1	1400	1500	15,2	2,5	ZMA50/75	AM50	150	28 451
3	1400	1500	15,2	2,5	ZMA50/75	AM50	150	28 451



ZKLF..-2Z (60 mm \leq d \leq 100 mm)



Dimension table (continue	ed) · [Dimensions in	mm						
Designation	Mass	Dimensions							
	<u>e</u>	m	d	D	В	d_1	r	r ₁	
	X-life	≈ kg	-0,008	-0,015	-0,25		min.	min.	
ZKLF60145-2Z	XL	4,3	60	145	45	85	0,6	0,6	
ZKLF70155-2Z	XL	4,9	70	155	45	95	0,6	0,6	
ZKLF80165-2Z	XL	5,3	80	165	45	105	0,6	0,6	
ZKLF90190-2Z	XL	8,7	90	190	55	120	0,6	0,6	
ZKLF100200-2Z	XL	9,3	100	200	55	132	0,6	0,6	
ZKLF100200-2Z	-	17,6	100	230	85	146	0,6	0,6	
		•	-	•	•	•		•	

Performance data	Performance data										
Designation				Fatigue limit load	Limiting speeds		Bearing frictional torque				
	X-life	dyn. C _a	stat. C _{Oa}	C _u	n _G grease	n _ϑ	M _{RL}				
	×	N	N	N	min ⁻¹	min ⁻¹	Nm				
ZKLF60145-2Z	XL	93 000	214 000	12 600	4 000	3 000	1				
ZKLF70155-2Z	XL	97 000	241 000	14 200	3 800	2800	1,2				
ZKLF80165-2Z	XL	100 000	265 000	15 400	3 600	2700	1,4				
ZKLF90190-2Z	XL	149 000	395 000	21 300	3 500	2300	2,3				
ZKLF100200-2Z	200-2Z XL 154 000		435 000	22 400	3 300	2150	2,6				
ZKLF100200-2Z	-	295 000	790 000	29 000	2 900	2000	3				

① Extraction slot; ② Adjacent construction, bearing screw mounted on flat surface; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition

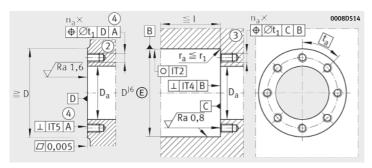
 $^{^{1)}}$ The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a .

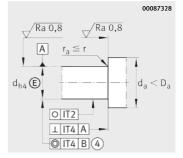
²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁴⁾ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





Housing design

Shaft design (threaded spindle)

£				-	ł
5	0		×	N	ı
	×	4	-	×	l
î T		-1	_	_	1

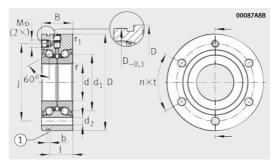
		Pitch	Mounting dimensions			Fixing screws ¹⁾ DIN EN ISO 4762 10.9			
J	d ₂	b	I	n×t	D _a max.	d _a min.	t ₁	Size	$n_a \times t_a$
120	8,8	3	35	8×45°	100	82	0,2	M8	8×45°
130	8,8	3	35	8×45°	110	92	0,2	M8	8×45°
140	8,8	3	35	8×45°	120	102	0,2	M8	8×45°
165	11	3	45	8×45°	138	116	0,2	M10	8×45°
175	11	3	45	8×45°	150	128	0,2	M10	8×45°
200	14	3	73	12×30°	175	130	0,4	M12	12×30°

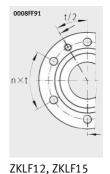




Rigidity	Tilting rigidity	Mass moment of inertia ²⁾	Axial runout ²⁾	Recommended INA pr	Required locknut force ⁴⁾		
axial				For radial locking	For axial locking	Tightening torque ⁵⁾	axial
c _{aL}	c _{kL}	M_{m}				M_A	
N/µm			μm			Nm	N
1300	1650	13,7	3	ZMA60/98	AM60	100	16700
1450	2250	19,8	3	ZMA70/110	AM70	130	19 031
1600	3000	27,6	3	ZMA80/120	AM80	160	20 604
1700	4400	59,9	3	ZMA90/130	AM90	200	22 731
1900	5800	85,3	3	ZMA100/140	AM100	250	25 624
2450	8200	185	3	_	AM100	500	52 000

For screw mounting Less stringent tolerances





ZKLF..-2RS-PE

$\textbf{Dimension table} \cdot \textbf{Dimension}$	s in mm									
Designation	Mass	Dimensions								
	m	d	D	В	d_1	r	r ₁			
	≈ kg	-0,01		-0,25		min.	min.			
ZKLF1255-2RS-PE	0,37	12	55 _{-0,013}	25	25	0,3	0,6			
ZKLF1560-2RS-PE	0,43	15	60 _{-0,013}	25	28	0,3	0,6			
ZKLF1762-2RS-PE	0,45	17	62 _{-0,013}	25	30	0,3	0,6			
ZKLF2068-2RS-PE	0,61	20	68 _{-0,013}	28	34,5	0,3	0,6			
ZKLF2575-2RS-PE	0,72	25	75 _{-0,013}	28	40,5	0,3	0,6			
ZKLF3080-2RS-PE	0,78	30	80 _{-0,013}	28	45,5	0,3	0,6			
ZKLF3590-2RS-PE	1,13	35	90 _{-0,015}	34	52	0,3	0,6			
ZKLF40100-2RS-PE	1,46	40	100 _{-0,015}	34	58	0,3	0,6			
ZKLF50115-2RS-PE	1,86	50	115-0,015	34	72	0,3	0,6			

Performance data							
Designation			Fatigue limit load	Limiting speed	S	Bearing frictional torque	
	dyn. C _a	stat. C _{0a} N	C _u	n_G grease min^{-1}	$\rm n_{\vartheta}$ $\rm min^{-1}$	M _{RL}	
ZKLF1255-2RS-PE	16 900	24 700	1 090	9 200	3 800	0,16	
ZKLF1560-2RS-PE	17 900	28 000	1 240	8 200	3 500	0,2	
ZKLF1762-2RS-PE	18 800	31 000	1 380	7 600	3 300	0,24	
ZKLF2068-2RS-PE	26 000	47 000	2 0 7 0	6 600	3 000	0,3	
ZKLF2575-2RS-PE	27 500	55 000	2 450	5 700	2 600	0,4	
ZKLF3080-2RS-PE	29 000	64 000	2 800	5 000	2 200	0,5	
ZKLF3590-2RS-PE	41 000	89 000	3 950	4 400	2 000	0,6	
ZKLF40100-2RS-PE	43 000	101 000	4 450	4 000	1 800	0,7	
ZKLF50115-2RS-PE	46 500	126 000	5 600	3 200	1 500	0,9	

 $[\]textcircled{1} \textbf{ Extraction slot; @ Adjacent construction, bearing screw mounted on flat surface; @ Adjacent construction, bearing screw mounted in bore; @ Only valid in assembled condition$

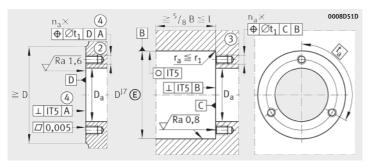
The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a.

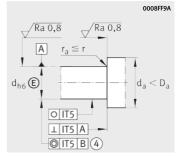
²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{\}rm 4)}\,$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





Housing design

Shaft design (threaded spindle)

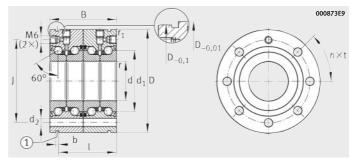


				Pitch	Mounting o	limensions		Fixing screws DIN EN ISO 47 10.9	
J	d ₂	b	_	n×t	D _a max.	d _a min.	t ₁	Size	$n_a \times t_a$
42	6,8	3	17	3×120°	33	16	0,1	M6	3×120°
46	6,8	3	17	3×120°	35	20	0,1	M6	3×120°
48	6,8	3	17	6×60°	37	23	0,1	M6	3×120°
53	6,8	3	19	8×45°	43	25	0,1	M6	4×90°
58	6,8	3	19	8×45°	48	32	0,1	M6	4×90°
63	6,8	3	19	12×30°	53	40	0,1	M6	6×60°
75	8,8	3	25	8×45°	62	45	0,2	M8	4×90°
80	8,8	3	25	8×45°	67	50	0,2	M8	4×90°
94	8,8	3	25	12×30°	82	63	0,2	M8	6×60°



Rigio	dity	Tilting rigidity		Axial runout ²⁾	Recommended	INA precision lockn		Required locknut force ⁴⁾
axia	l				For radial locking	For axial locking	Tightening torque ⁵⁾	axial
c_aL		c _{kL}	M_{m}				M_A	
N/µ	m			μm			Nm	N
37	5	50	0,068	5	ZM12	_	8	5 307
40	0	65	0,102	5	ZM15	AM15	10	5 484
45	0	80	0,132	5	ZM17	AM17	15	7 514
65	0	140	0,273	5	ZM20	AM20	18	8 258
75	0	200	0,486	5	ZM25	AM25	25	9 1 2 3
85	0	300	0,73	5	ZM30	AM30	32	9 947
90	0	400	1,51	5	ZM35	AM35/58	40	10 770
1 00	0	550	2,26	5	ZM40	AM40	55	13 412
1 25	0	1 000	5,24	5	ZM50	AM50	85	16 280

For screw mounting Matched pair





ZKLF..-2RS-2AP

$\textbf{Dimension table} \cdot \textbf{Dimensions i}$	in mm	l								
Designation			Dimensions							
	Ę.	m	d	D	В	d_1	r	r ₁		
	X-life	≈ kg	-0,005	-0,01	-0,25		min.	min.		
ZKLF1762-2RS-2AP	XL	0,9	17	62	50	30	0,3	0,6		
ZKLF2068-2RS-2AP	XL	1,22	20	68	56	34,5	0,3	0,6		
ZKLF2575-2RS-2AP	XL	1,44	25	75	56	40,5	0,3	0,6		
ZKLF3080-2RS-2AP	XL	1,56	30	80	56	45,5	0,3	0,6		
ZKLF3590-2RS-2AP	XL	2,26	35	90	68	52	0,3	0,6		
ZKLF40100-2RS-2AP	XL	2,92	40	100	68	58	0,3	0,6		
ZKLF50115-2RS-2AP	XL	3,72	50	115	68	72	0,3	0,6		

Performance data							
Designation		Basic load ratir	ngs	Fatigue limit load	Limiting speed	ds	Bearing frictional torque
	X-life	dyn. C _a	stat. C _{0a}	C _u	n _G grease	n _ϑ	M _{RL}
ZKLF1762-2RS-2AP	XL	33 500	62 000	3 700	7 600	3 300	0,36
ZKLF1/62-2K3-2AF	ΛL	33300	62 000	3700	7 600	3 300	0,36
ZKLF2068-2RS-2AP	XL	46 000	94 000	5 500	6 600	3 000	0,45
ZKLF2575-2RS-2AP	XL	49 000	111 000	6 500	5 700	2 600	0,6
ZKLF3080-2RS-2AP	XL	52000	127 000	7 500	5 000	2 200	0,75
ZKLF3590-2RS-2AP	XL	73 000	177 000	10 500	4 400	2 000	0,9
ZKLF40100-2RS-2AP	XL	77 000	202 000	11 900	4 000	1 800	1,05
ZKLF50115-2RS-2AP	XL	83 000	250 000	14 900	3 200	1 500	1,35

Further sizes and designs with seals 2Z available by agreement.

① Extraction slot; ② Adjacent construction, bearing screw mounted on flat surface; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition; ⑤ Threaded extraction holes, see page 47

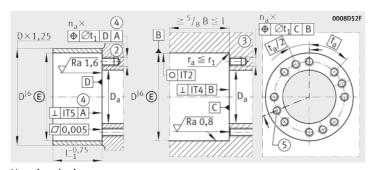
¹⁾ The screws are not included in the scope of delivery. Tightening torque in accordance with manufacturer's data. Number of holes in adjacent construction corresponds to t_a.

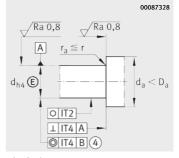
²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{}m 4)}$ The required axial locknut force must be observed when using other locknuts.

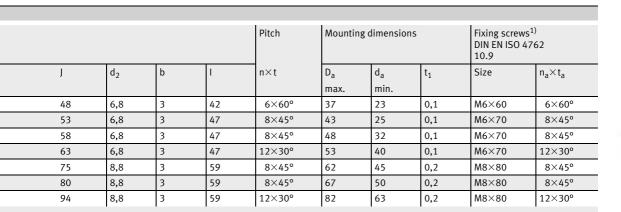
⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.

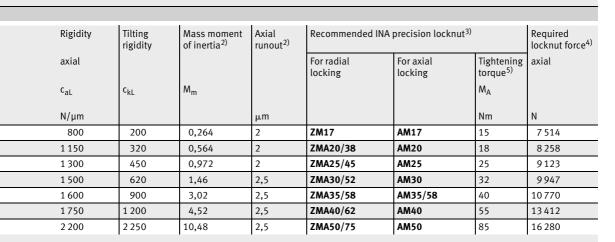




Housing design

Shaft design (threaded spindle)



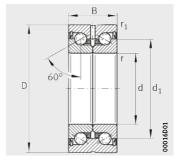


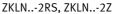






Not for screw mounting







$\textbf{Dimension table} \cdot Dis$	mens	ions in m	m										
Designation		Mass	Dimension	S					Mount dimen	_	Basic load	ratings	
											axial		
		m	d	D	В	d ₁	r	r ₁	Da	da	dyn.	stat.	
	X-life					_		_	_	C_a	C _{0a}		
	×	≈ kg		-0,010	-0,25		min.	min.	max.	min.	N	N	
ZKLN0619-2Z ⁵⁾	XL	0,02	6 +0,002 -0,003	19	12	12	0,3	0,3	16	9	5 400	6100	
ZKLN0624-2RS	XL	0,03	6 ^{+0,002} _{-0,003}	24	15	14	0,3	0,6	19	9	7 600	8 500	
ZKLN0624-2Z	XL	0,03	6 +0,002 -0,003	24	15	14	0,3	0,6	19	9	7 600	8 500	
ZKLN0832-2RS	XL	0,09	8 _{-0,005}	32	20	19	0,3	0,6	26	11	13 800	16 300	
ZKLN0832-2Z	XL	0,09	8 _{-0,005}	32	20	19	0,3	0,6	26	11	13 800	16 300	
ZKLN1034-2RS	XL	0,1	10 _{-0,005}	34	20	21	0,3	0,6	28	14	14 700	18 800	
ZKLN1034-2Z	XL	0,1	10 _{-0,005}	34	20	21	0,3	0,6	28	14	14700	18 800	
ZKLN1242-2RS	XL	0,2	12 _{-0,005}	42	25	25	0,3	0,6	33	16	18 600	24 700	
ZKLN1242-2Z	XL	0,2	12 _{-0,005}	42	25	25	0,3	0,6	33	16	18 600	24 700	
ZKLN1545-2RS	XL	0,21	15 _{-0,005}	45	25	28	0,3	0,6	35	20	19 600	28 000	
ZKLN1545-2Z	XL	0,21	15 _{-0,005}	45	25	28	0,3	0,6	35	20	19 600	28 000	
ZKLN1747-2RS	XL	0,22	17 _{-0,005}	47	25	30	0,3	0,6	37	23	20 700	31 000	
ZKLN1747-2Z	XL	0,22	17 _{-0,005}	47	25	30	0,3	0,6	37	23	20 700	31 000	
ZKLN2052-2RS	XL	0,31	20 _{-0,005}	52	28	34,5	0,3	0,6	43	25	28 500	47 000	
ZKLN2052-2Z	XL	0,31	20 _{-0,005}	52	28	34,5	0,3	0,6	43	25	28 500	47 000	
ZKLN2557-2RS	XL	0,34	25 _{-0,005}	57	28	40,5	0,3	0,6	48	32	30 500	55 000	
ZKLN2557-2Z	XL	0,34	25 _{-0,005}	57	28	40,5	0,3	0,6	48	32	30 500	55 000	
ZKLN3062-2RS	XL	0,39	30 _{-0,005}	62	28	45,5	0,3	0,6	53	40	32 000	64 000	
ZKLN3062-2Z	XL	0,39	30 _{-0,005}	62	28	45,5	0,3	0,6	53	40	32 000	64 000	
ZKLN3072-2RS ⁶⁾	XL	0,72	30 _{-0,005}	72	38	51	0,3	0,6	64	47	65 000	108 000	
ZKLN3072-2Z ⁶⁾	XL	0,72	30 _{-0,005}	72	38	51	0,3	0,6	64	47	65 000	108 000	
ZKLN3572-2RS	XL	0,51	35 _{-0,005}	72	34	52	0,3	0,6	62	45	45 000	89 000	
ZKLN3572-2Z	XL	0,51	35 _{-0,005}	72	34	52	0,3	0,6	62	45	45 000	89 000	

④ Only valid in assembled condition

¹⁾ Based on rotating inner ring.

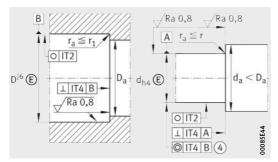
²⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{}m 3)}$ The required axial locknut force must be observed when using other locknuts.

⁴⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.

⁵⁾ No relubrication facility.

⁶⁾ Heavy series.



Design of housing and shaft (threaded spindle)

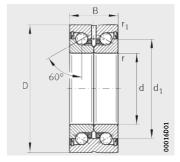
Fatigue L limit load		Limiting s	peeds	Bearing fric- tional	Rigid- ity	Tilting rigidity	Mass mo- ment of inertia ¹⁾	Axial run- out ¹⁾	Recommende locknut ²⁾	d INA precisio	on	Required locknut force ³⁾
				torque	axial				For radial locking	For axial locking	Tight- ening torque ⁴⁾	axial
C_{u}		n _G grease	n _ϑ	M _{RL}	c _{aL}	c _{kL}	M _m				M _A	
N		${\rm min}^{-1}$	min ⁻¹	Nm	N/µm	Nm/mrad	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N
36	0	22 800	14 000	0,01	150	4	0,0019	2	ZM06	_	1	2 010
50	0	16 400	6 800	0,04	200	8	0,0044	2	ZM06	-	2	2 404
50	0	19 900	12 000	0,02	200	8	0,0044	2	ZM06	_	2	2 404
96	0	12 100	5 100	0,08	250	20	0,02	2	ZM08	-	4	3 468
96	0	15 500	9 500	0,04	250	20	0,02	2	ZM08	_	4	3 468
1 11	0	10 900	4 600	0,12	325	25	0,029	2	ZM10	-	6	4 891
1 11	0	14 400	8 600	0,06	325	25	0,029	2	ZM10	-	6	4 891
1 46	0	9 200	3 800	0,16	375	50	0,068	2	ZM12	-	8	5 307
1 46	0	12 000	7 600	0,08	375	50	0,068	2	ZM12	-	8	5 307
1 65	0	8 200	3 500	0,2	400	65	0,102	2	ZM15	AM15	10	5 484
1 65	0	10800	7 000	0,1	400	65	0,102	2	ZM15	AM15	10	5 484
1 84	0	7 600	3 300	0,24	450	80	0,132	2	ZM17	AM17	15	7 514
1 84	0	10 100	6 600	0,12	450	80	0,132	2	ZM17	AM17	15	7 514
2 75	0	6600	3 000	0,3	650	140	0,273	2	ZM20	AM20	18	8 258
2 75	0	8 700	5 400	0,15	650	140	0,273	2	ZM20	AM20	18	8 258
3 25	0	5 700	2 600	0,4	750	200	0,486	2	ZM25	AM25	25	9 123
3 25	0	7 500	4 700	0,2	750	200	0,486	2	ZM25	AM25	25	9 123
3 75	0	5 000	2 200	0,5	850	300	0,73	2,5	ZM30	AM30	32	9 947
3 75	0	6700	4 300	0,25	850	300	0,73	2,5	ZM30	AM30	32	9 947
6 40	0	4 500	2 100	0,8	950	400	1,91	2,5	ZMA30/52	AM30	65	19 509
6 40	0	5 600	4 000	0,4	950	400	1,91	2,5	ZMA30/52	AM30	65	19 509
5 20	0	4 400	2 000	0,6	900	400	1,51	2,5	ZM35	AM35/58	40	10 770
5 20	0	5 800	3 800	0,3	900	400	1,51	2,5	ZM35	AM35/58	40	10 770

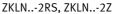






Not for screw mounting







Dimension table (cor	ntinu	ed) · Dim	ensions in m	m										
Designation		Mass	Dimensions	3					Mount dimen	-	Basic load	ratings		
											axial			
	ĺ	m	d	D	В	d_1	r	r ₁	Da	da	dyn.	stat.		
	Į.					u 1		1.1	a	a	C _a	C _{0a}		
	X-life	≈ kg			-0,25		min.	min.	max.	min.	N	N		
ZKLN4075-2RS	XL	0,61	40 _{-0,005}	75 _{-0,01}	34	58	0,3	0,6	67	50	47 500	101 000		
ZKLN4075-2Z	XL	0,61	40 _{-0,005}	75 _{-0,01}	34	58	0,3	0,6	67	50	47 500	101 000		
ZKLN4090-2RS ⁵⁾	XL	0,95	40 _{-0,005}	90_0,01	46	65	0,6	0,6	80	56	79 000	149 000		
ZKLN4090-2Z ⁵⁾	XL	0,95	40 _{-0,005}	90_0,01	46	65	0,6	0,6	80	56	79 000	149 000		
ZKLN5090-2RS	XL	0,88	50 _{-0,005}	90_0,01	34	72	0,3	0,6	82	63	51 000	126 000		
ZKLN5090-2Z	XL	0,88	50 _{-0,005}	90_0,01	34	72	0,3	0,6	82	63	51 000	126 000		
ZKLN50110-2RS ⁵⁾	XL	2,5	50 _{-0,005}	110_0,01	54	80	0,6	0,6	98	63	125 000	250 000		
ZKLN50110-2Z ⁵⁾	XL	2,5	50 _{-0,005}	110_0,01	54	80	0,6	0,6	98	63	125 000	250 000		
ZKLN60110-2Z	XL	2,2	60 _{-0,008}	110_0,015	45	85	0,6	0,6	100	82	93 000	214 000		
ZKLN70120-2Z	XL	2,4	70 _{-0,008}	120_0,015	45	95	0,6	0,6	110	92	97 000	241 000		
ZKLN80130-2Z	XL	2,7	80 _{-0,008}	130_0,015	45	105	0,6	0,6	120	102	100000	265 000		
ZKLN90150-2Z	XL	4,5	90 _{-0,008}	150_0,015	55	120	0,6	0,6	138	116	149 000	395 000		
ZKLN100160-2Z	XL	4,9	100 _{-0,008}	160_0,015	55	132	0,6	0,6	150	128	154 000	435 000		

④ Only valid in assembled condition

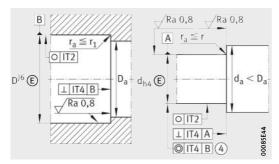
¹⁾ Based on rotating inner ring.

²⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{\}rm 3)}$ The required axial locknut force must be observed when using other locknuts.

⁴⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.

⁵⁾ Heavy series.



Design of housing and shaft (threaded spindle)

Fatigue limit load	Limiting	speeds	Bearing fric- tional	Rigidity	Tilting rigidity	Mass mo- ment of	Axial run- out ¹⁾	Recommended I locknut ²⁾	NA precisio	on	Required locknut force ³⁾
			torque	axial		inertia ¹⁾		For radial locking	For axial locking	Tight- ening torque ⁴⁾	axial
C _u	n _G grease	n _ϑ	M _{RL}	c _{aL}	c _{kL}	M _m				M _A	
N	${\rm min}^{-1}$	${\rm min}^{-1}$	Nm	N/µm	Nm/mrad	$kg \cdot cm^2$	μm			Nm	N
6 000	4 000	1 800	0,7	1 000	550	2,26	2,5	ZM40	AM40	55	13 412
6 000	5 200	3 300	0,35	1 000	550	2,26	2,5	ZM40	AM40	55	13 412
8 800	3 500	1 600	1,3	1 200	750	5,5	2,5	ZMA40/62	AM40	110	25 185
8 800	4 400	3 100	0,65	1 200	750	5,5	2,5	ZMA40/62	AM40	110	25 185
7 400	3 200	1 500	0,9	1 250	1 000	5,24	2,5	ZM50	AM50	85	16 280
7 400	4 200	3 000	0,45	1 250	1 000	5,24	2,5	ZM50	AM50	85	16 280
14 800	2900	1 200	2,6	1 400	1 500	15,2	2,5	ZMA50/75	AM50	150	28 451
14800	3 500	2 500	1,3	1 400	1 500	15,2	2,5	ZMA50/75	AM50	150	28 451
12 600	4000	3 000	1	1 300	1 650	13,7	3	ZMA60/98	AM60	100	16 700
14 200	3800	2800	1,2	1 450	2 250	19,8	3	ZMA70/110	AM70	130	19 031







15 400

21 300

22 400

3 6 0 0

3 5 0 0

3 300

2700

2 300

2 1 5 0

1,4

2,3

2,6

1600

1700

1 900

3 000

4 400

5 800

27,6

59,9

85,3

3

3

3

ZMA80/120

ZMA90/130

ZMA100/140

AM80

AM90

AM100

160

200

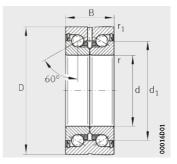
250

20 604

22731

25 624

Not for screw mounting Less stringent tolerances



ZKLN..-2RS-PE

$\textbf{Dimension table} \cdot Di$	limension table · Dimensions in mm													
Designation	Mass	Dimension	S					Moun dimer	_	Basic loa	d ratings	Fatigue limit load		
	m ≈ kg	d	D	B -0,25	d ₁	r min.	r ₁	D _a	d _a	dyn. C _a N	stat. C _{0a}	C _u		
ZKLN0624-2RS-PE	0,03	6 ^{+0,003} _{-0,005}	24_0,01	15	14	0,3	0,6	19	9	6 9 0 0	8 500	375		
ZKLN1034-2RS-PE	0,1	100,008	34_0,011	20	21	0,3	0,6	28	14	13 400	18 800	830		
ZKLN1242-2RS-PE	0,2	120,01	42_0,011	25	25	0,3	0,6	33	16	16 900	24 700	1 090		
ZKLN1545-2RS-PE	0,21	15_0,01	45_0,011	25	28	0,3	0,6	35	20	17 900	28 000	1 240		
ZKLN1747-2RS-PE	0,22	170,01	47_0,011	25	30	0,3	0,6	37	23	18 800	31 000	1 380		
ZKLN2052-2RS-PE	0,31	200,01	52_0,013	28	34,5	0,3	0,6	43	25	26 000	47 000	2 070		
ZKLN2557-2RS-PE	0,34	25 _{-0,01}	57_0,013	28	40,5	0,3	0,6	48	32	27 500	55 000	2 450		
ZKLN3062-2RS-PE	0,39	300,01	62_0,013	28	45,5	0,3	0,6	53	40	29 000	64 000	2 800		
ZKLN3572-2RS-PE	0,51	35 _{-0,01}	72 _{-0,013}	34	52	0,3	0,6	62	45	41 000	89 000	3 950		
ZKLN5090-2RS-PE	0,88	500,01	90_0,015	34	72	0,3	0,6	82	63	46 500	126 000	5 600		

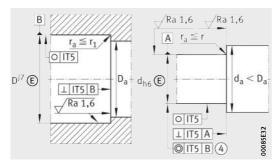
 $[\]textcircled{4}$ Only valid in assembled condition

¹⁾ Based on rotating inner ring.

²⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{\}rm 3)}\,$ The required axial locknut force must be observed when using other locknuts.

⁴⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

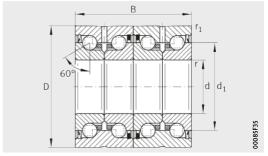
	Limiting spe	eeds	Bearing frictional torque	Rigidity	Tilting rigidity	Mass moment of inertia ¹⁾	Axial run- out ¹⁾	Recommendocknut ²⁾	ded INA precisio	on	Required locknut force ³⁾
				axial				For radial locking	For axial locking	Tight- ening torque ⁴⁾	axial
•	n _G grease	n _ϑ	M_{RL}	c _{aL}	c _{kL}	M _m				M _A	
	min ⁻¹	min ⁻¹	Nm	N/µm	Nm/mrad	kg⋅cm²	μm			Nm	N
	16 400	6 8 0 0	0,04	200	8	0,0044	5	ZM06 -		2	2 404
	10 900	4 600	0,12	325	25	0,029	5	ZM10	=	6	4891
	9 200	3 800	0,16	375	50	0,068	5	ZM12	=	8	5 307
	8 200	3 500	0,2	400	65	0,102	5	ZM15	AM15	10	5 484
	7 600	3 300	0,24	450	80	0,132	5	ZM17	AM17	15	7 514
	6 600	3 000	0,3	650	140	0,273	5	ZM20	AM20	18	8 258
	5 700	2 600	0,4	750	200	0,486	5	ZM25	AM25	25	9 1 2 3
	5 000	2 200	0,5	850	300	0,73	5	ZM30	AM30	32	9 947
	4 400	2 000	0,6	900	400	1,51	5	ZM35	AM35/38	40	10770
	3 200	1 500	0,9	1 250	1 000	5,24	5	ZM50	AM50	85	16 280







Not for screw mounting Matched pair







$\textbf{Dimension table} \cdot Dim$	ensio	ns in mr	n										
Designation		Mass	Dimensi	ons					Moun dimer	ting nsions	Basic load	d ratings	Fatigue limit load
	X-life	m ≈ kg	d -0,005	0,005 -0,01 -0,5 min. min.					D _a	d _a min.	dyn. C _a N	stat. C _{0a} N	C _u
ZKLN1747-2RS-2AP	XL	0,44	17	47	50	30	0,3	0,6	37	23	33 500	62 000	3 700
ZKLN2052-2RS-2AP	XL	0,62	20	52	56	34,5	0,3	0,6	43	25	46 000	94 000	5 500
ZKLN2557-2RS-2AP	XL	0,68	25	57	56	40,5	0,3	0,6	48	32	49 000	111 000	6 500
ZKLN3062-2RS-2AP	XL	0,78	30					53	40	52 000	127 000	7 500	
ZKLN3572-2RS-2AP	XL	1,02					0,6	62	45	73 000	177 000	10 500	
ZKLN4075-2RS-2AP	XL	1,22	40				0,3	0,6	67	50	77 000	202 000	11 900
ZKLN5090-2RS-2AP	XL	1,76	50	90	68	72	0,3	0,6	82	63	83 000	250 000	14 900

Further sizes and designs with seals 2Z available by agreement.

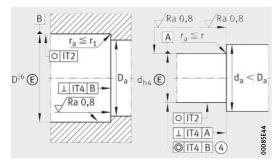
⁴ Only valid in assembled condition

¹⁾ Based on rotating inner ring.

²⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{\}rm 3)}$ The required axial locknut force must be observed when using other locknuts.

⁴⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

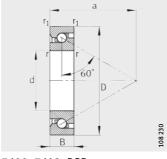
Limiting s _l	peeds	Bearing frictional torque	Rigidity	rigidity	Mass moment of inertia ¹⁾	Axial run- out ¹⁾	Recommended	INA precision	locknut ²⁾	Required locknut force ³⁾
			axial				For radial locking	For axial locking	Tight- ening torque ⁴⁾	axial
n _G grease	n _ϑ	M_{RL}	c _{aL}	c _{kL}	M _m				M _A	
${\rm min^{-1}}$	${\rm min^{-1}}$	Nm	N/µm	Nm/mrad	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N
7 600	3 300	0,36	800	200	0,264	2	ZM17	AM17	15	7 514
6 600	3 000	0,45	1 150	320	0,546	2	ZMA20/38	AM20	18	8 258
5 700	2 600	0,6	1 300	450	0,972	2	ZMA25/45	AM25	25	9 123
5 000	2 200	0,75	1 500	620	1,46	2,5	ZMA30/52	AM30	32	9 947
4 400	2 000	0,9	1 600	900	3,02	2,5	ZMA35/58	AM35/58	40	10770
4 000	1800	1,05	1750	1 200	4,52	2,5	ZMA40/62	AM40	55	13 412
3 200	1 500	1,35	2 200	2 2 5 0	10,48	2,5	ZMA50/75	AM50	85	16 280



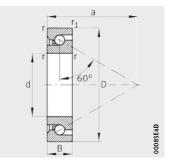




Single direction Open



7602, 7603, BSB



BSB..-SU

Dimension table · Dimensions in mm												
Designation	Mass	Dimension	S					Mounti	-	Basic load	d ratings	
	m	d	D	В	r	r ₁	a	D _a H12	d _a h12	dyn. C _a	stat. C _{0a}	
	\approx kg				min.	min.	~			N	N	
7602012-TVP	0,042	12_0,004	32_0,006	10_0,08	0,6	0,6	24	27	17	12 200	20 700	
7602015-TVP	0,052	15-0,004	35_0,006	11_0,08	0,6	0,6	27,5	30	20,5	13 100	24700	
7602017-TVP	0,074	17_0,004	40_0,006	12_0,08	0,6	0,6	31	34,5	23	17 200	32 500	
BSB1747-SU	0,13	17_0,004	47 _{-0,006}	15_0,12	1	0,6	29,9	43	27	26 000	47 000	
BSB1747-SU-L055	0,13	17_0,004	47 _{-0,006}	15_0,12	1	0,6	29,9	43	27	26 000	47 000	
7602020-TVP	0,139	20_0,005	47_0,006	14_0,12	1	1	36	39,5	27,5	19 100	38 000	
BSB2047-SU	0,12	20_0,005	47_0,006	15_0,12	1	0,6	29,9	43	27	26 000	47 000	
BSB2047-SU-L055	0,12	20_0,005	47 _{-0,006}	15_0,12	1	0,6	29,9	43	27	26 000	47 000	
7603020-TVP	0,17	20_0,005	52_0,007	15_0,12	1,1	1,1	39,5	43,5	30,5	25 500	53 000	
7602025-TVP	0,147	25_0,005	52_0,007	15-0,12	1	1	41	45	32	23 200	50 000	
BSB2562-SU	0,24	25_0,005	62_0,007	15-0,12	1	0,6	39,4	54	38	29 000	64 000	
BSB2562-SU-L055	0,24	25_0,005	62_0,007	15_0,12	1	0,6	39,4	54	38	29 000	64 000	
7603025-TVP	0,275	25_0,005	62_0,007	17 _{-0,12}	1,1	1,1	47,5	52	38	29 500	68 000	
BSB3062-SU	0,22	30_0,005	62_0,007	15_0,12	1	0,6	39,4	54	38	29 000	64 000	
BSB3062-SU-L055	0,22	30_0,005	62_0,007	15_0,12	1	0,6	39,4	54	38	29 000	64 000	
7602030-TVP	0,232	30_0,005	62_0,007	16_0,12	1	1	48	52,5	39,5	27 500	66 000	
7603030-TVP	0,409	30_0,005	72_0,007	19_0,12	1,1	1,1	55,5	61	45	36 500	89 000	

⁴ Only valid in assembled condition

¹⁾ Valid for single bearings with stated preload.

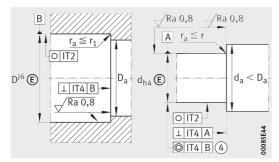
²⁾ Valid for matched pair of bearings in O or X arrangement.

³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{5)}\,}$ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

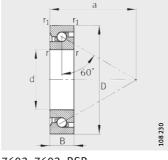
Fatigue limit load	Limiting spe	eds	Bearing frictional to	orque ¹⁾	Rigidity ²⁾	Axial run- out ³⁾	Recommended locknut ⁴⁾	I INA precisio	on	Required locknut force ⁵⁾
			lightly oiled	greased	axial		For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial
C_{u}	n _G grease	n _ð	M _{RL}	M _{RL}	c _{aL}				M _A	
N	min ⁻¹	min ⁻¹	Nm	Nm	N/µm	μm			Nm	N
 920	14 100	8000	0,015	-	476	2	ZM12	-	8	6 110
 1 090	12 300	6700	0,02	-	516	2	ZM15	AM15	10	5 740
1 430	10 600	6000	0,03	_	596	2	ZM17	AM17	15	8 060
2 070	8 700	6 200	_	0,2	764	2	ZM17	AM17	15	9 000
 2 070	8 700	6 200	_	0,2	764	2	ZM17	AM17	15	9 000
1 670	9 200	5 000	0,05	-	703	2	ZM20	AM20	18	8 490
2 070	8 700	6 200	_	0,2	764	2	ZM20	AM20	18	9 000
2 070	8 700	6 200	-	0,2	764	2	ZM20	AM20	18	9 000
 2 360	8 200	4 500	0,06	_	787	2	ZM20	AM20	25	11 600
2 220	8 000	4 500	0,065	-	772	2	ZM25	AM25	25	9 430
2 800	6 700	5 400	_	0,25	1 001	2	ZMA25/45	AM25	30	11 810
2 800	6 700	5 400	_	0,25	1 001	2	ZMA25/45	AM25	30	11 810
3 000	6 800	3 800	0,085	-	917	2	ZM25	AM25	40	14 480
2 800	6 700	5 400	-	0,25	1 034	2,5	ZM30	AM30	32	11 080
2 800	6 700	5 400	_	0,25	1 034	2,5	ZM30	AM30	32	11 080
2 900	6 700	3 800	0,085	-	893	2	ZM30	AM30	32	10 240
3 950	5 700	3 200	0,13	_	1 073	2	ZM30	AM30	55	16 180



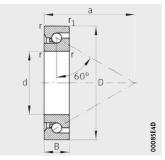




Single direction Open







BSB..-SU

Dimension table (continued) · Dimensions in mm												
Dimension table (co	ntinuea) ·	Dimensions	in mm									
Designation	Mass	Dimension	S					Mountin dimens		Basic loa	d ratings	
										axial		
	m	d	D	В	r	r ₁	а	D _a H12	d _a h12	dyn. C _a	stat.	
	≈ kg				min.	min.	≈			N	N	
BSB3572-SU	0,3	35_0,006	72_0,007	15_0,12	1	0,6	48,5	65	47	37 000	89 000	
BSB3572-SU-L055	0,3	35_0,006	72_0,007	15_0,12	1	0,6	48,5	65	47	37 000	89 000	
7602035-TVP	0,339	35 _{-0,006}	72_0,007	81 000								
7603035-TVP	0,546	35_0,006	80_0,007	17 _{-0,12} 21 _{-0,12}	1,5	1,5	61,5	67	51	38 000	100 000	
BSB4072-SU	0,26	40_0,006	72_0,007	15-0,12	1	0,6	48,5	65	47	37 000	89 000	
BSB4072-SU-L055	0,26	40_0,006	72_0,007	15_0,12	1	0,6	48,5	65	47	37 000	89 000	
7602040-TVP	0,418	40_0,006	80_0,007	18_0,12	1,1	1,1	62,5	69,5	53,5	39 000	106 000	
BSB040090-T	0,65	40_0,006	90_0,008	20_0,12	1,5	1,5	67	75,5	56,5	52 000	138 000	
7603040-TVP	0,751	40_0,006	90_0,008	23_0,12	1,5	1,5	68,5	75,5	56,5	52 000	138 000	
BSB045075-T	0,26	45_0,006	75 _{-0,007}	15-0,12	1	1	59,5	68	52	30 000	85 000	
7602045-TVP	0,488	45 _{-0,006}	85_0,008	19_0,12	1,1	1,1	66	73	57	39 500	111 000	
BSB045100-T	0,81	45_0,006	100_0,008	20_0,12	1,5	1,5	75	85,5	64,5	62 000	172 000	
7603045-TVP	0,992	45_0,006	100_0,008	25-0,12	1,5	1,5	77,5	85,5	64,5	62 000	172 000	
7602050-TVP	0,557	50_0,006	90_0,008	20_0,12	1,1	1,1	71,5	79	63	41 000	122 000	
BSB050100-T	0,75	50_0,006	100_0,008	20_0,12	1,5	1,5	75	85,5	64,5	62 000	172 000	
7603050-TVP	1,29	50_0,006	110_0,008	27_0,12	2	2	85,5	94	72	72 000	203 000	
BSB055090-T	0,38	55 _{-0,007}	90_0,008	15-0,15	1,0	1,0	70,5	80	65	33 500	98 000	
7602055-TVP	0,74	55 _{-0,007}	100_0,008	21_0,15	1,5	1,5	77,5	85,5	69,5	42 000	132 000	
BSB055120-T ⁷⁾	1,2	55_0,007	120_0,008	20_0,15	2,0	2,0	86	97,5	77	63 000	188 000	
7603055-TVP ⁷⁾	1,67	55_0,007	120_0,008	29_0,15	2	2	91,5	101	77	85 000	255 000	
BSB060120-T	1,1	60_0,007	120_0,008	20_0,15	1,5	1,5	88	100,5	79,5	64 000	196 000	

Only valid in assembled condition

¹⁾ Valid for single bearings with stated preload.

²⁾ Valid for matched pair of bearings in O or X arrangement.

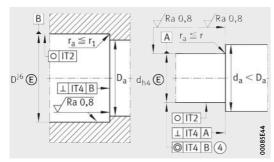
³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{5)}\,}$ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.

⁷⁾ Delivery possibility by agreement.



Design of housing and shaft (threaded spindle)

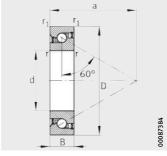
Fatigue limit load	Limiting sp			orque ¹⁾	Rigidity ²⁾	Axial run- out ³⁾	Recommen locknut ⁴⁾	ded INA prec	ision	Required locknut force ⁵⁾
			lightly oiled	greased	axial		For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial
C _u	n _G grease	n _ϑ	M_{RL}	M_{RL}	c _{aL}				M _A	
N	${\rm min^{-1}}$	${\rm min}^{-1}$	Nm	Nm	N/µm	μm			Nm	N
3 950	5 400	4 700	_	0,35	1 196	2,5	ZM35	AM35	50	15 220
3 950	5 400	4 700	-	0,35	1 196	2,5	ZM35	AM35	50	15 220
3 600	5 700	3 200	0,115	-	1 020	2	ZM35	AM35	40	11 310
4 450	5 100	3 000	0,17	-	1 192	2	ZM35	AM35	65	17 380
3 950	5 400	4 700	_	0,35	1 235	2,5	ZM40	AM40	60	15 650
3 950	5 400	4 700	_	0,35	1 235	2,5	ZM40	AM40	60	15 650
4 650	4 900	2 800	0,17	_	1 190	2	ZM40	AM40	55	13 330
6 100	4 500	3 100	0,225	-	1 292	2	ZM40	AM40	110	25 120
6 100	4 500	2 600	0,225	_	1 292	2	ZM40	AM40	110	25 120
3 750	5 100	3 300	0,13	-	1 072	2	ZM45	AM45	50	11 140
4 900	4700	2 600	0,19	_	1 247	2	ZM45	AM45	65	14 410
7 600	4 000	2 700	0,3	-	1 473	2	ZM45	AM45	120	25 160
7 600	4 000	2 200	0,3	_	1 473	2	ZM45	AM45	120	25 160
5 400	4 300	2 400	0,23	-	1 360	2	ZM50	AM50	85	16 810
7 600	4 000	2 700	0,33	_	1 473	2	ZM50	AM50	120	23 770
9 000	3 600	2 000	0,36	-	1 601	2	ZM50	AM50	150	28 930
4 300	4 200	2 800	0,19	-	1 246	3	ZM55	AM55	60	12 040
5 800	3 900	2 200	0,25	-	1 394	3	ZM55	AM55	85	15 060
8 300	3 400	2 400	0,36	_	1 553	3	ZM55	AM55	110	20 070
11 200	3 300	1 900	0,46	-	1 723	3	ZM55	AM55	130	24 120
8 700	3 800	2 300	0,38	-	1 623	3	ZM60	AM60	120	20 020



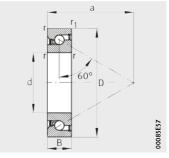




Single direction
Sealed on both sides



7602..-2RS, 7603..-2RS, BSB..-2RS



BSB..-2Z-SU

Dimension table · Dimensions in mm											
Designation	Mass	Dimension	Dimensions						Mounting dimensions		l ratings
	m	d	D	В	r,	r ₁	a	D _a H12	d _a h12	dyn. C _a	stat. C _{0a}
	\approx kg				min.	min.	~			N	N
7602012-2RS-TVP	0,042	12_0,004	32_0,006	10_0,08	0,6	0,6	24	27	17	12 200	20 700
7602015-2RS-TVP	0,052	15_0,004	35_0,006	11_0,08	0,6	0,6	27,5	30	20,5	13 100	24700
BSB1747-2Z-SU	0,13	17_0,004	47_0,006	15_0,12	1	0,6	29,9	43	27	26 000	47 000
7602020-2RS-TVP	0,12	20_0,005	47_0,006	14_0,12	1	1	36	39,5	27,5	19 100	38 000
BSB2047-2Z-SU	0,12	20_0,005	47_0,006	15-0,12	1	0,6	29,9	43	27	26 000	47 000
7603020-2RS-TVP	0,17	20_0,005	52_0,007	15-0,12	1,1	1,1	36,5	43,5	30,5	25 500	53 000
7602025-2RS-TVP	0,15	25_0,005	52_0,007	15-0,12	1	1	41	45	32	23 200	50 000
BSB2562-2Z-SU	0,24	25_0,005	62_0,007	15-0,12	1	0,6	39,4	54	38	29 000	64 000
7603025-2RS-TVP	0,27	25_0,005	62_0,007	17 _{-0,12}	1,1	1,1	47,5	52	38	29 500	68 000
BSB3062-2Z-SU	0,22	30_0,005	62_0,007	15_0,12	1	0,6	39,4	54	38	29 000	64 000
7602030-2RS-TVP	0,23	30_0,005	62_0,007	16-0,12	1	1	48	52,5	39,5	27 500	66 000
BSB3572-2Z-SU	0,3	35_0,006	72_0,007	15-0,12	1	0,6	48,5	65	47	37 000	89 000
BSB4072-2Z-SU	0,26	40_0,006	72_0,007	15_0,12	1	0,6	48,5	65	47	37 000	89 000

⁴ Only valid in assembled condition

¹⁾ Valid for single bearings with stated preload.

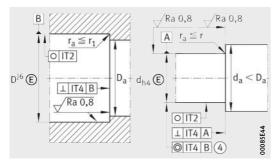
²⁾ Valid for matched pair of bearings in O or X arrangement.

³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁵⁾ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

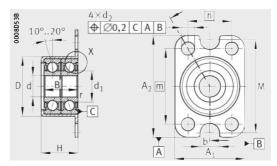
Fatigue limit load	Limiting spee	Limiting speeds f		Rigidity ²⁾	Axial runout ³⁾	Recommended	Required locknut force ⁵⁾		
				axial		For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial
C_{u}	n _G grease	n _ϑ	M_{RL}	c _{aL}				M _A	
N	min ⁻¹	min ⁻¹	Nm	N/µm	μm			Nm	N
920	10 400	8 000	0,018	476	2	ZM12	AM12	8	6110
1 090	9 000	6 700	0,024	516	2	ZM15	AM15	10	5 740
2 750	8 700	6 200	0,2	764	2	ZM20	AM20	18	9 000
1 670	6 800	5 000	0,06	703	2	ZM20	AM20	18	8 490
2 070	8 700	6 200	0,2	764	2	ZM20	AM20	18	9 000
2 360	6 200	4 500	0,07	787	2	ZM20	AM20	18	11600
2 220	5 100	3 800	0,08	772	2	ZM25	AM25	25	9 430
2 800	6 700	5 400	0,25	1 001	2	ZMA25/45	AM25	30	11810
3 000	5 100	3 800	0,1	917	2	ZM25	AM25	40	14480
2 800	6700	5 400	0,25	1 034	2,5	ZM30	AM30	32	11 080
2 900	5 000	3 800	0,1	893	2	ZM30	AM30	32	10 240
3 950	5 400	4 700	0,35	1196	2,5	ZM35	AM35	50	15 220
 3 950	5 400	4 700	0,35	1 235	2,5	ZM40	AM40	60	15 650







Angular contact ball bearing units



ZKLR0624-2Z, ZKLR0828-2Z

Dimension table ⋅ Dimensions in mm										
Designation	Mass	Dimensions								
	m	d	A ₁	A ₂	D	В	d_1	r	d ₂	
	≈ kg				+0,03 -0,01	±0,25		min.		
ZKLR0624-2Z	0,023	6_0,008	24	35	20,5	12	9	0,3	4,5	
ZKLR0828-2Z	0,03	8_0,008	28	35	23,9	14	11	0,3	4,5	
ZKLR1035-2Z	0,05	10_0,008	35	35	28,14	16	13,55	0,3	4,5	
ZKLR1244-2RS	0,12	12_0,007	44	50	35,45	20	16,6	0,3	6,6	
ZKLR1547-2RS	0,14	15_0,007	47	51	38,45	22	18	0,3	6,6	
ZKLR2060-2RS	0,3	20_0,008	60	60	50,45	28	24,4	0,3	6,6	

Performance data							
Designation	Basic load rating	gs			Fatigue limit	Limiting speed	
	radial		axial		load		
	dyn. C _r	stat. C _{Or}	dyn. C _a	stat. C _{0a}	C _u	n _G grease	
	N	N	N	N	N	min ⁻¹	
ZKLR0624-2Z	4 100	2060	1 650	1 520	104	7 300	
ZKLR0828-2Z	5 000	2700	2 0 6 0	1 850	138	6 200	
ZKLR1035-2Z	7 000	3 900	2 430	2 420	197	5 100	
ZKLR1244-2RS	13 600	8 500	13 200	17 900	430	3 700	
ZKLR1547-2RS	16 700	10700	16 400	22 400	540	3 400	
ZKLR2060-2RS	28 000	19 100	970	2 800			

 $[\]textcircled{2} \ \mathsf{Adjacent} \ \mathsf{construction}, \ \mathsf{bearing} \ \mathsf{screw} \ \mathsf{mounted} \ \mathsf{on} \ \mathsf{flat} \ \mathsf{surface}; \ \textcircled{4} \ \mathsf{Only} \ \mathsf{valid} \ \mathsf{in} \ \mathsf{assembled} \ \mathsf{condition}$

The screws are not included in the scope of delivery.
 Tightening torque according to manufacturer's data.

²⁾ Statistically determined mean values from measurements on the complete bearing. Due to the specific design, larger fluctuations in rigidity should be expected with axial angular contact ball bearing units ZKLR.

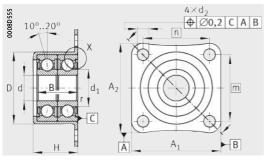
³⁾ Based on rotating inner ring.

 $^{^{4)}}$ The locknut is only used to axially locate the bearing unit. It has no influence on the bearing preload.

⁵⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁶⁾ Only valid in conjunction with INA precision locknuts.

⁷⁾ The geometrical tolerances stated are only required in the area of the diameter between Dai and Daa.



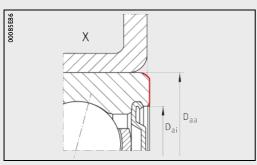
0008D577 D d d_1 \boxed{m} **♦** Ø0,2 C A B

ZKLR1244-2RS, ZKLR1547-2RS, ZKLR2060-2RS ZKLR1035-2Z

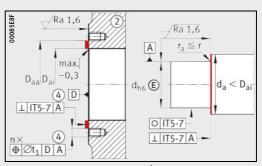


		Fixing screws ¹⁾ DIN EN ISO 4762								
b	m	n	М		d _a	D _{ai}	D _{aa}	Size	n	t ₁
				-0,5			min.			
6,6	26	15	32	13	8	16	19	M4 M6	4 2	0,2
6,6	26	20	35	15,5	10,4	18	22	M4 M6	4 2	0,2
_	26	26	-	17,5	12,4	22	26	M4	4	0,2
_	38	32	_	22	14	27	32	M6	4	0,2
=	39	35	-	24	17,5	29	35	M6	4	0,2
	47	47	-	30	24	39	47	M6	4	0,2

	Bearing frictional torque	Rigidity ²⁾	Mass moment of inertia ³⁾	Axial runout ³⁾	Recommended IN	commended INA precision locknut ⁴⁾⁵⁾		
		axial			For radial locking	For axial locking	Tightening torque ⁶⁾	
	M_{RL}	c _{aL}	M _m				M_A	
	Nm	N/μm	$\text{kg}\cdot\text{cm}^2$	μm			Nm	
	0,04	14	0,0014	7	ZM06	_	2	
	0,08	16	0,0028	7	ZM08	_	4	
	0,12	19	0,0075	7	ZM10	1	6	
•	0,16	170	0,0102	7	ZM12	_	8	
•	0,2	200	0,0178	7	ZM15	AM15	10	
	0,3	250	0,263	8	ZM20	AM20	18	



Axial abutment area of outer ring



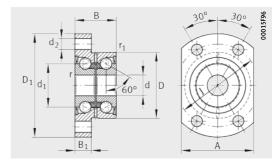
Design of housing and ${\rm shaft}^{7)}$







Double row axial angular contact ball bearings with flange



ZKLFA..-2RS, ZKLFA..-2Z

Dimension table · Dimen	sions in mm									
Designation	Mass	Dimensions								
	m	d	D	В	d_1	D_1	r			
	\approx kg		-0,01	-0,25			min.			
ZKLFA0630-2Z ²⁾	0,05	6 +0,002 -0,003	19	12	12	30	0,3			
ZKLFA0640-2RS	0,08	6 +0,002 -0,003	24	15	14	40	0,3			
ZKLFA0640-2Z	0,08	6 +0,002 -0,003	24	15	14	40	0,3			
ZKLFA0850-2RS	0,17	8 0 0 0	32	20	19	50	0,3			
ZKLFA0850-2Z	0,17	8 _0,005	32	20	19	50	0,3			

Performance data										
Designation	Basic load ratings		Fatigue Limiting speeds limit load			Bearing frictional torque				
	dyn. C _a	stat. C _{0a}	C _u	n _G grease	n _ϑ	M _{RL}				
	N	N	N	min ⁻¹	${\rm min^{-1}}$	Nm				
ZKLFA0630-2Z ²⁾	4 900	6 100	270	22 800	14 000	0,01				
ZKLFA0640-2RS	6 900	8 500	375	16 400	6 800	0,04				
ZKLFA0640-2Z	6 900	8 500	375	19 900	12 000	0,02				
ZKLFA0850-2RS	12 500	16300	720	12 100	5 100	0,08				
ZKLFA0850-2Z	10850-2Z 12 500 16 300		720	15 500	9 500	0,04				

 $[\]textcircled{2} \ \mathsf{Adjacent} \ \mathsf{construction}, \ \mathsf{bearing} \ \mathsf{screw} \ \mathsf{mounted} \ \mathsf{on} \ \mathsf{flat} \ \mathsf{surface}; \\ \textcircled{4} \ \mathsf{Only} \ \mathsf{valid} \ \mathsf{in} \ \mathsf{assembled} \ \mathsf{condition}$

¹⁾ The screws are not included in the scope of delivery. Tightening torque according to manufacturer's data.

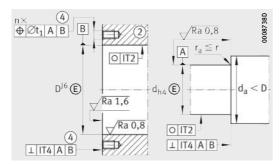
²⁾ No relubrication facility.

³⁾ Based on rotating inner ring.

⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{5)}}$ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

					Mounting dimensions			Fixing screws ¹⁾ DIN EN ISO 4762	
				10.9					
r ₁	B ₁	d_2	J	Α	da	d _a		Size	n
min.					min.	max.			
0,3	5	3,5	24	22	9	15	0,1	M3	4
0,6	6	4,5	32	27	9	18	0,1	M4	4
0,6	6	4,5	32	27	9	18	0,1	M4	4
0,6	8	5,5	40	35	11	25	0,1	M5	4
0,6	8	5,5	40	35	11	25	0,1	M5	4

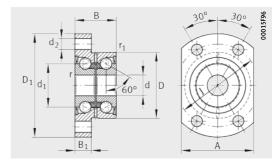
Rigidit	y Tilting rigidity	Mass moment	Axial runout ³⁾	Recommended IN	A precision lo		Required locknut force ⁵⁾
axial		of inertia ³⁾		For radial locking	For axial locking	Tightening torque ⁶⁾	axial
c_aL	c _{kL}	M _m				M _A	
N/µm	Nm/ mrad	kg⋅cm ²	μm			Nm	N
N/μm 150	Nm/ mrad	kg · cm ² 0,0019	μm 2	ZM06	_	Nm 2	N 2010
			•	ZM06 ZM06		2	
150	4	0,0019	2			2	2010
150 200	4 8	0,0019 0,0044	2 2	ZM06	-	2 2	2 010 2 404







Double row axial angular contact ball bearings with flange



ZKLFA..-2RS, ZKLFA..-2Z

Dimension table (continued) ⋅ Dimensions in mm									
Designation	Mass	Dimensions							
	m	d	D	В	d_1	D_1	r		
	≈ kg	-0,005	-0,01	-0,25			min.		
ZKLFA1050-2RS	0,18	10	32	20	21	50	0,3		
ZKLFA1050-2Z	0,18	10	32	20	21	50	0,3		
ZKLFA1263-2RS	0,3	12	42	25	25	63	0,3		
ZKLFA1263-2Z	0,3	12	42	25	25	63	0,3		
ZKLFA1563-2RS	0,31	15	42	25	28	63	0,3		
ZKLFA1563-2Z	0,31	15	42	25	28	63	0,3		
	•		•	·	•	· ·			

Performance data										
Designation	Basic load rating	rs	Fatigue limit load	Limiting speeds	Bearing frictional torque					
	dyn. C _a	stat. C _{0a}	C _u	n _G grease	n _ϑ	M _{RL}				
	N	N	N	min ⁻¹	min ⁻¹	Nm				
ZKLFA1050-2RS	13 400	18 800	830	10 900	4 600	0,12				
ZKLFA1050-2Z	13 400	18 800	830	14 400	8 600	0,06				
ZKLFA1263-2RS	16 900	24 700	1 090	9 200	3 800	0,16				
ZKLFA1263-2Z	16 900	24 700	1 090	12000	7 600	0,08				
ZKLFA1563-2RS	17 900	28 000	1 240	8 200	3 500	0,2				
ZKLFA1563-2Z	17 900	28 000	1 240	10 800	7 000	0,1				

²⁾ Adjacent construction, bearing screw mounted on flat surface; (4) Only valid in assembled condition

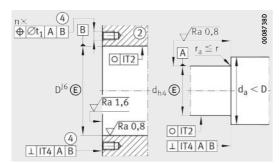
The screws are not included in the scope of delivery. Tightening torque according to manufacturer's data.

²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁴⁾ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Design of housing and shaft (threaded spindle)

						limensions		Fixing screws ¹⁾ DIN EN ISO 4762 10.9	
r ₁	B ₁	d_2	J	А	d _a				n
min.					min.	max.			
0,6	8	5,5	40	35	14	27	0,1	M5	4
0,6	8	5,5	40	35	14	27	0,1	M5	4
0,6	10	6,8	53	45	16	31	0,1	M6	4
0,6	10	6,8	53	45	16	31	0,1	M6	4
0,6	10	6,8	53	45	20	34	0,1	M6	4
0,6	10	6,8	53	45	20	34	0,1	M6	4

Rigidity		Mass	Axial	Recommended I	Required			
axial c _{aL}	rigidity c _{kL}	moment of inertia ²⁾ M _m	runout ²⁾	For radial locking	For axial locking	Tightening torque ⁵⁾ M _A	locknut force ⁴⁾ axial	
N/µm	Nm/ mrad	kg⋅cm²	μm			Nm	N	
325	25	0,029	2	ZM10	_	6	4 891	
325	25	0,029	2	ZM10	-	6	4 891	
375	50	0,068	2	ZM12	_	8	5 307	
375	50	0,068	2	ZM12	_	8	5 307	
400	65	0,102	2	ZM15	AM15	10	5 484	
400	65	0,102	2	ZM15	AM15	10	5 484	

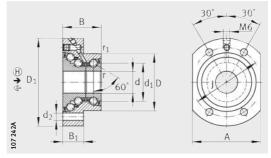






Triple row axial angular contact ball bearings with flange

For screw mounting



DKLFA..-2RS (d \leq 20 mm)

Attention!

The bearings require continuous load in the main load direction Θ .

Dimension table · Dimensions in mm														
Designation	Mass Dimension			Dimensions										ing sions
	m	d	D	В	d ₁	D_1	r	r ₁	B ₁	d ₂	J	А	d _a	
	≈ kg	-0,010	-0,013	-0,25			min.	min.					min.	max.
DKLFA1575-2RS	0,53	15	45	32	28	75	0,3	0,6	18	6,8	58	55	20	35
DKLFA2080-2RS	0,7	20	52	35	34,5	80	0,3	0,6	19	6,8	63	62	25	43
DKLFA2590-2RS	0,9	25	57	38	40,5	90	0,3	0,6	22	8,8	75	70	32	48
DKLFA30100-2RS	1	30	62	38	45,5	100	0,3	0,6	22	8,8	80	72	40	53
DKLFA30110-2RS ³⁾	2,5	30	75	56	51	110	0,3	0,6	35	8,8	95	85	47	64
DKLFA40115-2RS	1,5	40	72	42	58	115	0,3	0,6	23	8,8	94	90	50	67
DKLFA40140-2RS ³⁾	4,2	40	90	60	65	140	0,3	0,6	35	11	118	110	56	80

- ④ Only valid in assembled condition;
 ⑤ Adjacent construction, bearing screw mounted on flat surface;
 ⑥ Adjacent construction, bearing centred in adjacent construction

INA precision locknuts (accessories)

Triple row angular contact ball bearing	Recommended IN/	Recommended INA precision locknut ⁴⁾									
	For radial locking		For axial locking	Tightening torque ⁶⁾	axial						
	1,2	3	1,2	1	1						
				M _A Nm	N						
DKLFA1575-2RS	ZMA15/33	ZM17	AM15	10	6 270						
DKLFA2080-2RS	ZMA20/38	ZM25	AM20	18	8 580						
DKLFA2590-2RS	ZMA25/45	ZM30	AM25	25	9 6 7 0						
DKLFA30100-2RS	ZMA30/52	ZM35	AM30	32	10 350						
DKLFA30110-2RS	-	ZM35	AM30/65	65	20 500						
DKLFA40115-2RS	ZMA40/62	ZM45	AM40	55	13 420						
DKLFA40140-2RS	-	ZM45	AM40/85	110	26 600						

¹⁾ The screws are not included in the scope of delivery. Tightening torque according to manufacturer's data.

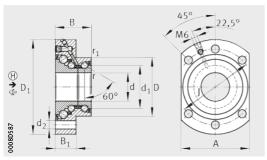
²⁾ Based on rotating inner ring.

³⁾ Heavy series.

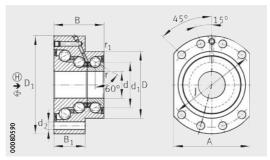
⁴⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁵⁾ The required axial locknut force must be observed when using other locknuts.

⁶⁾ Only valid in conjunction with INA precision locknuts in locating/non-locating bearing arrangements and for ① in locating/locating bearing arrangements. Note the guidelines on locating/locating bearing arrangements, see page 54.



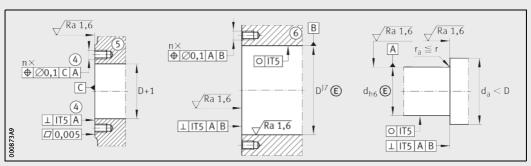
DKLFA..-2RS (d \geq 25 mm)



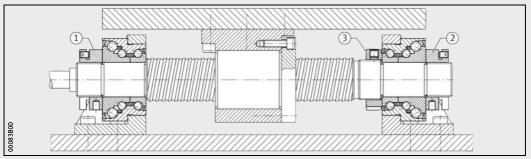
DKLFA..-2RS Heavy series



Fixing screws DIN EN ISO 47	sys ¹⁾ gue limit speeds		•	Bear- Rigidity ing fric-tional		Rigidity		ing mo- rigid- ment ity of		ment of	Axial run- out ²⁾			
10.9		axial	Œ	axial \oplus	→				torque axial		axial		inertia ²⁾	
Size	n	dyn. C _a	stat. C _{0a}	dyn. C _a	stat. C _{0a}	C _u	n _G grease	n _{\vartheta}	M _{RL}	c _{aL}	c _{aL}	c _{kL}	M _m	
		N	N	N	N	N	min ⁻¹	min ⁻¹	Nm	N/μm	N/µm	Nm/ mrad	kg⋅cm²	μm
M6	4	17 900	28 000	37 000	83 000	2 480	5 700	2 600	0,35	500	950	140	0,278	5
M6	4	26 000	47 000	44 500	110 000	2 850	5 000	2 200	0,45	750	1 100	260	0,553	5
M8	6	27 500	55 000	52 000	144 000	4 000	4 400	2 000	0,6	850	1 200	370	1,12	5
M8	6	29 000	64 000	55 000	165 000	4 5 5 0	4 000	1800	0,75	900	1 400	500	1,7	5
M8	8	59 000	108 000	106 000	257 000	6 700	3 500	1 600	1,5	1 300	1 600	650	3,23	5
M8	6	43 000	101 000	73 000	227 000	5 700	3 200	1 500	1	1 100	1 700	1000	4,23	5
M10	8	72 000	149 000	126 000	363 000	9 600	2 900	1 200	2,5	1 800	2 000	1 370	9,32	5



Design of housing and shaft (threaded spindle)



Ball screw drive spindle with locating bearing arrangement on both sides Numbers in circles: see table INA precision locknuts (accessories), see page 98











For screw mounting Not for screw mounting

		Page
Product overview	Needle roller/axial cylindrical roller bearings	102
Features	Radial and axial load carrying capacity	103
	Preload and internal clearance	103
	Bearings for screw mounting	103
	Bearings not for screw mounting	104
	Operating temperature	104
	Suffixes	104
Design and	Basic rating life	105
safety guidelines	Static load safety factor	105
	Resultant and equivalent bearing load	106
	Design of adjacent construction	108
	Speeds	108
	Friction	109
	Lubrication	109
	Mounting guidelines	111
Accuracy		112
Dimension tables	Needle roller/axial cylindrical roller bearings, light series, for screw mounting	114
	Needle roller/axial cylindrical roller bearings, heavy series, for screw mounting	118
	Needle roller/axial cylindrical roller bearings, light series, not for screw mounting	124
	Needle roller/axial cylindrical roller bearings, heavy series, not for screw mounting	128





Schaeffler Technologies TPI 123 | 101

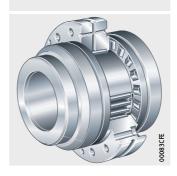
Product overview Needle roller/axial cylindrical roller bearings

For screw mounting



With stepped shaft locating washer extended on one side

ZARF..-L



Not for screw mounting

ZARN



With stepped shaft locating washer extended on one side

ZARN..-L



Features

Needle roller/axial cylindrical roller bearings comprise an outer ring with radial and axial raceways, shaft locating washers, an inner ring, a radial needle roller and cage assembly and axial cylindrical roller and cage assemblies. The bearings are available in versions for screw mounting and not for screw mounting.

Radial and axial load carrying capacity

In addition to radial forces, the bearings can also support axial forces from both directions and tilting moments.

Preload and internal clearance

The outer ring, inner ring and axial cages are matched to each other such that the bearing is axially clearance-free after preloading by means of an INA precision locknut. The radial internal clearance conforms to Group 2 in accordance with ISO 5753-1 (DIN 620-4).

Bearings for screw mounting

Needle roller/axial cylindrical roller bearings ZARF(L) have holes in the outer ring. These are used for screw mounting directly on the adjacent construction or in a radial locating bore, *Figure 1*.

Due to screw mounting of the outer ring, the cover that would otherwise be required and the matching work can therefore be dispensed with. The bearings are preloaded against the shaft shoulder by means of a locknut AM or ZM(A).

With seal carrier assembly

In the interests of a simpler design, a seal carrier assembly DRS, *Figure 1* is recommended. The seal carrier assembly is centred on the outer ring and seals the bearing against outside influences.

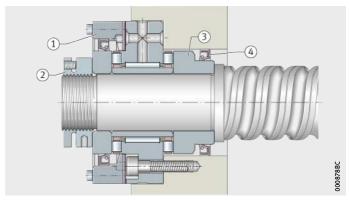
ZARF..-L

① Seal carrier assembly DRS
② Locknut AM
③ Stepped shaft locating washer
extended on one side

4 Rotary shaft seal DIN 3760-..-NBR

Figure 1
Stepped shaft locating washer with sealing ring

With stepped shaft locating washer



ZARF..-L has a stepped shaft locating washer extended on one side, *Figure 1*. These series are used in preference where the shaft locating washers are not adequately supported axially by the shaft shoulder or the bearing unit cannot be sealed on the outside surface of the normal shaft locating washer due to space conditions in the adjacent construction.

Heavy series

ZARF(L) is also available in a heavy series. This series has a larger cross-section for the same shaft diameter and therefore higher basic load ratings.

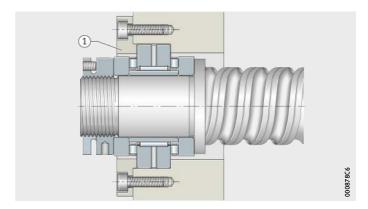




Schaeffler Technologies

Bearings not for screw mounting

Bearings ZARN(L) are mounted in a housing bore and the outer ring is located using a cover, *Figure 2*. They are preloaded against the shaft shoulder by means of a locknut AM or ZM(A).



ZARN

1) Cover

Figure 2
Bearing outer ring located using cover

With stepped shaft locating washer

ZARF..-L has a stepped shaft locating washer extended on one side, *Figure 3*. This series is used in preference where the shaft locating washer is not adequately supported axially by the shaft shoulder or the bearing unit cannot be sealed on the outside surface of the normal shaft locating washer due to space conditions in the adjacent construction.



① Stepped shaft locating washer extended on one side ② Rotary shaft seal DIN 3760

Figure 3
Stepped shaft locating washer with sealing ring

Heavy series

ZARN(L) is also available in a heavy series. This series has a larger cross-section for the same shaft diameter and therefore higher basic load ratings.

Operating temperature

The bearings are suitable for temperatures from -30 °C to +120 °C.

Suffixes

Suffixes for available designs, see table.

Available designs

Suffix	Description	Design
L	Stepped shaft locating washer extended on one side	Standard
TV	Axial bearing cages made from polyamide PA66	

104 | TPI 123 Schaeffler Technologies

Design and safety guidelines Basic rating life

The decisive factors in determining the bearing size are the basic rating life, the static load safety factor and the axial limiting load.

The rating life L_{10} or L_{10h} is calculated as follows:

$$L_{10} = \left(\frac{C}{P}\right)^{p}$$

$$L_{10h} = \frac{16666}{n} \cdot \left(\frac{C}{P}\right)^{p}$$

106 revolutions

Basic rating life in millions of revolutions

 $\begin{array}{cc} {\rm L_{10h}} & {\rm h} \\ {\rm Basic\ rating\ life\ in\ operating\ hours} \end{array}$

Basic dynamic radial or axial load rating, see dimension tables

 C_r is valid for a contact angle $\alpha \le 45^\circ$, C_a is valid for a contact angle $\alpha > 45^\circ$

P N Equivalent dynamic bearing load

Life exponent p = 10/3

Operating speed.

Static load safety factor

The static load safety factor S_0 indicates the security against impermissible permanent deformations in the bearing, see page 29:

$$S_0 = \frac{C_0}{P_0}$$

Solution Static load safety factor

Basic dynamic radial or axial load rating, see dimension tables

 C_{0r} is valid for a contact angle $\alpha \le 45^{\circ}$, C_{0a} is valid for a contact angle $\alpha > 45^{\circ}$

P₀ IN Maximum static load of bearing.



In machine tools, S_0 should be ≥ 4 .





Resultant and equivalent bearing load

The resultant axial bearing load $F_{a \text{ res}}$ is determined from the axial operating load F_{aB} and taking account of the axial preload, *Figure 4*, page 107, to *Figure 6*, page 107.

Under pure axial load, $P = F_{a res}$. If additional radial operating loads are present, these must be calculated separately using the basic radial load ratings. The limiting loads up to which the axial load can be supported without clearance are shown, *Figure 4*, page 107, to *Figure 6*, page 107.

The needle roller/axial cylindrical roller bearings ZARN and ZARF can be subjected to higher axial preload in order to increase the lift-off load. In this case, it must be taken into consideration that this will cause an increase in friction and temperature as well as a reduction in the basic rating life. The increase in preload is achieved by using a higher tightening torque for the locknut.



In the case of roller bearings, the lift-off load must not be exceeded during highly dynamic acceleration events. As a result, higher wear will occur under rapid acceleration.

For extreme moment loads and statically overdefined systems (locating/locating bearing arrangements), please contact us. The calculation program BEARINX can give a precise design in this case.

Load varying in steps

In this case, P and n are calculated as follows:

$$\mathsf{P} = \sqrt[p]{\frac{\mathsf{q}_1 \cdot \mathsf{n}_1 \cdot \mathsf{P}_1^\mathsf{p} + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z \cdot \mathsf{P}_z^\mathsf{p}}{\mathsf{q}_1 \cdot \mathsf{n}_1 + \ldots + \mathsf{q}_z \cdot \mathsf{n}_z}}$$

$$n = \frac{q_1 \cdot n_1 + \dots + q_z \cdot n_z}{100}$$

N

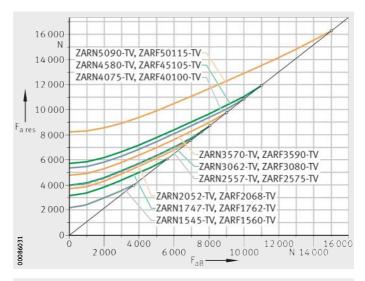
Equivalent dynamic bearing load

Life exponent p = 10/3

- cxponent p = 10/3

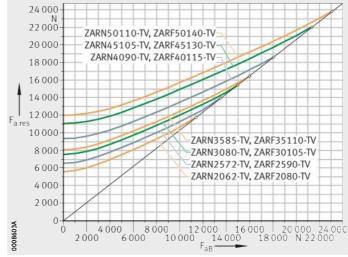
 $\begin{array}{ll} \text{Time proportion} \\ \text{n} & \text{min}^{-1} \end{array}$

Operating speed.



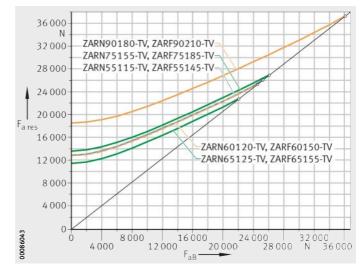
 $F_{aB} = operating\ load$ $F_{a\ res} = resultant\ bearing\ load$ $^{o} = limiting\ load$

Figure 4
Resultant bearing load
for ZARN, ZARF, light series



 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 5
Resultant bearing load for ZARN, ZARF, heavy series up to d = 50 mm



 F_{aB} = operating load $F_{a res}$ = resultant bearing load $^{\circ}$ = limiting load

Figure 6
Resultant bearing load for ZARN, ZARF, heavy series starting from d = 55 mm



Design of adjacent construction

The adjacent construction (the shaft and housing) must be designed in accordance with the data in the dimension tables.

The abutment diameters for the shaft and housing shoulders d_a and D_a must be in accordance with the dimension tables.



The permissible contact pressure for the adjacent construction must be observed.

The design of the adjacent construction must include the provision of lubricant feeds to page 110.

Sealing of the bearing position

The bearings are sealed, Figure 1, page 103:

- sealing in the direction of the threaded spindle by a rotary shaft seal on the outside surface machined free from spiral marks of the stepped shaft locating washer (ZARN..-L, ZARF..-L)
- sealing in the direction of the drive by the seal carrier assembly DRS.

Speeds

The limiting speeds n_G given in the dimension tables are based on the following conditions:

- bearing preloaded, no external operating load
- operating duration 25%
- maximum equilibrium temperature +50 °C.



The limiting speeds $\boldsymbol{n}_{\boldsymbol{G}}$ are valid for oil lubrication with adequate cooling.

Friction

In most applications, preloading of bearings by means of the locknut tightening torque gives sufficiently accurate setting values. The reference here is the tightening torque M_A in accordance with the dimension tables in conjunction with an INA precision locknut.

The frictional torque M_{RL} given in the dimension tables is a guide value. It is based on lightly oiled bearings, measured at a speed of $n = 5 \text{ min}^{-1}$.

For dimensioning of the drive, the starting frictional torque and the frictional torque at high speeds in each case of $2 \cdot M_{RL}$ to $3 \cdot M_{RL}$ must be taken into consideration.

Frictional torque and bearing preload

For applications in which the frictional torque is decisive for example temperature development, frictional torque compensation between different bearing positions, etc.), it is recommended that the bearing preload should be set in accordance with the bearing frictional torque M_{RI} .

Frictional energy

The frictional energy N_R of the bearings can be calculated as follows:

$$N_{R} = \frac{M_{RL} \cdot n}{9,55}$$

 $\begin{array}{lll} N_R & W \\ Frictional energy & \\ M_{RL} & Nm \\ Bearing frictional torque \\ n & min^{-1} \\ Operating speed. & \end{array}$

In any analysis of the thermal balance, the various operating speeds n_i must be taken into consideration with their time durations $\boldsymbol{q}_i.$

Lubrication

Needle roller/axial cylindrical roller bearings can be relubricated via the outer ring They are supplied with oil-based or dry preservative and should preferably be lubricated with oil.

Oils CLP to DIN 51517 and HLP to DIN 51524 of viscosity classes ISO VG 32 to ISO VG 100 have proved effective.

Relubrication for grease lubrication

Relubrication should be carried out with the bearing rotating and still warm from operation in order to achieve good replacement and distribution of grease.

The relubrication interval and quantity can only be determined under operating conditions since it is not possible to calculate all the influences in advance. For information on relubrication, see TPI 100, Bearings for Screw Drives, Mounting and Maintenance Manual.



Where a vertical axis of rotation is used in conjunction with automatic lubrication systems, the lubrication impulse should be selected such that the supply to the upper axial bearing is sufficient.



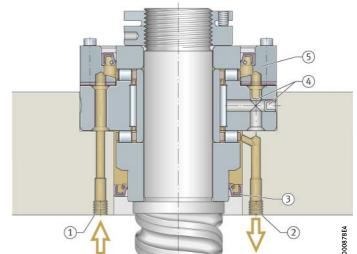


Lubricant feed

In the selection of lubricant feeds for ZARF(L), the mounting position must be observed, *Figure 7* and *Figure 8*.



Before commissioning, it must be ensured that all raceways are adequately supplied with lubricant.



ZARF..-L

① Oil inlet
② Oil outlet
③ Rotary shaft seal
④ Cover
⑤ Seal carrier assembly

Figure 7 Lubricant feed for vertical mounting position

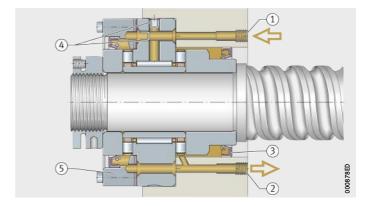
ZARF..-L

1) Oil inlet

② Oil outlet
③ Rotary shaft seal
④ Cover

Seal carrier assembly

Figure 8
Lubricant feed for horizontal mounting position



Mounting guidelines

Needle roller/axial cylindrical roller bearings ZARN and ZARF are not self-retaining. The individual bearing components are matched to each other. The individual parts of different bearings must not be interchanged during mounting.

The characteristics of the bearings are only valid when used in combination with INA precision locknuts and the associated tightening torques given in the dimension tables.



During mounting of bearings, mounting forces should be applied only to the bearing ring to be mounted. Mounting forces must never be directed through the rolling elements.

Bearings should only be mounted and dismounted in accordance with the Mounting and Maintenance Manual.

Further information

- TPI 100, Bearings for Screw Drives, Mounting and Maintenance Manual
- Download and ordering: http://medien.schaeffler.de.

Setting the axial preload

Since direct force measurement of this value during mounting operation is not cost-effective, the axial preload is set indirectly and using one of the following methods:

- by means of the tightening torque M_A of the precision locknut. The frictional torque may deviate from the value given in the dimension table
- by means of the bearing frictional torque M_{RI}.



In the axial component of needle roller/axial cylindrical roller bearings ZARF(L), the preload is a decisive factor for correct functioning. It must therefore be set with sufficient accuracy.





Preloading by locknut

Needle roller/axial cylindrical roller bearings must be axially preloaded during mounting by means of a precision locknut.

If the bearing unit is preloaded via the bearing inner rings using the recommended precision locknut, the tightening torques given in the dimension tables must be observed or the preload must be set using the bearing frictional torque given in the dimension tables. The tightening torques given for the individual bearing sizes are only valid for the precision locknuts listed.

For locating/non-locating bearing arrangements and bearing arrangements for rotating ball screw drive nuts, the tightening torques can be taken directly from the dimension tables.

In the case of needle roller/axial cylindrical roller bearings ZARN and ZARF, this tightening torque can be increased if a higher axial limiting load is required. The increase must take account of the increasing bearing friction, increase in bearing temperature and the decreased basic rating life. For information on limit values, please consult Schaeffler.

For locating/locating bearing arrangements, the locknut tightening torques must be calculated by means of BEARINX. In this case, please contact us or use BEARINX-online.

In order to counteract settling, it is recommended that the locknut should initially be tightened to twice the tightening torque M_A and then relieved of load again. It should only then be tightened again to the stated tightening torque $M_A.$ Finally, the precision locknut should be secured against rotation by the torque-controlled tightening of the grub screws.

Fixing screws

For location of the outer ring, screws of grade 10.9 should be used. Screws should be fully tightened with torque control in accordance with the manufacturer's instructions and in a crosswise sequence.

If the bearing outer ring is supported by an additional housing cover, it must be ensured that the fixing screws are sufficiently well dimensioned.

Accuracy

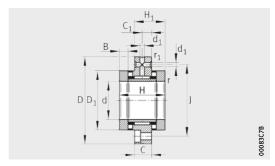
The accuracy of dimensions must be configured as follows:

- axial: tolerance class 4 to ISO 199
- radial: tolerance class 6 to ISO 492.





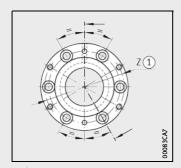
Light series For screw mounting



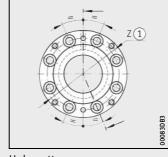
ZARF

Dimension table	· Dimens	ions	in mr	n															
Designation	Mass	Dim	ensio	ons															
	m	d	D	Н	H ₁	H ₂	H ₃	С	C_1	D_1	D_2	D ₃	В	B ₁	B ₂	r	r ₁	d ₁	J
	≈ kg															min.	min.		
ZARF1560-TV	0,42	15	60	40	26	_	_	14	8	35	_	_	7,5	_	_	0,3	0,6	3,2	46
ZARF1560-L-TV	0,45	15	60	_	-	53	39	14	8	35	24	34	7,5	20,5	11	0,3	0,6	3,2	46
ZARF1762-TV	0,49	17	62	43	27,5	_	_	14	8	38	_	_	9	_	_	0,3	0,6	3,2	48
ZARF1762-L-TV	0,52	17	62	-	-	57	41,5	14	8	38	28	38	9	23	11	0,3	0,6	3,2	48
ZARF2068-TV	0,56	20	68	46	29	-	_	14	8	42	-	-	10	-	-	0,3	0,6	3,2	53
ZARF2068-L-TV	0,61	20	68	-	-	60	43	14	8	42	30	40	10	24	12	0,3	0,6	3,2	53
ZARF2575-TV	0,78	25	75	50	33	-	_	18	10	47	-	-	10	-	-	0,3	0,6	3,2	58
ZARF2575-L-TV	0,84	25	75	-	-	65	48	18	10	47	36	45	10	25	12	0,3	0,6	3,2	58
ZARF3080-TV	0,85	30	80	50	33	-	-	18	10	52	-	-	10	_	-	0,3	0,6	3,2	63
ZARF3080-L-TV	0,9	30	80	-	-	65	48	18	10	52	40	50	10	25	13	0,3	0,6	3,2	63

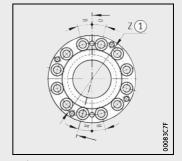
① Mounting dimension for seal carrier assembly DRS, see page 146; ③ Adjacent construction, bearing screw mounted in bore;



Hole pattern ZARF1560, ZARF1762



Hole pattern ZARF2068, ZARF2575



Hole pattern ZARF3080

Only valid in assembled condition

¹⁾ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

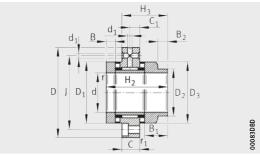
²⁾ Based on rotating inner ring.

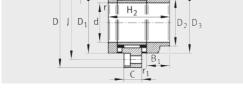
³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁴⁾ The required axial locknut force must be observed when using other locknuts.

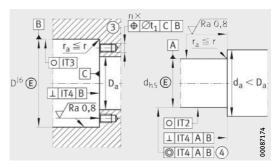
⁵⁾ The screws are not included in the scope of delivery. Tightening torque according to manufacturer's data.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





ZARF..-L



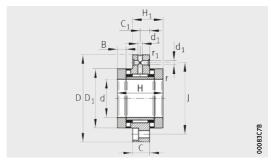
Design of housing and shaft (threaded spindle)

	Mount	ting isions	1)	Basic loa	nd ratings			Fatigue l	imit load	Limiting speeds		Bearing frictional	Rigidity	Tilting rigidity
				axial		radial		axial	radial			torque	axial	
	D _a	d _a	t ₁	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M_{RL}	c _{aL}	c _{kL}
	max.	min.		N	N	N	N	N	N	min ⁻¹	${\rm min}^{-1}$	Nm	N/μm	Nm/mrad
	36	28	0,1	24 900	53 000	13 000	17 500	7 300	2 600	8 5 0 0	2 200	0,35	1 400	110
	36	22	0,1	24 900	53 000	13 000	17 500	7 300	2 600	8 5 0 0	2 200	0,35	1 400	110
	39	28	0,1	26 000	57 000	14 000	19 900	7 900	2 9 5 0	7800	2100	0,4	1 600	160
	39	26	0,1	26 000	57 000	14 000	19900	7 900	2 9 5 0	7800	2100	0,4	1 600	160
	43	33	0,1	33 500	76 000	14900	22 400	6 300	3 3 5 0	7 000	2 000	0,5	1800	230
	43	28	0,1	33 500	76 000	14900	22 400	6 300	3 3 5 0	7 0 0 0	2000	0,5	1800	230
	48	39	0,1	35 500	86 000	22 600	36 000	7 100	5 100	6 0 0 0	1 900	0,55	1 900	350
	48	34	0,1	35 500	86 000	22 600	36 000	7 100	5 100	6 0 0 0	1 900	0,55	1 900	350
•	53	44	0,1	39 000	101 000	24 300	41 500	8 400	5 900	5 500	1 800	0,65	2 200	520
	53	38	0,1	39 000	101 000	24 300	41 500	8 400	5 900	5 500	1800	0,65	2 200	520

1 1	

Dimension table (cor	ntinued) · Dir	nensior	ıs in mm						
Designation	Mass moment of inertia ²⁾	Axial run- out ²⁾	Recommended locknut ³⁾	INA precisio	n	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760;	Fixing sc DIN EN IS 10.9	
			For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial	to be ordered separately	Size	n
	M _m				M _A				
	kg⋅cm ²	μm			Nm	N			
ZARF1560-TV	0,24	1	ZMA15/33	AM15	10	6 506	_	M6	6
ZARF1560-L-TV	0,274	1	ZMA15/33	AM15	10	6 506	24×35×7	M6	6
ZARF1762-TV	0,373	1	ZM17	AM17	12	7 078	-	M6	6
ZARF1762-L-TV	0,464	1	ZM17	AM17	12	7 078	28×40×7	M6	6
ZARF2068-TV	0,615	1	ZMA20/38	AM20	18	8 972	-	M6	8
ZARF2068-L-TV	0,683	1	ZMA20/38	AM20	18	8 972	30×42×7	M6	8
ZARF2575-TV	0,989	1	ZMA25/45	AM25	25	9 745	-	M6	8
ZARF2575-L-TV	1,15	1	ZMA25/45	AM25	25	9745	36×47×7	M6	8
ZARF3080-TV	1,46	1	ZMA30/52	AM30	32	10 662	_	M6	12
ZARF3080-L-TV	1,7	1	ZMA30/52	AM30	32	10 662	40×52×7	M6	12

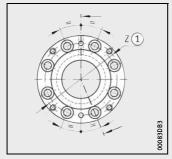
Light series For screw mounting



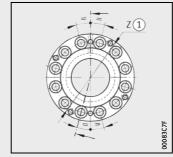
ZARF

Dimension table (ontinue	d) · D	imens	ions i	in mr	n													
Designation	Mass	Dim	iensioi	าร															
	m	d	D	Н	H ₁	H ₂	H ₃	С	C ₁	D ₁	D ₂	D ₃	В	B ₁	B ₂	r	r ₁	d ₁	J
	\approx kg															min.	min.		
ZARF3590-TV	1,12	35	90	54	35	_	_	18	10	60	_	-	11	_	1	0,3	0,6	3,2	73
ZARF3590-L-TV	1,25	35	90	-	-	70	51	18	10	60	45	58	11	27	13	0,3	0,6	3,2	73
ZARF40100-TV	1,35	40	100	54	35	-	-	18	10	65	-	-	11	-	-	0,3	0,6	3,2	80
ZARF40100-L-TV	1,45	40	100	-	-	70	51	18	10	65	50	63	11	27	14	0,3	0,6	3,2	80
ZARF45105-TV	1,7	45	105	60	40	-	-	22,5	12,5	70	-	-	11,5	-	-	0,3	0,6	6	85
ZARF45105-L-TV	1,85	45	105	-	-	75	55	22,5	12,5	70	56	68	11,5	26,5	13	0,3	0,6	6	85
ZARF50115-TV	2,1	50	115	60	40	-	-	22,5	12,5	78	-	-	11,5	_	-	0,3	0,6	6	94
ZARF50115-L-TV	2,45	50	115	-	-	78	58	22,5	12,5	78	60	78	11,5	29,5	14	0,3	0,6	6	94

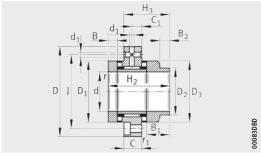
- ① Mounting dimension for seal carrier assembly DRS, see page 146; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition
- 1) If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.
- 2) Based on rotating inner ring.
- ³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.
- 4) The required axial locknut force must be observed when using other locknuts.
- 5) Tightening torque of fixing screws according to manufacturer's data. The screws are not included in the scope of delivery.
- 6) Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



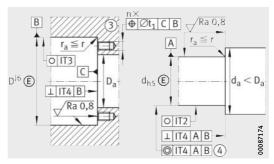
Hole pattern ZARF40100, ZARF45105



Hole pattern ZARF3590, ZARF50115







Design of housing and shaft (threaded spindle)

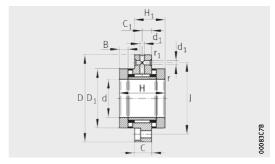
	Mount dimen		ı	Basic loa	nd ratings			Fatigue li	imit load	Limiting speeds	,	Bearing frictional	Rigid- ity	Tilting rigidity
				axial		radial		axial	radial			torque	axial	
•	D _a	d _a	t ₁	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M _{RL}	c _{aL}	c _{kL}
	max.	min.		N	N	N	N	N	N	min ⁻¹	min ⁻¹	Nm	N/µm	Nm/mrad
	61	50	0,1	56 000	148 000	26 000	47 000	12800	6 700	4 800	1700	0,9	2 600	740
	61	43	0,1	56 000	148 000	26 000	47 000	12800	6 700	4 800	1700	0,9	2 600	740
	66	55	0,2	59 000	163 000	27 500	53 000	14 100	7 500	4 400	1 600	1	2800	1 030
	66	48	0,2	59 000	163 000	27 500	53 000	14 100	7 500	4 400	1 600	1	2 800	1 030
	71	60	0,2	61 000	177 000	38 000	74 000	15 400	10 200	4 000	1 500	1,2	3 000	1 340
	71	54	0,2	61 000	177 000	38 000	74 000	15 400	10 200	4 000	1 500	1,2	3 000	1 340
	79	67	0,2	90 000	300 000	40 000	82 000	27 500	11 300	3 600	1 200	2,2	4 800	2 470
	79	58	0,2	90 000	300 000	40 000	82 000	27 500	11 300	3 600	1 200	2,2	4800	2 470

Dimension table (con	tinued) · Dir	nensio	ns in mm						
Designation	Mass mo- ment of inertia ²⁾	Axial run- out ²⁾	Recommended	INA precision l	ocknut ³⁾	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760;	Fixing sc DIN EN IS 10.9	
			For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial	to be ordered separately	Size	n
	M_{m}				M _A				
	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N			
ZARF3590-TV	2,8	1	ZMA35/58	AM35/58	40	12143	_	M6	12
ZARF3590-L-TV	3,21	1	ZMA35/58	AM35/58	40	12 143	45×60×8	M6	12
ZARF40100-TV	3,78	1	ZMA40/62	AM40	55	14 240	_	M8	8
ZARF40100-L-TV	4,35	1	ZMA40/62	AM40	55	14 240	50×65×8	M8	8
ZARF45105-TV	5,33	1	ZMA45/68	AM45	65	15 112	_	M8	8
ZARF45105-L-TV	6,03	1	ZMA45/68	AM45	65	15 112	56×70×8	M8	8
ZARF50115-TV	8,42	1	ZMA50/75	AM50	85	18 410	_	M8	12
ZARF50115-L-TV	10,46	1	ZMA50/75	AM50	85	18 410	60×80×8	M8	12





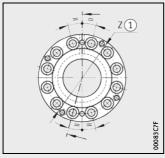
Heavy series For screw mounting



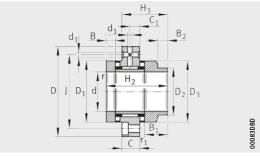
ZARF

$\textbf{Dimension table} \cdot \\$	Dimensi	ions i	n mm																
Designation	Mass	Dim	iensio	ns															
	m	d	D	Н	H ₁	H ₂	H ₃	С	C ₁	D ₁	D ₂	D ₃	В	B ₁	B ₂	r	r ₁	d ₁	J
	\approx kg															min.	min.		
ZARF2080-TV	1,1	20	80	60	38	-	_	18	10	52	-	-	12,5	-	-	0,3	0,6	3,2	63
ZARF2080-L-TV	1,22	20	80	_	_	75	53	18	10	52	40	50	12,5	27,5	13	0,3	0,6	3,2	63
ZARF2590-TV	1,6	25	90	60	38	_	_	18	10	62	-	-	12,5	_	_	0,3	0,6	3,2	73
ZARF2590-L-TV	1,75	25	90	-	-	75	53	18	10	62	48	60	12,5	27,5	13	0,3	0,6	3,2	73
ZARF30105-TV	1,95	30	105	66	41	-	-	18	10	68	-	-	14	-	-	0,3	0,6	3,2	85
ZARF30105-L-TV	2,15	30	105	-	-	82	57	18	10	68	52	66	14	30	13	0,3	0,6	3,2	85
ZARF35110-TV	1,6	35	110	66	41	-	-	18	10	73	-	-	14	-	-	0,3	0,6	3,2	88
ZARF35110-L-TV	1,85	35	110	-	-	82	57	18	10	73	60	73	14	30	13	0,3	0,6	3,2	88
ZARF40115-TV	2,7	40	115	75	47,5	_	_	22,5	12,5	78	-	-	16	_	-	0,3	0,6	6	94
ZARF40115-L-TV	3	40	115	-	-	93	65,5	22,5	12,5	78	60	78	16	34	14	0,3	0,6	6	94
ZARF45130-TV	3,9	45	130	82	51	-	-	22,5	12,5	90	-	-	17,5	-	-	0,3	0,6	6	105
ZARF45130-L-TV	4,3	45	130	-	-	103	72	22,5	12,5	90	70	88	17,5	38,5	18	0,3	0,6	6	105

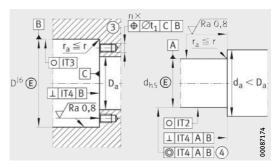
- ① Mounting dimension for seal carrier assembly DRS, see page 146; ③ Adjacent construction, bearing screw mounted in bore;
- (4) Only valid in assembled condition.
- 1) If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.
- 2) Based on rotating inner ring.
- 3) The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.
- $^{4)}$ The required axial locknut force must be observed when using other locknuts.
- 5) Tightening torque of fixing screws according to manufacturer's data. Screws are not included in the delivery.
- 6) Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



Hole pattern







Design of housing and shaft (threaded spindle)

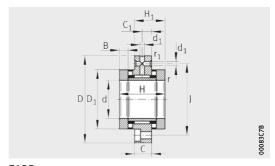
Mount dimen		1)	Basic load	d ratings			Fatigue li	imit load	Limiting speeds		Bearing frictional	Rigidity	Tilting rigidity
			axial		radial		axial	radial			torque	axial	
D _a	d _a	t ₁	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M_{RL}	c _{aL}	c _{kL}
max.	min.		N	N	N	N	N	N	${\rm min^{-1}}$	min ⁻¹	Nm	N/µm	Nm/mrad
53	38	0,1	64 000	141 000	22 600	36 000	12900	5 100	6 000	1 500	1,3	2 300	400
53	38	0,1	64 000	141 000	22 600	36 000	12900	5 100	6 000	1 500	1,3	2 300	400
63	45	0,1	80 000	199 000	24 300	41 500	18 300	5 900	4 900	1 400	1,6	3 000	800
63	45	0,1	80 000	199 000	24 300	41 500	18 300	5 900	4 900	1 400	1,6	3 000	800
69	52	0,2	107 000	265 000	26 000	47 000	22 500	6700	4 400	1 300	2,1	3 300	1 100
69	50	0,2	107 000	265 000	26 000	47 000	22 500	6700	4 400	1 300	2,1	3 300	1 100
74	60	0,2	105 000	265 000	27 500	53 000	22 500	7 500	4 000	1 250	2,3	3 500	1 300
74	58	0,2	105 000	265 000	27 500	53 000	22 500	7 500	4 000	1 250	2,3	3 500	1 300
79	65	0,2	117 000	315 000	38 000	74 000	26 500	10 200	3 700	1 200	2,5	3 800	1 800
79	58	0,2	117 000	315 000	38 000	74 000	26 500	10 200	3 700	1 200	2,5	3 800	1 800
91	70	0,2	154 000	405 000	40 000	82 000	36 500	11 300	3 300	1150	3,5	4 000	2 100
91	68	0,2	154 000	405 000	40 000	82 000	36 500	11 300	3 300	1 150	3,5	4 000	2 100

		F
		L

Dimension table (con	tinued) · Dir	nension	ns in mm						
Designation	Mass mo- ment of inertia ²⁾	Axial run- out ²⁾	Recommended I locknut ³⁾	NA precisior	1	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760;	Fixing sci DIN EN IS 10.9	
			For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial	to be ordered separately	Size	n
	M _m				M _A				
	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N			
ZARF2080-TV	1,98	1	ZMA20/52	AM20	38	17 623	_	M6	12
ZARF2080-L-TV	2,27	1	ZMA20/52	AM20	38	17 623	40×52×7	M6	12
ZARF2590-TV	3,88	1	ZMA25/58	AM25	55	20 790	-	M6	12
ZARF2590-L-TV	4,51	1	ZMA25/58	AM25	55	20 790	48×62×8	M6	12
ZARF30105-TV	6,53	1	ZMA30/65	AM30	75	24 287	-	M8	12
ZARF30105-L-TV	7,43	1	ZMA30/65	AM30	75	24 287	52×68×8	M8	12
ZARF35110-TV	8,47	1	ZMA35/70	AM35	100	27 480	-	M8	12
ZARF35110-L-TV	10,4	1	ZMA35/70	AM35	100	27 480	60×75×8	M8	12
ZARF40115-TV	13,3	1	ZMA40/75	AM40	120	29834	-	M8	12
ZARF40115-L-TV	15,5	1	ZMA40/75	AM40	120	29 834	60×80×8	M8	12
ZARF45130-TV	23,7	1	ZMA45/85	AM45	150	33 549	-	M8	12
ZARF45130-L-TV	28,1	1	ZMA45/85	AM45	150	33 549	70×90×10	M8	12



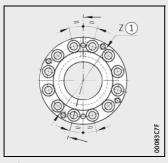
Heavy series For screw mounting



ZARF

Dimension table (c	ontinue	ed) • I	Dimen	sions	s in m	ım													
Designation	Mass	Dim	nensio	ns															
	m ≈ kg	d	D	Н	H ₁	H ₂	H ₃	С	C ₁	D ₁	D ₂	D ₃	В	B ₁	B ₂	r min.	r ₁	d ₁	J
ZARF50140-TV	4,2	50	140	82	51	_	_	22,5	12,5	95	_	-	17,5	-	_	0,3	0,6	6	113
ZARF50140-L-TV	4,65	50	140	-	-	103	72	22,5	12,5	95	75	93	17,5	38,5	18	0,3	0,6	6	113
ZARF55145-TV	4,5	55	145	82	51	_	-	22,5	12,5	100	-	-	17,5	-	-	0,3	0,6	6	118
ZARF55145-L-TV	5	55	145	-	-	103	72	22,5	12,5	100	80	98	17,5	38,5	18	0,3	0,6	6	118
ZARF60150-TV	4,7	60	150	82	51	_	_	22,5	12,5	105	-	-	17,5	-	-	0,3	0,6	6	123
ZARF60150-L-TV	5,35	60	150	-	-	103	72	22,5	12,5	105	90	105	17,5	38,5	18	0,3	0,6	6	123
ZARF65155-TV	5,1	65	155	82	51	_	-	22,5	12,5	110	-	-	17,5	-	-	0,3	0,6	6	128
ZARF65155-L-TV	5,7	65	155	-	-	103	72	22,5	12,5	110	90	108	17,5	38,5	18	0,3	0,6	6	128

① Mounting dimension for seal carrier assembly DRS, see page 146; ③ Adjacent construction, bearing screw mounted in bore; ④ Only valid in assembled condition.



Hole pattern ZARF5, ZARF6

¹⁾ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

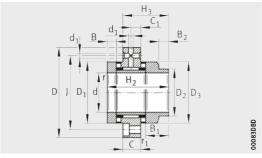
²⁾ Based on rotating inner ring.

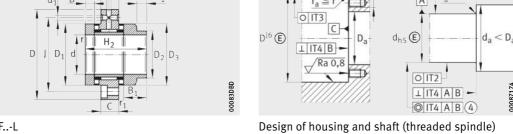
³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

⁴⁾ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Tightening torque of fixing screws according to manufacturer's data. The screws are not included in the scope of delivery.

⁶⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





ZARF..-L

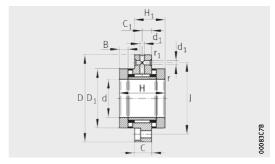
	Mount dimen	ing sions ¹	.)	Basic load	l ratings			Fatigue li	mit load	Limiting speeds	o .	Bearing frictional	Rigidity	Tilting rigidity
				axial		radial		axial	radial			torque	axial	
Ī	D _a	d _a	t ₁	dyn. C _a	stat. C _{0a}	dyn. stat. C _{or}		C _{ua}	C _{ur}	n _G oil	n _G grease	M _{RL}	c _{aL}	c _{kL}
I	max.	min.		N	N	N	N	N	N	min ⁻¹	min ⁻¹	Nm	$N/\mu m$	Nm/mrad
	96	75	0,2	172 000	480 000	42 000	90 000	43 500	12 400	3 100	1 100	3,8	4 600	2 900
	96	73	0,2	172 000	480 000	42 000	90 000	43 500	12 400	3 100	1 100	3,8	4 600	2 900
	101	85	0,2	177 000	500 000	44 000	98 000	46 000	13 500	2 900	1 000	4	4 900	3 600
;	101	78	0,2	177 000	500 000	44 000	98 000	46 000	13 500	2 900	1 000	4	4 900	3 600
	106	90	0,2	187 000	550 000	44 500	92 000	50 000	12 600	2 700	950	4,2	5 300	4 300
:	106	88	0,2	187 000	550 000	44 500	92 000	50 000	12 600	2 700	950	4,2	5 300	4 300
	111	97	0,2	172 000	500 000	54000	104 000	46 000	14 600	2 600	900	4	4 800	4 000
	111	88	0,2	172 000	500 000	54000	104 000	46 000	14 600	2 600	900	4	4 800	4 000

Dimension table (cor	ıtinued) · Di	mensio	ns in mm						
Designation	Mass mo- ment of inertia ²⁾	Axial run- out ²⁾	Recommended II locknut ³⁾	NA precisio	n	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760;	Fixing sc DIN EN IS 10.9	
			For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial	to be ordered separately	Size	n
	M_{m}				M _A				
	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N			
ZARF50140-TV	29,8	1	ZMA50/92	AM50	180	36 224	_	M10	12
ZARF50140-L-TV	35,3	1	ZMA50/92	AM50	180	36 224	75×95×10	M10	12
ZARF55145-TV	36,1	1	ZMA55/98	AM55	220	39 807	_	M10	12
ZARF55145-L-TV	43	1	ZMA55/98	AM55	220	39 807	80×100×10	M10	12
ZARF60150-TV	43,8	1	ZMA60/98	AM60	250	41 144	_	M10	12
ZARF60150-L-TV	54,5	1	ZMA60/98 AM60 250		250	41 144	90×110×12	M10	12
ZARF65155-TV	51	1	ZMA65/105	AM65	270	40 652	_	M10	12
ZARF65155-L-TV	60,1	1	ZMA65/105	AM65	270	40 652	90×110×12	M10	12





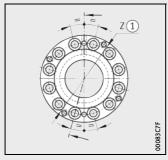
Heavy series For screw mounting



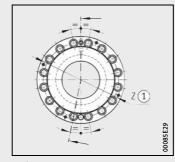
ZARF

Dimension table (continue	d) · l	Dimer	sions	in mn	1													
Designation	Mass	Dim	nensio	ns															
	m	d	D	Н	H ₁	H ₂	H ₃	С	C ₁	D ₁	D_2	D ₃	В	B ₁	B ₂	r	r ₁	d_1	<u> </u>
		ŭ			1	12	3		C 1		D2	23		<i>D</i> ₁	52		'1	u ₁	,
	≈ kg															min.	min.		
ZARF70160-TV	5,2	70	160	82	51	_	-	22,5	12,5	115	-	-	17,5	-	1	0,3	0,6	6	133
ZARF70160-L-TV	5,95	70	160	-	-	103	72	22,5	12,5	115	100	115	17,5	38,5	18	0,3	0,6	6	133
ZARF75185-TV	9,4	75	185	100	62	-	_	27	15	135	-	-	21	-	-	0,3	1	6	155
ZARF75185-L-TV	10,6	75	185	-	-	125	87	27	15	135	115	135	21	46	20	0,3	1	6	155
ZARF90210-TV	13,7	90	210	110	69,5	-	-	32	17,5	160	-	-	22,5	-	-	0,3	1	8	180
ZARF90210-L-TV	15,1	90	210	-	-	135	94,5	32	17,5	160	130	158	22,5	47,5	18	0,3	1	8	180

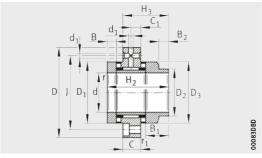
- ① Mounting dimension for seal carrier assembly DRS, see page 146; ③ Adjacent construction, bearing screw mounted in bore;
- 4 Only valid in assembled condition.
- $^{1)}$ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.
- 2) Based on rotating inner ring.
- 3) The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.
- ⁴⁾ The required axial locknut force must be observed when using other locknuts.
- 5) Tightening torque of fixing screws according to manufacturer's data. The screws are not included in the scope of delivery.
- 6) Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.

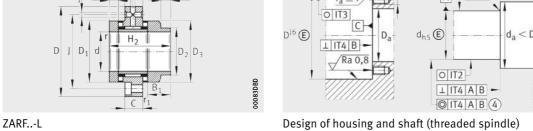


Hole pattern ZARF7



Hole pattern ZARF90210





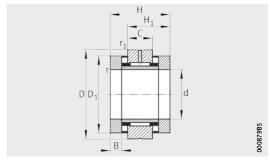
	Mount			Basic load	d ratings			Fatigue li	mit load	Limiting speeds	,	Bearing frictional	Rigidity	Tilting rigidity
				axial		radial		axial	radial			torque	axial	
	D _a d _a t ₁		t ₁	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M_{RL}	c _{aL}	c _{kL}
	max.	min.		N	N	N	N	N	N	min ⁻¹	${\rm min^{-1}}$	Nm	$\text{N}/\mu\text{m}$	Nm/mrad
	116	100	0,2	201 000	630 000	56 000	119 000	57 000	15 800	2 400	800	4,8	5 800	6 000
	116	98	0,2	201 000	630 000	56 000	119 000	57 000	15 800	2 400	800	4,8	5 800	6 000
	136	113	0,4	290 000	890 000	72 000	132 000	84 000	18 800	2 100	700	8	6 600	8 500
	136	110	0,4	290 000	890 000	72 000	132 000	84 000	18 800	2 100	700	8	6 600	8 500
·	161	130	0,4	325 000	1 030 000	98 000	210 000	93 000	28 500	1 800	700	10,5	7 700	14 500
	161	125	0,4	325 000	1 030 000	98 000	210 000	93 000	28 500	1 800	700	10,5	7 700	14 500

			ns in mm						
Dimension table (co	ontinued) · [Dimens	ions in mm						
Designation	Mass mo- ment of inertia ²⁾	Axial run- out ²⁾	Recommended I locknut ³⁾	NA precisio	on	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered	Fixing sc DIN EN IS 10.9	
			For radial locking	For axial locking	Tight- ening torque ⁶⁾	axial	separately	Size	n
	M _m		M _A						
	$\text{kg}\cdot\text{cm}^2$	μm		Nm		N			
ZARF70160-TV	62,2	1	ZMA70/110	AM70	330	46 786	_	M10	12
ZARF70160-L-TV	77,3	1	ZMA70/110	AM70	330	46 786	100×120×12	M10	12
ZARF75185-TV	149	2	ZMA75/125	AM75	580	72 971	_	M12	12
ZARF75185-L-TV	188	2	ZMA75/125	AM75 580		72 971	115×140×12	M12	12
ZARF90210-TV	312	2	ZMA90/155	55 AM90 960		100 669	_	M12	16
ZARF90210-L-TV	372	2	ZMA90/155	AM90	960	100 669	130×160×12	M12	16





Light series Not for screw mounting



ZARN

	Rimanalan takla. Dimanalang in mm															
$\textbf{Dimension table} \cdot Di$	imension	s in m	m													
Designation	Mass	Dime	ension	S												
	l _m	d	D	Н	H ₁	H ₂	H ₃	С	D_1	D_2	D_3	В	B ₁	B ₂	r	r ₁
	'''	l u			1''1	112	113			D2	D3		D ₁	52		'1
	≈ kg														min.	min.
ZARN1545-TV	0,34	15	45	40	28	-	-	16	35	_	_	7,5	-	-	0,3	0,6
ZARN1545-L-TV	0,37	15	45	_	-	53	41	16	35	24	34	7,5	20,5	11	0,3	0,6
ZARN1747-TV	0,37	17	47	43	29,5	-	-	16	38	_	_	9	-	-	0,3	0,6
ZARN1747-L-TV	0,41	17	47	-	-	57	43,5	16	38	28	38	9	23	11	0,3	0,6
ZARN2052-TV	0,41	20	52	46	31	-	-	16	42	_	_	10	-	-	0,3	0,6
ZARN2052-L-TV	0,46	20	52	-	-	60	45	16	42	30	40	10	24	12	0,3	0,6
ZARN2557-TV	0,53	25	57	50	35	-	-	20	47	_	_	10	-	-	0,3	0,6
ZARN2557-L-TV	0,59	25	57	-	-	65	50	20	47	36	45	10	25	12	0,3	0,6
ZARN3062-TV	0,6	30	62	50	35	-	-	20	52	-	-	10	-	-	0,3	0,6
ZARN3062-L-TV	0,75	30	62	-	-	65	50	20	52	40	50	10	25	13	0,3	0,6

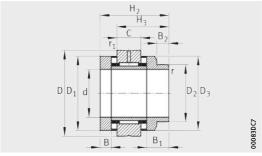
¹⁾ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

²⁾ Based on rotating inner ring.

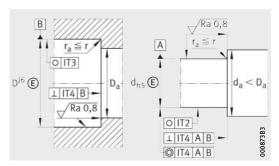
³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{4)}}$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.







Design of housing and shaft (threaded spindle)

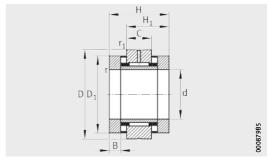
Mounti dimens		Basic load	l ratings			Fatigue lii	mit load	Limiting speeds		Bearing frictional	Rigidity
		axial		radial		axial	radial			torque	axial
D _a	d _a	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{Or}	C _{ua}	C _{ur}	n _G oil	n _G grease	M _{RL}	c _{aL}
max.	min.	N	N	N	N	N	N	${\rm min^{-1}}$	min ⁻¹	Nm	N/µm
36	28	24 900	53 000	13 000	17 500	7 300	2600	8 500	2 200	0,35	1 400
36	22	24 900	53 000	13 000	17 500	7 300	2600	8 500	2 200	0,35	1 400
39	28	26 000	57 000	14000	19 900	7 900	2950	7 800	2 100	0,4	1 600
39	26	26 000	57 000	14000	19 900	7 900	2950	7 800	2 100	0,4	1 600
43	33	33 500	76 000	14900	22 400	6 300	3 3 5 0	7 000	2 000	0,5	1 800
43	28	33 500	76 000	14900	22 400	6 300	3 3 5 0	7 000	2 000	0,5	1 800
48	39	35 500	86 000	22600	36 000	7 100	5 100	6 000	1 900	0,55	1 900
48	34	35 500	86 000	22600	36 000	7 100	5 100	6 000	1 900	0,55	1 900
53	44	39 000	101 000	24300	41 500	8 400	5 900	5 500	1 800	0,65	2 200
53	38	39 000	101 000	24300	41 500	8 400	5 900	5 500	1 800	0,65	2 200

Ħ	
T;	

Dimension table (conf	tinued) · Dim	iensions in m	ım					
Designation	Tilting rigidity	Mass moment of inertia ²⁾	Axial run- out ²⁾	Recommended IN	NA precision l	ocknut ³⁾	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered
				For radial locking	For axial locking	Tight- ening torque ⁵⁾	axial	separately
	c _{kL}	M _m				M _A		
	Nm/mrad	kg⋅cm²	μm			Nm	N	
ZARN1545-TV	110	0,24	1	ZMA15/33	AM15	10	6 5 0 6	_
ZARN1545-L-TV	110	0,274	1	ZMA15/33	AM15	10	6 5 0 6	24×35×7
ZARN1747-TV	160	0,373	1	ZM17	AM17	12	7 0 7 8	-
ZARN1747-L-TV	160	0,464	1	ZM17	AM17	12	7 0 7 8	28×40×7
ZARN2052-TV	230	0,615	1	ZMA20/38	AM20	18	8 9 7 2	_
ZARN2052-L-TV	230	0,683	1	ZMA20/38	AM20	18	8 9 7 2	30×42×7
ZARN2557-TV	350	0,989	1	ZMA25/45	AM25	25	9745	-
ZARN2557-L-TV	350	1,15	1	ZMA25/45	AM25	25	9745	36×47×7
ZARN3062-TV	520	1,46	1	ZMA30/52	AM30	32	10 662	-
ZARN3062-L-TV	520	1,7	1	ZMA30/52	AM30	32	10 662	40×52×7



Light series Not for screw mounting



ZARN

Dimension table (cor	ntinued) ·	Dime	nsions	in mr	n											
Designation	Mass	Dime	ension	S												
	m	d	D	Н	H ₁	H ₂	H ₃	С	D ₁	D ₂	D ₃	В	B ₁	B ₂	r	r ₁
	$\approx kg$														min.	min.
ZARN3570-TV	0,8	35	70	54	37	-	-	20	60	-	-	11	-	-	0,3	0,6
ZARN3570-L-TV	0,93	35	70	-	_	70	53	20	60	45	58	11	27	13	0,3	0,6
ZARN4075-TV	0,9	40	75	54	37	-	-	20	65	-	-	11	-	-	0,3	0,6
ZARN4075-L-TV	1	40	75	-	-	70	53	20	65	50	63	11	27	14	0,3	0,6
ZARN4580-TV	1,12	45	80	60	42,5	-	-	25	70	-	-	11,5	-	-	0,3	0,6
ZARN4580-L-TV	1,27	45	80	-	-	75	57,5	25	70	56	68	11,5	26,5	13	0,3	0,6
ZARN5090-TV	1,43	50	90	60	42,5	-	-	25	78	-	-	11,5	-	-	0,3	0,6
ZARN5090-L-TV	1,78	50	90	_	-	78	60,5	25	78	60	78	11,5	29,5	14	0,3	0,6

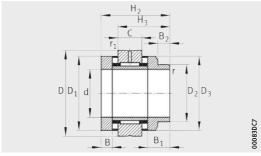
 $^{^{1)}}$ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

²⁾ Based on rotating inner ring.

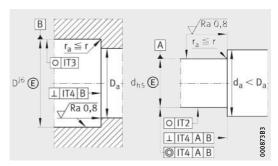
³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{4)}}$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.







Design of housing and shaft (threaded spindle)

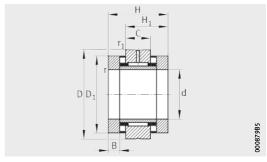
	Mount dimen	ing sions ¹⁾	Basic load	ratings			Fatigue lim	it load	Limiting speeds		Bearing frictional	Rigidity
			axial		radial		axial	radial			torque	axial
·-	D _a	d _a	C _a C _{0a}		dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M_{RL}	c _{aL}
	max.	min.	N	N	N	N	N	N	\min^{-1}	${\rm min^{-1}}$	Nm	$N/\mu m$
	61	50	56000	148 000	26 000	47 000	12800	6 700	4 800	1700	0,9	2 600
	61	43	56000	148 000	26 000	47 000	12800	6700	4 800	1700	0,9	2 600
	66	55	59 000	163 000	27 500	53 000	14 100	7 500	4 400	1600	1	2 800
	66	48	59000	163 000	27 500	53 000	14 100	7 500	4 400	1600	1	2 800
	71	60	61 000	177 000	38 000	74 000	15 400	10 200	4 000	1500	1,2	3 000
	71	54	61 000	177 000	38 000	74 000	15 400	10 200	4 000	1 500	1,2	3 000
	79	67	90 000	300 000	40 000	82 000	27 500	11 300	3 600	1 200	2,2	4 800
	79	58	90 000	300 000	40 000	82 000	27 500	11 300	3 600	1 200	2,2	4 800

Dimension table (cor	ntinued) · Dir	nensions in n	nm					
Designation	Tilting rigidity	Mass moment of inertia ²⁾	Axial run- out ²⁾	Recommended II	NA precision loo	cknut ³⁾	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered
				For radial locking	For axial locking	Tight- ening torque ⁵⁾	axial	separately
	c _{kL}	M _m				M _A		
	Nm/mrad	kg⋅cm²	μm			Nm	N	
ZARN3570-TV	740	2,8	1	ZMA35/58	AM35/58	40	12 143	-
ZARN3570-L-TV	740	3,21	1	ZMA35/58	AM35/58	40	12 143	45×60×8
ZARN4075-TV	1 0 3 0	3,78	1	ZMA40/62	AM40	55	14 240	_
ZARN4075-L-TV	1 0 3 0	4,35	1	ZMA40/62	AM40	55	14 240	50×65×8
ZARN4580-TV	1 340	5,33	1	ZMA45/68	AM45	65	15 112	-
ZARN4580-L-TV	1340	6,03	1	ZMA45/68	AM45	65	15 112	56×70×8
ZARN5090-TV	2 4 7 0	8,42	1	ZMA50/75	AM50	85	18 410	_
ZARN5090-L-TV	2 470	10,46	1	ZMA50/75	AM50	85	18 410	60×80×8





Heavy series Not for screw mounting



ZARN

Dimension table ⋅ Dimensions in mm																
Designation	Mass	Dime	nensions													
	m	d	D	Н	H ₁	H ₂	H ₃	С	D_1	D_2	D_3	В	B ₁	B ₂	r	r ₁
	\approx kg														min.	min.
ZARN2062-TV	0,87	20	62	60	40	_	_	20	52	-	-	12,5	-	-	0,3	0,6
ZARN2062-L-TV	0,99	20	62	-	-	75	55	20	52	40	50	12,5	27,5	13	0,3	0,6
ZARN2572-TV	1,17	25	72	60	40	-	_	20	62	-	-	12,5	-	-	0,3	0,6
ZARN2572-L-TV	1,32	25	72	-	-	75	55	20	62	48	60	12,5	27,5	13	0,3	0,6
ZARN3080-TV	1,5	30	80	66	43	-	_	20	68	-	-	14	-	-	0,3	0,6
ZARN3080-L-TV	1,7	30	80	-	-	82	59	20	68	52	66	14	30	13	0,3	0,6
ZARN3585-TV	1,65	35	85	66	43	_	_	20	73	_	-	14	-	-	0,3	0,6
ZARN3585-L-TV	1,8	35	85	-	_	82	59	20	73	60	73	14	30	13	0,3	0,6

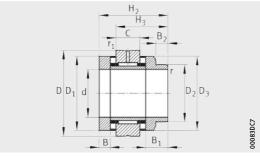
 $^{^{1)}}$ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

²⁾ Based on rotating inner ring.

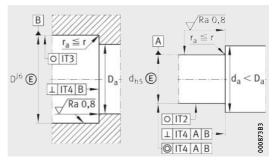
³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{4)}}$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.







Design of housing and shaft (threaded spindle)

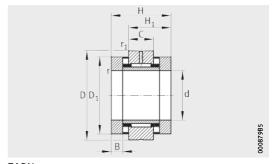
	Mountir dimensi		Basic load	ratings			Fatigue lim	it load	Limiting speeds		Bearing frictional	Rigidity
			axial		radial		axial	radial			torque	axial
-	D _a	d _a	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M _{RL}	c _{aL}
	max.	min.	N	N	N	N	N	N	${\rm min}^{-1}$	min ⁻¹	Nm	$N/\mu m$
	53	38	64 000	141 000	22 600	36 000	12 900	5 100	6 000	1 500	1,3	2 300
	53	38	64 000	141 000	22 600	36 000	12 900	5 100	6 000	1 500	1,3	2 300
	63	45	80 000	199 000	24 300	41 500	18 300	5 900	4 900	1 400	1,6	3 000
	63	45	80 000	199 000	24 300	41 500	18 300	5 900	4 900	1 400	1,6	3 000
	69	52	107 000	265 000	26 000	47 000	22 500	6 700	4 400	1 300	2,1	3 300
	69	50	107 000	265 000	26 000	47 000	22 500	6 700	4 400	1 300	2,1	3 300
	74	60	105 000	265 000	27 500	53 000	22 500	7 500	4 000	1 250	2,3	3 500
	74	58	105 000	265 000	27 500	53 000	22 500	7 500	4 000	1 250	2,3	3 500

Dimension table (cor	Dimension table (continued) · Dimensions in mm									
Designation	Tilting rigidity	Mass moment of inertia ²⁾	Axial run- out ²⁾	Recommended I	NA precision	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered			
				For radial locking	For axial locking	Tight- ening torque ⁵⁾	axial	separately		
	c _{kL}	M _m				M _A				
	Nm/mrad	kg⋅cm²	μm			Nm	N			
ZARN2062-TV	400	1,98	1	ZMA20/52	AM20	38	17 623	_		
ZARN2062-L-TV	400	2,27	1	ZMA20/52	AM20	38	17 623	40×52×7		
ZARN2572-TV	800	3,88	1	ZMA25/58	AM25	55	20 790	_		
ZARN2572-L-TV	800	4,51	1	ZMA25/58	AM25	55	20 790	48×62×8		
ZARN3080-TV	1100	6,53	1	ZMA30/65	AM30	75	24 287	-		
ZARN3080-L-TV	1100	7,43	1	ZMA30/65	AM30	75	24 287	52×68×8		
ZARN3585-TV	1 300	8,47	1	ZMA35/70	AM35	100	27 480	-		
ZARN3585-L-TV	1 300	10,4	1	ZMA35/70	AM35	100	27 480	60×75×8		





Heavy series Not for screw mounting



ZARN

Dimension table (continued) · Dimensions in mm																
Designation	Mass	Dime	mensions													
	m	d	D	Н	H ₁	H ₂	H ₃	С	D_1	D ₂	D ₃	В	B ₁	B ₂	r	r ₁
	≈ kg														min.	min.
ZARN4090-TV	2,09	40	90	75	50	-	-	25	78	-	-	16	-	-	0,3	0,6
ZARN4090-L-TV	2,39	40	90	-	-	93	68	25	78	60	78	16	34	14	0,3	0,6
ZARN45105-TV	3,02	45	105	82	53,5	-	-	25	90	-	_	17,5	-	_	0,3	0,6
ZARN45105-L-TV	3,42	45	105	-	-	103	74,5	25	90	70	88	17,5	38,5	18	0,3	0,6
ZARN50110-TV	3,3	50	110	82	53,5	-	-	25	95	-	_	17,5	-	_	0,3	0,6
ZARN50110-L-TV	3,75	50	110	-	-	103	74,5	25	95	75	93	17,5	38,5	18	0,3	0,6
ZARN55115-TV	3,5	55	115	82	53,5	-	-	25	100	-	-	17,5	-	-	0,3	0,6
ZARN55115-L-TV	4	55	115	-	-	103	74,5	25	100	80	98	17,5	38,5	18	0,3	0,6

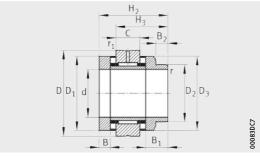
 $^{^{1)}}$ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

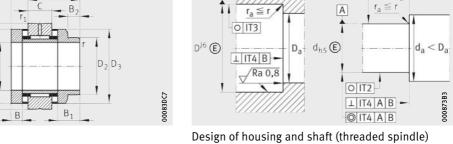
²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{4)}}$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.





ZARN..-L

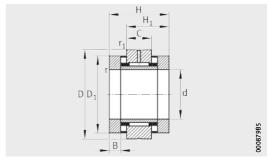
	Mounting dimension		Basic load	ratings			Fatigue lin	nit load	Limiting speeds		Bearing frictional	Rigidity
			axial		radial		axial	radial			torque	axial
-	D _a	d _a	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M_{RL}	c _{aL}
	max.	min.	N	N	N	N	N	N	min ^{−1}	${\rm min^{-1}}$	Nm	$N/\mu m$
	79	65	117 000	315 000	38 000	74 000	26 500	10 200	3 700	1 200	2,5	3 800
	79	58	117 000	315 000	38 000	74 000	26 500	10 200	3 700	1 200	2,5	3 800
	91	70	154 000	405 000	40 000	82 000	36 500	11 300	3 300	1 150	3,5	4000
	91	68	154 000	405 000	40 000	82 000	36 500	11 300	3 300	1 150	3,5	4000
	96	75	172 000	480 000	42 000	90 000	43 500	12 400	3 100	1 100	3,8	4600
	96	73	172 000	480 000	42 000	90 000	43 500	12 400	3 100	1 100	3,8	4 600
	101	85	177 000	500 000	44 000	98 000	46 000	13 500	2 900	1 000	4	4900
	101	78	177 000	500 000	44 000	98 000	46 000	13 500	2 900	1 000	4	4900

Dimension table (cont	Dimension table (continued) · Dimensions in mm										
Designation	Tilting rigidity	Mass moment of inertia ²⁾	Axial run- out ²⁾	Recommended I	NA precision	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered				
				For radial locking	For axial locking	Tight- ening torque ⁵⁾	axial	separately			
	c _{kL}	M _m				M _A					
	Nm/mrad	kg⋅cm²	μm			Nm	N				
ZARN4090-TV	1800	13,3	1	ZMA40/75	AM40	120	29834	_			
ZARN4090-L-TV	1800	15,5	1	ZMA40/75	AM40	120	29834	60×80×8			
ZARN45105-TV	2100	23,7	1	ZMA45/85	AM45	150	33 549	_			
ZARN45105-L-TV	2100	28,1	1	ZMA45/85	AM45	150	33 549	70×90×10			
ZARN50110-TV	2900	29,8	1	ZMA50/92	AM50	180	36 224	-			
ZARN50110-L-TV	2900	35,3	1	ZMA50/92	AM50	180	36 224	75×95×10			
ZARN55115-TV	3 600	36,1	1	ZMA55/98	AM55	220	39807	-			
ZARN55115-L-TV	3 600	43	1	ZMA55/98	AM55	220	39807	80×100×10			





Heavy series Not for screw mounting



ZARN

Dimension table (c	ontinuec	d) · Dimensions in mm														
Designation	Mass	Dime	nensions													
	m	d	D	Н	H ₁	H ₂	H ₃	С	D_1	D_2	D_3	В	B ₁	B_2	r	r ₁
	≈ kg														min.	min.
ZARN60120-TV	3,7	60	120	82	53,5	-	-	25	105	-	-	17,5	-	-	0,3	0,6
ZARN60120-L-TV	4,85	60	120	-	_	103	74,5	25	105	90	105	17,5	38,5	18	0,3	0,6
ZARN65125-TV	4	65	125	82	53,5	-	-	25	110	-	-	17,5	-	-	0,3	0,6
ZARN65125-L-TV	4,6	65	125	-	_	103	74,5	25	110	90	108	17,5	38,5	18	0,3	0,6
ZARN70130-TV	4,1	70	130	82	53,5	-	-	25	115	_	-	17,5	_	_	0,3	0,6
ZARN70130-L-TV	4,85	70	130	-	_	103	74,5	25	115	100	115	17,5	38,5	18	0,3	0,6
ZARN75155-TV	7,9	75	155	100	65	-	-	30	135	_	-	21	_	-	0,3	1
ZARN75155-L-TV	9,1	75	155	-	_	125	90	30	135	115	135	21	46	20	0,3	1
ZARN90180-TV	11,8	90	180	110	72,5	-	-	35	160	_	-	22,5	-	-	0,3	1
ZARN90180-L-TV	13,2	90	180	-	_	135	97,5	35	160	130	158	22,5	47,5	18	0,3	1

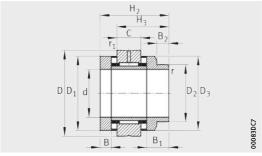
 $^{^{1)}}$ If rotary shaft seals are used, the outside diameter of the sealing ring must be taken into consideration.

²⁾ Based on rotating inner ring.

³⁾ The recommended INA precision locknuts are not included in the scope of delivery and must be ordered separately.

 $^{^{4)}}$ The required axial locknut force must be observed when using other locknuts.

⁵⁾ Only valid in conjunction with INA precision locknuts. Note the guidelines on locating/locating bearing arrangements, see page 54.



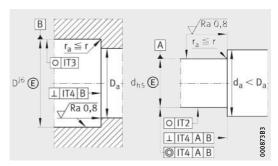
Dimension table (continued) · Dimensions in mm

8 500

14 500

14 500





Design of housing and shaft (threaded spindle)

Mountii dimens		Basic load	ratings			Fatigue li	mit load	Limiting speeds		Bearing frictional	Rigidity
		axial		radial		axial	radial			torque	axial
D _a	d _a	dyn. C _a	stat. C _{0a}	dyn. C _r	stat. C _{0r}	C _{ua}	C _{ur}	n _G oil	n _G grease	M _{RL}	c _{aL}
max.	min.	N	N	N	N	N	N	min ⁻¹	min ⁻¹	Nm	N/µm
106	90	187 000	550 000	44 500	92 000	50 000	12600	2 700	950	4,2	5 300
106	88	187 000	550 000	44 500	92 000	50 000	12600	2 700	950	4,2	5 300
111	97	172 000	500 000	54 000	104 000	46 000	14600	2 600	900	4	4 800
111	88	172 000	500 000	54 000	104 000	46 000	14600	2 600	900	4	4 800
116	100	201 000	630 000	56 000	119 000	57 000	15800	2 400	800	4,8	5 800
116	98	201 000	630 000	56 000	119 000	57 000	15800	2 400	800	4,8	5 800
136	113	290 000	890 000	72 000	132 000	84 000	18800	2 100	700	8	6 600
136	110	290 000	890 000	72 000	132 000	84 000	18800	2 100	700	8	6 600
161	130	325 000	1 030 000	98 000	210 000	93 000	28 500	1 800	700	10,5	7 700
161	125	325 000	1 030 000	98 000	210 000	93 000	28 500	1 800	700	10,5	7 700

Designation	Tilting rigidity	Mass mo- ment of inertia ²⁾	Axial run- out ²⁾	Recommended IN	A precision	Required locknut force ⁴⁾	Rotary shaft seal to DIN 3760; to be ordered	
				For radial locking	For axial locking	Tight- ening torque ⁵⁾	axial	separately
	c _{kL}	M _m				M _A		
	Nm/mrad	$\text{kg}\cdot\text{cm}^2$	μm			Nm	N	
ZARN60120-TV	4 300	43,8	1	ZMA60/98	AM60	250	41 144	_
ZARN60120-L-TV	4 300	54,5	1	ZMA60/98	AM60	250	41 144	90×110×12
ZARN65125-TV	4 000	51	1	ZMA65/105	AM65	270	40 652	_
ZARN65125-L-TV	4 000	60,1	1	ZMA65/105	AM65	270	40 652	90×110×12
ZARN70130-TV	6 000	62,2	1	ZMA70/110	AM70	330	46 786	_
ZARN70130-L-TV	6 0 0 0	77,3	1	ZMA70/110	AM70	330	46 786	100×120×12
ZARN75155-TV	8 500	149	2	ZMA75/125	AM75	580	72 971	_

ZMA75/125

ZMA90/155

ZMA90/155

AM75

AM90

AM90

580

960

960

72 971

100 669

100 669

2

2

2

188

312

372





ZARN75155-L-TV

ZARN90180-TV

ZARN90180-L-TV

115×140×12

130×160×12





Seal carrier assemblies Precision locknuts Sockets BEARINX-online Easy BallScrew

		Page
Product overview	Seal carrier assemblies, precision locknuts, sockets, calculation of screw drive bearing arrangements	136
Features	Seal carrier assemblies	137
	Operating temperature	137
	Limiting speeds	138
	Precision locknuts	139
Design and	Breakaway torque	141
safety guidelines	Ultimate axial load	141
	Mounting guidelines	141
Accuracy		145
Dimension tables	Seal carrier assemblies DRS	146
	Precision locknuts AM, for axial locking	148
	Precision locknuts ZM, ZMA, for radial locking	149
	Sockets	151
Calculation of screw drive	BEARINX-online	154
bearing arrangements	Module Easy BallScrew	154
	User-friendly interface	155
	Data interchange with Schaeffler	156







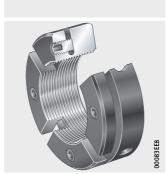
Product overview Accessories

Seal carrier assemblies



Precision locknuts For axial locking For radial locking

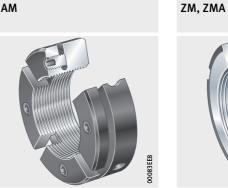
AM



Sockets



Calculation of screw drive bearing arrangements



LOCKNUT-SOCKET-KM



BEARINX-online Easy BallScrew



Features Seal carrier assemblies

Seal carrier assemblies DRS are screw mounted to the outer ring of needle roller/axial cylindrical roller bearings ZARF(L) and precisely centred in this position, *Figure 1*. They seal the bearings against outside influences.

The sealing elements are supplied as a complete set of individual parts and comprise:

- a seal carrier
- a rotary shaft seal to DIN 3760, with an elastomer part made from NBR
- a gasket
- hexagonal socket head screws for fixing the carrier to the central washer of the bearing.

DRS ZMA ZARF..-L

- Seal carrier assembly DRS
 Precision locknut ZM or ZMA
 Rotary shaft seal DIN 3760
- *Figure 1* Needle roller/axial

cylindrical roller bearing with seal carrier assembly

Operating temperature

AK\$98000

Seal carrier assemblies are suitable for operating temperatures from $-30\,^{\circ}\text{C}$ to $+120\,^{\circ}\text{C}$, restricted by the seal material.

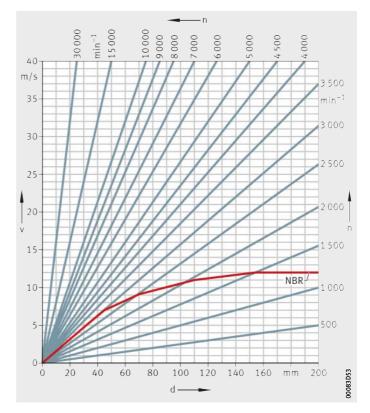






Limiting speeds

The limiting speeds for rotary shaft seals made from NBR conform to DIN 3760, *Figure 2*. These may be lower than the limiting speeds of the bearings. For the bearing arrangement, the lower of the two limiting speeds is always the decisive factor.



v = circumferential velocity d = seal lip diameter of seal carrier assembly DRS n = speed

Figure 2 Limiting speeds under pressure-free operation for rotary shaft seals

Precision locknuts

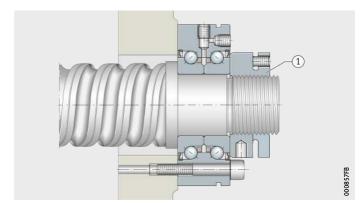
Precision locknuts are used where high axial forces must be supported and high axial runout accuracy and rigidity are required, *Figure 3*, *Figure 4*, page 140.

The thread and the axial face of the locknut in contact with the rolling bearing are produced in a single clamping operation. This allows very high axial runout accuracy to be achieved.

The available precision locknuts AM, ZM and ZMA can be used several times if handled correctly.

Axial locking by segments

Precision locknuts AM with axial locking are divided into several segments for applying the locking forces. When the hexagonal socket grub screws are tightened, the segments are elastically deformed. As a result, the thread flanks of the segments are pressed against the flanks of the shaft thread and give a high frictional force that acts to prevent loosening of the nut. The axial runout of the nut is not affected by this method of locking.



AM ZKLF..-2RS

1) Precision locknut AM

Figure 3

Mounting clearance with locknut AM for axial locking







Locking by radial locking pegs

Precision locknuts ZM and ZMA are secured against rotation by means of two radially acting locking pegs, *Figure 4*. ZMA is the heavy series.

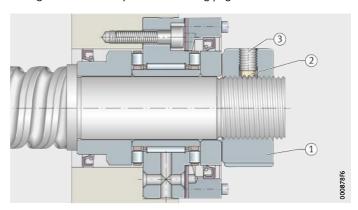
The locking pegs are manufactured together with the internal thread of the locknut. They mesh like comb teeth in the thread of the shaft without affecting the axial runout or damaging the thread.

The locking pegs are secured by grub screws with a hexagonal socket arranged concentrically over the locking pegs.

DRS ZMA ZARF..-L

① Precision locknut ZM or ZMA
② Locking peg
③ Grub screw

Figure 4
Mounting clearance
with locknut ZM, ZMA
for radial locking



Design and safety guidelines

Shaft threads for the precision locknuts must be precision machined, see table.

Recommended shaft thread

Shaft thread to DIN 13-21 to DIN 13-24 Tolerance class	
Medium	Fine
6g	4h



The journal thread must support the locknut over its whole width.

Breakaway torque

The breakaway torques M_L given in the dimension tables are based on a locknut tightened against a rigid shaft shoulder to the reference tightening torque M_{AL} and secured.

Ultimate axial load

The axial ultimate loads F_{aB} are valid for a journal thread of tolerance class 6g or better and a minimum strength of 700 N/mm². For dynamic loading, the permissible value can be taken as 75%

For dynamic loading, the permissible value can be taken as 7 of the ultimate load F_{aB} .

Mounting guidelines

The characteristics of the bearings are only valid when used in combination with INA precision locknuts and the associated tightening torques given in the dimension tables.



Locknuts must be fully screwed onto the shaft thread.

Bearings should only be mounted and dismounted in accordance with the Mounting and Maintenance Manual TPI 100.

Further information

- TPI 100, Bearings for Screw Drives, Mounting and Maintenance Manual
- Download and ordering: http://medien.schaeffler.de.







Precision locknuts for axial locking

Precision locknuts AM for axial locking can be tightened using a hook wrench of type B to DIN 1810 that engages in four, six or eight holes around the circumference, Figure 5.

The grub screws are then tightened in a crosswise sequence to the specified tightening torque M_A using an Allen key. Dismounting is carried out by uniformly loosening the grub screws of all segments in order to prevent a single segment being left tensioned.

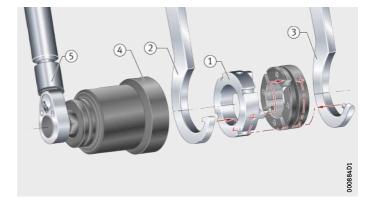


Tightening by means of just one segment is not permissible. For tightening, a socket AMS can be used that ensures uniform loading of all segments. The socket AMS must be ordered separately, see page 151.

For tightening using the socket AMS, a hook wrench of type B or type A to DIN 1810 or an FAG socket can be used, see page 144.

If precision locknuts AM are fitted using the socket AMS, a maximum of twice the tightening torque of the bearing according to the dimension table is permissible.

Segments can be axially deformed if the grub screws are not tightened uniformly in a crosswise sequence or the locknut AM is not fully screwed onto the shaft thread. The specified tightening torque M_A in the dimension table must be observed.



(1) Socket AMS (2) Hook wrench type A to DIN 1810

(3) Hook wrench type B to DIN 1810

(4) FAG socket LOCKNUT-SOCKET-KM

(5) Torque wrench

Figure 5 Mounting tools for precision locknuts AM

Schaeffler Technologies 142 | **TPI 123**

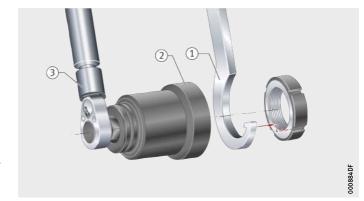
Precision locknuts for radial locking

For tightening of locknuts ZM and ZMA, a hook wrench to DIN 1810A that engages in one of the four slots around the circumference or an FAG socket can be used, *Figure 6*.

The two grub screws are then tightened alternately to the specified tightening torque M_A using an Allen key; for M_A , see dimension tables for the bearings.

For dismounting, the two grub screws are first loosened and the locking pegs loosened by light impacts with a plastic hammer on the outside surface of the locknut (in the vicinity of the screw holes).

The locknut can then be easily unscrewed without damaging the journal thread.



- ① Hook wrench type A to DIN 1810 ② FAG socket LOCKNUT-SOCKET-KM
 - 3 Torque wrench

Figure 6
Mounting tools
for precision locknuts ZM







Schaeffler Technologies TPI 123 | 143

Accessories

FAG sockets

For simple tightening and loosening of precision locknuts ZM on shafts, the sockets LOCKNUT-SOCKET-KM, *Figure* \nearrow are suitable. In combination with the socket AMS, this socket is also suitable for precision locknuts AM. They require less space on the circumference of the nut than hook wrenches and allow the use of ratchets and torque wrenches.

For increased reliability, FAG sockets should be secured using a locking pin and rubber washer. They therefore have a hole for the locking pin and a groove for the rubber washer. The locking pin and rubber washer are included in the scope of delivery.



Figure 7
FAG socket
for precision locknuts ZM
and sockets AMS

Accuracy

Accuracy of precision locknuts: see table.

Axial runout

	Metric thread in accordance with DIN 13-21 to DIN 13-24 Tolerance class
μm	Fine
5	5H

The axial face and thread are machined in a single clamping operation, in order to achieve very high accuracy. Before measurement, the precision locknut is clamped against a rigid shoulder on a threaded mandrel to a defined tightening torque. The grub screws remain loosened.



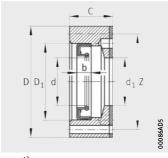




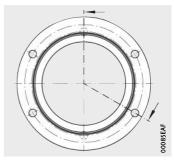
Schaeffler Technologies TPI 123 | 145

Seal carrier assemblies

For ZARF, light series





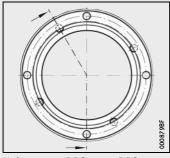


Hole pattern DRS1560, DRS1762

Dimension table · Dimensions in mm											
Designation	Mass	Dimens				Rotary	shaft sea	al		Matching bearing	
	m	D C d ₁ Z		Z	d	D ₁	b	Socket head screw to DIN EN ISO 4762			
	≈ kg								each 4 piece		
DRS1560	0,16	60	14	35	52,4	35	45	7	M3×20	ZARF1560(-L)-TV	
DRS1762	0,18	62	15,5	38	54,4	38	47	7	M3×25	ZARF1762(-L)-TV	
DRS2068	0,2	68	17	42	60,4	42	55	8	M3×25	ZARF2068(-L)-TV	
DRS2575	0,22	75	17	47	67,4	47	62	6	M3×25	ZARF2575(-L)-TV	
DRS3080	0,26	80	17	52	73,4	52	68	8	M3×25	ZARF3080(-L)-TV	
DRS3590	0,38	90	19	60	80	60	72	8	M4×25	ZARF3590(-L)-TV	
DRS40100	0,47	100	19	65	90	65	80	8	M4×30	ZARF40100(-L)-TV	
DRS45105	0,53	105 20 70 95			95	70	85	8	M4×30	ZARF45105(-L)-TV	
DRS50115	0,54	115	20	78	106	78	100	10	M3×30	ZARF50115(-L)-TV	

¹⁾ The seal carrier assembly is supplied as a sealing set in individual parts comprising:

- a seal carrier
- a rotary shaft seal
- a gasket
- socket head screws.



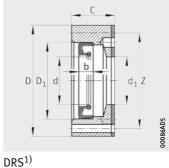
Hole pattern DRS3080, DRS3590, DRS50115

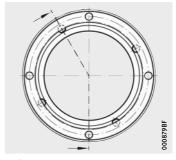


Hole pattern DRS2068, DRS2575, DRS40100, DRS45105

Seal carrier assemblies

For ZARF, heavy series





Hole pattern

Dimension table - Dimensions in mm												
Designation	Mass	Dimen	sions			Rotary	shaft sea	Matching bearing				
	m ≈ kg	D	С	d ₁	Z	d D ₁ b		Socket head screw to DIN EN ISO 4762 each 4 piece				
DRS2080	0,3	80	22	52	73,4	52	68	8	M3×30	ZARF2080(-L)-TV		
DRS2590	0,38	90	22	62	81	62	75	10	M3×30	ZARF2590(-L)-TV		
DRS30105	0,67	105	25	68	95	68	85	10	M4×35	ZARF30105(-L)-TV		
DRS35110	0,6	110	25	73	101	73	95	10	M3×30	ZARF35110(-L)-TV		
DRS40115	0,7	115	27,5	78	106	78	100	10	M3×35	ZARF40115(-L)-TV		
DRS45130	1,1	130	31	90	120	90	110	12	M4×40	ZARF45130(-L)-TV		
DRS50140	1,4	140	30	95	127,5	95	115	13	M5×40	ZARF50140(-L)-TV		
DRS55145	1,42	145	30	100	132,5	100	120	12	M5×40	ZARF55145(-L)-TV		
DRS60150	1,42	150	30	105	137,5	105	125	12	M5×40	ZARF60150(-L)-TV		
DRS65155	1,43	155	30	110	142,5	110	130	12	M5×40	ZARF65155(-L)-TV		
DRS70160	1,5	160	30	115	147,5	115	135	13	M5×40	ZARF70160(-L)-TV		
DRS75185	2,4	185	36	135	172,5	135	160	15	M5×50	ZARF75185(-L)-TV		
DRS90210	2,7	210	38	160	194	160	180	15	M5×50	ZARF90210(-L)-TV		

¹⁾ The seal carrier assembly is supplied as a sealing set in individual parts comprising:

- a seal carrier
- a rotary shaft seal
- a gasket
- socket head screws.

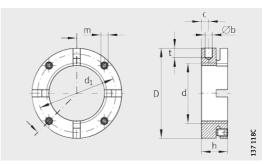






Precision locknuts

For axial locking



AM15 to AM40 with 4 segments AM45 to AM90 with 6 segments AM100 to AM130 with 8 segments

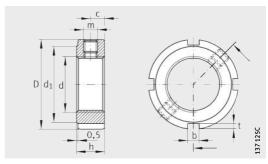
Dimension	table · [imensions in	mm											
Desig- nation	Mass	Dimensions	i							Grub screw	Locknut ¹⁾			
										Tight- ening torque	Axial rupture load	Break- away torque	Reference tightening torque	Mass moment of inertia
	m	d	D	h	b	t	d_1	С	m	MA	F _{aB} M _L at M _{AL} M _A			M _M
	≈ kg				H11					Nm	N	Nm	Nm	kg⋅cm²
AM15	0,06	M15×1	30	18	4	5	24	5	M5	3	100 000	20	10	0,09
AM17	0,07	M17×1	32	18	4	5	26	5	M5	3	120 000	25	15	0,11
AM20	0,13	M20×1	38	18	4	6	31	5	M6	5	145 000	45	18	0,23
AM25	0,16	M25×1,5	45	20	5	6	38	6	M6	5	205 000	60	25	0,49
AM30	0,20	M30×1,5	52	20	5	7	45	6	M6	5	250 000	70	32	0,86
AM30/65	0,50	M30×1,5	65	30	6	8	45	6	M6	5	400 000	70	32	2,8
AM35/58	0,23	M35×1,5	58	20	5	7	51	6	M6	5	280 000	90	40	1,3
AM35	0,33	M35×1,5	65	22	6	8	58	6	M6	5	330 000	100	40	2,4
AM40	0,30	M40×1,5	65	22	6	8	58	6	M6	5	350 000	120	55	2,3
AM40/85	0,75	M40×1,5	85	32	6	8	58	6	M6	5	570 000	120	55	7,6
AM45	0,34	M45×1,5	70	22	6	8	63	6	M6	5	360 000	220	65	2,9
AM50	0,43	M50×1,5	75	25	6	8	68	8	M6	5	450 000	280	85	4,3
AM55	0,60	M55×2	85	26	6	8	75	8	M8	15	520 000	320	95	7,7
AM60	0,65	M60×2	90	26	6	8	80	8	M8	15	550 000	365	100	9,4
AM65	0,83	M65×2	100	26	8	10	88	8	M8	15	560 000	400	120	14,6
AM70	0,79	M70×2	100	28	8	10	90	9	M8	15	650 000	450	130	14,7
AM75	1,23	M75×2	115	30	8	10	102	10	M10	20	750 000	610	150	29
AM80	0,93	M80×2	110	30	8	10	98	10	M10	20	670 000	770	160	21,3
AM85	0,97	M85×2	115	30	8	10	102	10	M10	20	690 000	930	180	24,8
AM90	1,53	M90×2	130	32	8	10	118	13	M10	20	900 000	1 100	200	48
AM100	1,12	M100×2	130	30	8	10	118	10	M10	20	740 000	1 200	250	38
AM110	1,22	M110×2	140	30	8	10	128	10	M10	20	770 000	1 300	250	48
AM120	1,56	M120×2	155	30	8	10	142	10	M10	20	880 000	1 450	250	75
AM130	1,67	M130×2	165	30	8	10	152	10	M10	20	900 000	1 600	250	92

¹⁾ Attention!

If precision locknuts AM are fitted using the socket AMS, a maximum of twice the tightening torque of the bearing according to the dimension table is permissible.

Precision locknuts

For radial locking



ZM, ZMA

Dimension ta	1									1				
Desig- nation	Mass	Dimensions	5							Grub screw	Locknut			
										Tight- ening torque	Axial rupture load	Break- away torque	Reference tightening torque	Mass moment of inertia
	m	d	D	h	b	t	d_1	С	m	M _A	F _{aB}	M _L at M	AL	M _M
	≈ kg									Nm	N	Nm	Nm	kg⋅cm ²
ZM06	0,01	M6×0,5	16	8	3	2	11	4	M4	1	17 000	20	2	0,004
ZM08 ¹⁾	0,01	M8×0,75	16	8	3	2	11	4	M4	1	23 000	25	4	0,004
ZM10 ¹⁾	0,01	M10×1	18	8	3	2	14	4	M4	1	31 000	30	6	0,006
ZM12	0,015	M12×1	22	8	3	2	18	4	M4	1	38 000	30	8	0,013
ZM15	0,018	M15×1	25	8	3	2	21	4	M4	1	50 000	30	10	0,021
ZMA15/33	0,08	M15×1	33	16	4	2	28	8	M5	3	106 000	30	10	0,14
ZM17	0,028	M17×1	28	10	4	2	23	5	M5	3	57 000	30	15	0,401
ZM20	0,035	M20×1	32	10	4	2	27	5	M5	3	69 000	40	18	0,068
ZMA20/38	0,12	M20×1	38	20	5	2	33	10	M5	3	174 000	40	18	0,297
ZMA20/52	0,32	M20×1	52	25	5	2	47	12,5	M5	3	218 000	40	18	1,38
ZM25	0,055	M25×1,5	38	12	5	2	33	6	M6	5	90 000	60	25	0,157
ZMA25/45	0,16	M25×1,5	45	20	5	2	40	10	M6	5	211 000	60	25	0,572
ZMA25/58	0,43	M25×1,5	58	28	6	2,5	52	14	M6	5	305 000	60	25	2,36
ZM30	0,075	M30×1,5	45	12	5	2	40	6	M6	5	112 000	70	32	0,304
ZMA30/52	0,22	M30×1,5	52	22	5	2	47	11	M6	5	270 000	70	32	1,1
ZMA30/65	0,55	M30×1,5	65	30	6	2,5	59	15	M6	5	390 000	70	32	3,94
ZM35	0,099	M35×1,5	52	12	5	2	47	6	M6	5	134 000	80	40	0,537
ZMA35/58	0,26	M35×1,5	58	22	6	2,5	52	11	M6	5	300 000	80	40	1,66
ZMA35/70	0,61	M35×1,5	70	30	6	2,5	64	15	M6	5	460 000	80	40	5,2
ZM40	0,14	M40×1,5	58	14	6	2,5	52	7	M6	5	157 000	95	55	0,945
ZMA40/62	0,27	M40×1,5	62	22	6	2,5	56	11	M8	15	310 000	95	55	2,07
ZMA40/75	0,67	M40×1,5	75	30	6	2,5	69	15	M8	15	520 000	95	55	6,72
ZM45	0,17	M45×1,5	65	14	6	2,5	59	7	M6	5	181 000	110	65	1,48
ZMA45/68	0,35	M45×1,5	68	24	6	2,5	62	12	M8	15	360 000	110	65	3,2
ZMA45/85	0,92	M45×1,5	85	32	7	3	78	16	M8	15	630 000	110	65	11,9
ZM50	0,19	M50×1,5	70	14	6	2,5	64	7	M6	5	205 000	130	85	1,92
ZMA50/75	0,43	M50×1,5	75	25	6	2,5	68	12,5	M8	15	415 000	130	85	4,89
ZMA50/92	1,06	M50×1,5	92	32	8	3,5	84	16	M8	15	680 000	130	85	16,1
ZM55	0,23	M55×2	75	16	7	3	68	8	M6	5	229 000	150	95	2,77
ZMA55/98	1,17	M55×2	98	32	8	3,5	90	16	M8	15	620 000	150	95	20,5

¹⁾ When clamped, the grub screw protrudes by \approx 0,5 mm.

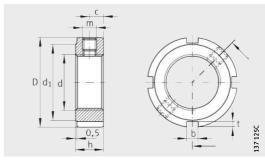






Precision locknuts

For radial locking

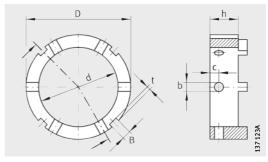


ZM, ZMA

Dimension tab	le (cont	inued) · Din	nensio	ns in	mm									
Designation	Mass	Dimension	ıs							Grub screw	Locknut			
										Tight- ening torque	Axial rupture load	Break- away torque	Reference tightening torque	Mass moment of inertia
	m	d	D	h	b	t	d_1	С	m	M _A	F _{aB} M _L at M _{AL}			M_{M}
	≈ kg									Nm	N	Nm	Nm	$\text{kg}\cdot\text{cm}^2$
ZM60	0,25	M60×2	80	16	7	3	73	8	M6	5	255 000	180	100	3,45
ZMA60/98	1,07	M60×2	98	32	8	3,5	90	16	M8	15	680 000	180	100	19,6
ZM65	0,27	M65×2	85	16	7	3	78	8	M6	5	280 000	200	120	4,24
ZMA65/105	1,21	M65×2	105	32	8	3,5	97	16	M8	15	750 000	200	120	25,6
ZM70	0,36	M70×2	92	18	8	3,5	85	9	M8	15	305 000	220	130	6,61
ZMA70/110	1,4	M70×2	110	35	8	3,5	102	17,5	M8	15	810 000	220	130	33
ZM75	0,4	M75×2	98	18	8	3,5	90	9	M8	15	331 000	260	150	8,41
ZMA75/125	2,11	M75×2	125	38	8	3,5	117	19	M8	15	880 000	260	150	62,2
ZM80	0,46	M80×2	105	18	8	3,5	95	9	M8	15	355 000	285	160	11,2
ZMA80/120	1,33	M80×2	120	35	8	4	105	17,5	M8	15	810 000	285	160	44,6
ZM85	0,49	M85×2	110	18	8	3,5	102	9	M8	15	385 000	320	190	13,1
ZM90	0,7	M90×2	120	20	10	4	108	10	M8	15	410 000	360	200	21,8
ZMA90/130	2,01	M90×2	130	38	10	4	120	19	M8	15	910 000	360	200	64,1
ZMA90/155	3,36	M90×2	155	38	10	4	146	19	M8	15	1 080 000	360	200	150
ZM100	0,77	M100×2	130	20	10	4	120	10	M8	15	465 000	425	250	28,6
ZMA100/140	2,23	M100×2	140	38	12	5	128	19	M10	20	940 000	425	250	82,8
ZM105	1,05	M105×2	140	22	12	5	126	11	M10	20	495 000	475	300	44,5
ZM110	1,09	M110×2	145	22	12	5	133	11	M10	20	520 000	510	350	50,1
ZM115	1,13	M115×2	150	22	12	5	137	11	M10	20	550 000	550	400	56,2
ZM120	1,28	M120×2	155	24	12	5	138	12	M10	20	580 000	600	450	68,4
ZM125	1,33	M125×2	160	24	12	5	148	12	M10	20	610 000	640	500	76,1
ZM130	1,36	M130×2	165	24	12	5	149	12	M10	20	630 000	700	550	84,3
ZM140	1,85	M140×2	180	26	14	6	160	13	M12	38	690 000	800	600	133
ZM150	2,24	M150×2	195	26	14	6	171	13	M12	38	750 000	900	650	188

Sockets

For precision locknuts suitable for axial locking



AMS

Dimension table · Dimensions in mm											
Designation	Mass	Dimens	ions						For precision locknuts		
	m ≈ kg	d	d D h b c B								
AMS20	0,047	22	22 32 14 4				4	2	AM15, AM17, AM20		
AMS30	0,093	35	45	15	5	5	5	2	AM25, AM30, AM35/58, AM30/65		
AMS40	0,217	45	65	16	6	6	6	2,5	AM35, AM40		
AMS50	0,245	53	70	19	6	6	6	2,5	AM45, AM50		
AMS60	0,37	65	85	20	6	6	7	3	AM55, AM60		
AMS70	0,615	75	98	25	8	10	8	3,5	AM65, AM70		
AMS80	0,755	85	110	25	8	10	8	3,5	AM75, AM80, AM85		
AMS90	1,215	95	130	25	8	10	10	4	AM90		
AMS110	0,74	110	130	25	8	10	10	4	AM100, AM110		
AMS130	1,485	130	155	25	8	10	12	5	AM120, AM130		

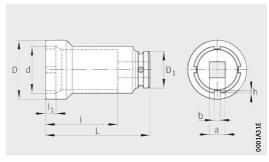






Sockets

For precision locknuts suitable for radial or axial locking For torque wrench Suitable for combination with sockets AMS

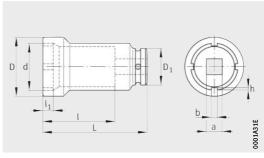


LOCKNUT-SOCKET-KM

Dimension table · Dimension	1							l		la
Designation	Mass	Dimen	sions					Suitable for	1 .	Suitable in combination
								Precision locknut	Socket	with AMS for
		d	D	D_1	L	l	a	ZM	AMS	AM
	≈ kg						inch			
LOCKNUT-SOCKET-KM0	0,1	18,1	22	22	57	44	3/8	ZM10	_	_
LOCKNUT-SOCKET-KM1	0,1	22,2	28	22	57	44	3/8	ZM12	_	_
LOCKNUT-SOCKET-KM2	0,2	25,2	33	30	82	60	1/2	ZM15	_	_
								-		AM15
LOCKNUT-SOCKET-KM3	0,24	28,2	36	30	82	60	1/2	ZM17	AMS20	AM17
								-		AM20
LOCKNUT-SOCKET-KM4	0,28	32,2	38	30	82	56	1/2	ZM20	_	_
LOCKNUT-SOCKET-KM5	0,38	38,2	46	30	82	56	1/2	ZMA20/38		_
LOCKNOT-SOCKLT-KNIS	0,56	36,2	40	50	02	50	1/2	ZM25		
								ZMA25/45		AM25
LOCKNUT-SOCKET-KM6	0,42	45,2	53	30	82	56	1/2	ZM30	AMS30	AM30
LOCKNOT-SOCKLT-KNIO	0,42	45,2	33	30	02	50	17/2	-	AMSJU	AM35/58
								-		AM30/65
								ZMA20/52		
LOCKNUT-SOCKET-KM7	0,45	52,2	60	30	82	56	1/2	ZMA30/52	_	_
								ZM35		
								ZMA25/58		
LOCKNUT-SOCKET-KM8	0,61	58,3	68	30	82	56	1/2	ZMA35/58		_
								ZM40		
								ZMA30/65	-	_
LOCKNUT-SOCKET-KM9	0,8	65,4	73,5	44	90	62	3/4	-	AMS40	AM35
LOCKHOT-SOCKET-KINS	0,0	05,4	7 5,5	44	100	02	- 7 4	-	AMS40	AM40
								ZM45	-	_
								ZMA35/70	_	_
LOCKNUT-SOCKET-KM10	0,85	70,4	78,5	44	90	62	3/4	-	AMS50	AM45
								ZM50	AMOOU	AM50
								ZMA40/75		
LOCKNUT-SOCKET-KM11	0,9	75,4	83,5	44	90	62	3/4	ZMA50/75		_
				7.7				ZM55		

Sockets

For precision locknuts suitable for radial or axial locking For torque wrench Suitable for combination with sockets AMS



LOCKNUT-SOCKET-KM

Dimension table (continued	l) · Dimer	nsions in r	nm							
Designation	Mass	Dimensi	ons					Suitable for		Suitable
								Precision locknut	Socket	in combination with AMS for
		d	D	D_1	L	l	a	ZM	AMS	AM
	≈ kg						inch			
								ZMA45/85	-	-
LOCKNUT-SOCKET-KM13	1,1	85,4	94	44	90	60	3/4	_	AMS60	AM55
LOCKNOT-SOCKLT-KM15	1,1	65,4	94	44	90	00	7/4	_	AMSOU	AM60
								ZM65	-	-
LOCKNUT-SOCKET-KM14	2,2	92,5	103	76	110	74	1	ZMA50/92		_
EGGRATOT GGGRAT RIMET	2,2	72,3	103	, 0	110	, -	_	ZM70		
								ZMA55/98		_
								ZMA60/98		
LOCKNUT-SOCKET-KM15	2,3	98,5	109	76	110	74	1	_	AMS70	AM65
								_		AM70
								ZM75	_	_
LOCKNUT-SOCKET-KM16	2,45	105,6	116	76	110	74	1	ZMA65/105		_
	_,,,,	,-		' -				ZM80		
								ZMA70/110	-	_
LOCKNUT-SOCKET-KM17	2,6	110,6	121	76	110	74	1	_		AM75
	2,0	,-					_	_	AMS80	AM80
								ZM85		AM85
LOCKNUT-SOCKET-KM18	2,9	120,6	131	76	110	74	1	ZMA80/120		_
		·						ZM90		
LOCKNUT-SOCKET-KM19	3,05	125,6	137	76	110	74	1	ZMA75/125	-	-
								ZMA90/130	AMS90	AM90
LOCKNUT-SOCKET-KM20	3,3	130,6	143	76	110	74	1	ZM100	AMS110	AM100
								-		AM110
LOCKNUT-SOCKET-KM21	3,43	115	153	76	110	80	1	ZMA100/140		_
	<u></u>							ZM105		
LOCKNUT-SOCKET-KM22	3,54	120	158	76	110	80	1	ZM110	_	_
								ZMA90/155	-	_
LOCKNUT-SOCKET-KM24	4,15	130	170	76	110	80	1	ZM120	AMS130	AM120
								_		AM130







BEARINX-online Easy BallScrew

Calculation of screw drive bearing arrangements

In order to ensure the correct use of products, Schaeffler offers the necessary support in partnership with the customer, from the very first design idea to delivery of the products. A key aspect of design advisory work is the selection and design of rolling bearings. The optimum selection and design of products gives Schaeffler customers a competitive advantage. To this end, Schaeffler has been successfully using calculation programs for many years.

BEARINX-online

Through the use of BEARINX, specific bearing loads can be calculated in detail, represented and documented taking account of operating and environmental conditions. The contact pressure on each individual rolling element is included in the calculation process.

BEARINX takes account of factors including the following:

- the non-linear elastic deflection behaviour of rolling elements
- elasticity of the bearing rings
- displacements in contact angle induced by load
- the actual contact pressure taking account of the misalignment and profiling of rolling elements.

Module Easy BallScrew

For the calculation of screw drive bearing arrangements, there is a separate BEARINX-online module which can be used to calculate the most commonly used arrangements of bearings, *Figure 1*.

In addition to the existing BEARINX-online software, the new module Easy BallScrew fits logically into the proven range of Easy calculation programs.

With Easy BallScrew, it is possible to calculate the minimumstatic load safety factor and the adjusted reference rating life for all arrangements of bearings. For locating/locating bearing arrangements, the spindle tensioning force and the value for effective spindle elongation due to tensioning as well as all adjustment values for mounting are also outputted.

Easy BallScrew has access to the Schaeffler bearing database. Axial angular contact ball bearings and needle roller/axial cylindrical roller bearings as well as other screw drive bearings can be easily integrated.



Figure 1
BEARINX-online,
module Easy BallScrew

Influencing factors typical for the application

The following influencing factors typical for the application are taken into consideration:

- location of the bearing on the shaft
- consideration of shaft material
- consideration of bearing set combinations
- operating temperature range
- lubrication and severity of contamination.

User-friendly interface

The program provides four common arrangements of bearings for screw drives as preconfigured calculation models, *Figure 2*:

- locating bearing arrangement on one side
- locating/non-locating bearing arrangement
- locating/locating bearing arrangement (spindle tensioning by locknut)
- locating/locating bearing arrangement (spindle tensioning by shim).

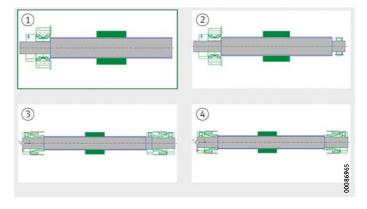
All the necessary operating data can be inputted separately, with navigation by means of user-friendly menus:

- arrangement of bearings
- bearing selection
- load case data
- lubrication.

Data can be inputted quickly and easily thanks to the self-explanatory menu navigation, *Figure 3*, page 156. The geometrical data of INA and FAG rolling bearings are easily loaded from an integrated database. Operating conditions are inputted in the form of a duty cycle in a table, *Figure 4*, page 156.

- Locating bearing arrangement, one side
- 2 Locating/non-locating bearing arrangement
- ③ Locating/locating bearing arrangement (spindle tensioning by locknut)
- 4 Locating/locating bearing arrangement (spindle tensioning by shim)

Figure 2
Preconfigured calculation models









Schaeffler Technologies TPI 123 | 155

BEARINX-online Easy BallScrew

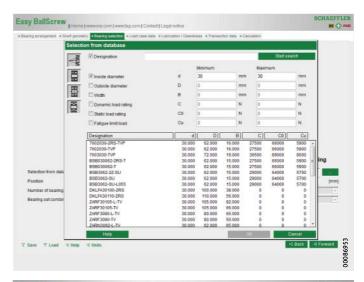


Figure 3 Bearing selection from database



Figure 4 Input of speed/load cycle

Data interchange with Schaeffler

All input data can be stored. As a result, relevant modifications can be quickly made to an existing application case without the need for duplicated input of data. Furthermore, the stored file can be exchanged with the Schaeffler engineering service in order to achieve an optimum bearing design.

The BEARINX-online modules run on the powerful calculation servers at Schaeffler. Once calculation has been successfully carried out, the most significant calculation results are displayed. In addition, a comprehensive results view can be called up.

Registration for the calculation program

http://bearinx-online-easy-ballscrew.schaeffler.com



The module BEARINX-online Easy BallScrew is available only in an online format and can be used free of charge.

Initial registration takes only a short time and you can then start your calculation work straight away.

Addresses

Worldwide

You can find all the addresses and contacts for Schaeffler at www.schaeffler.com

Argentina

Schaeffler Argentina S.r.l. Av. Alvarez Jonte 1938 C1416EXR Buenos Aires Tel. +(54) 11 / 40 16 15 00 Fax +(54) 11 / 45 82 33 20 info-ar@schaeffler.com

Armenia

Schaeffler Ukraine GmbH Zhylyanskaya Str. 75, 5. Stock, Businesscenter «Eurasia» 01032 Kiew Ukraine Tel. +(380) 44 520 13 80 Fax +(380) 44 520 13 81 info.ua@schaeffler.com

Australia

Schaeffler Australia Pty Ltd Level 1, Bldg 8, Forest Central Business Park 49 Frenchs Forest Road Frenchs Forst, NSW 2086 Tel. +(61) 2 8977 1000 Fax +(61) 2 9452 4242 sales.au@schaeffler.com

Schaeffler Australia Pty Limited Suite 14, Level 3 74 Doncaster Road North Balwyn, VIC 3104 Tel. +(61) 3 9859 8020 Fax +(61) 3 9859 8767 milos.grujic@schaeffler.com

Schaeffler Australia Pty Ltd Unit 3, 47 Steel Place Morningside, QLD 4170 Tel. +(61) 7 3399 9161 Fax +(61) 7 3399 9351 martin.grosvenor@schaeffler.com

Austria

Schaeffler Austria GmbH Ferdinand-Pölzl-Straße 2 2560 Berndorf-St. Veit Tel. +(43) 2672 202-0 Fax +(43) 2672 202-1003 info.at@schaeffler.com

Belarus

Schaeffler Technologies AG & Co. KG Repräsentanz Weißrussland Odoewskogo 117, office 317 220015 Minsk Tel. +(375) 17 336 94 81 Fax +(375) 17 336 94 82 info.by@schaeffler.com

Belgium

Schaeffler Belgium S.P.R.L./B.V.B.A. Avenue du Commerce, 38 1420 Braine L'Alleud Tel. +(32) 2 3 89 13 89 Fax +(32) 2 3 89 13 99 info.be@schaeffler.com

Bolivia

Schaeffler Chile Ltda. Jose Tomas Rider 1051 Providencia 7501037 Santiago Chile Tel. +(56) 2 477 5000 Fax +(56) 2 2223 9941 info-cl@schaeffler.com

Bosnia-Herzegovina

Schaeffler Hrvatska d.o.o. Ogrizovićeva 28b 10000 Zagreb Croatia Tel. +(385) 1 37 01 943 Fax +(385) 1 37 64 473 info.hr@schaeffler.com

Brazil

Schaeffler Brasil Ltda. Av. Independência, 3500-A Bairro Éden 18087-101 Sorocaba, SP Tel. +(55) 15 3335 1422 Tel. 0800 11 10 29 Fax +(55) 1533 35 19 60 sac.br@schaeffler.com

Bulgaria

Schaeffler Bulgaria OOD Dondukov-Blvd. No 62 Eing. A, 6. Etage, App. 10 1504 Sofia Tel. +(359) 2 946 3900 +(359) 2 943 4008 Fax +(359) 2 943 4134 info.bg@schaeffler.com

Canada

Schaeffler Canada Inc. 100 Alexis Nihon Suite 390 Montréal, QC H4M 2N8 Tel. +(1) 514-748-5111 Tel. 800-361-5841 Toll Free Fax +(1) 514-748-6111 info.ca@schaeffler.com

Schaeffler Canada Inc. 2871 Plymouth Drive Oakville, ON L6H 5S5 Tel. +(1) 905-829-2750 Tel. 800-263-4397 Toll Free Fax +(1) 905-829-2563 info.ca@schaeffler.com

Schaeffler Canada Inc. #106, 7611 Sparrow Drive Leduc, AB T9E 0H3 Tel. +(1) 780-980-3016 Tel. 800-663-9006 Toll Free Fax +(1) 780-980-3037 info.ca@schaeffler.com

Chile

Schaeffler Chile Ltda. Jose Tomas Rider 1051 Providencia 7501037 Santiago Chile Tel. +(56) 2 477 5000 Fax +(56) 2 2223 9941 info-cl@schaeffler.com

China

Schaeffler Holding (China) Co., Ltd. No. 1 Antuo Road (west side of Anhong Road) AnTing, JiaDing District 201804 Shanghai Tel. +(86) 21 3957 6666 Fax +(86) 21 3957 6600 info_china@schaeffler.com

Colombia

Schaeffler Colômbia Ltda. Cra. 10 N° 97A 13 Torre A Ofic 209 Bogotá Trade Center Bogotá Tel. +(57) 1 621 53 00 Fax +(57) 1 621 03 22 info-co@schaeffler.com

Addresses

Croatia

Schaeffler Hrvatska d.o.o. Ogrizovićeva 28b 10000 Zagreb Tel. +(385) 1 37 01 943 Fax +(385) 1 37 64 473 info.hr@schaeffler.com

Czech Republic

Schaeffler CZ s.r.o. Průběžná 74a 100 00 Praha 10 Tel. +(420) 267 298 111 Fax +(420) 267 298 110 info.cz@schaeffler.com

Denmark

Schaeffler Danmark ApS Jens Baggesens Vej 90P 8200 Aarhus N Tel. +(45) 70 15 44 44 Fax +(45) 70 15 22 02 info.dk@schaeffler.com

Egypt

Delegation Office Schaeffler Technologies 25, El Obour Buildings – Floor 18 – Flat 4 Salah Salem St. 11371 Cairo Tel. +(20) 2 24012432 Fax +(20) 2 22612637 schaeffleregypt@schaeffleregypt.com

Estonia

Schaeffler Technologies Repräsentanz Baltikum Duntes iela 23a 1005 Riga Latvia Tel. +(371) 67 06 37 95 Fax +(371) 67 06 37 96 info.lv@schaeffler.com

Finland

Schaeffler Finland Oy Lautamiehentie 3 02770 Espoo Tel. +(358) 207 36 6204 Fax +(358) 207 36 6205 info.fi@schaeffler.com

France

Schaeffler France SAS 93, route de Bitche, BP 30186 67506 Haguenau Tel. +(33) 3 88 63 40 40 Fax +(33) 3 88 63 40 41 info.fr@schaeffler.com

Germany

Schaeffler Technologies AG & Co. KG Industriestraße 1 – 3 91074 Herzogenaurach Tel. +(49) 9132 82-0 Fax +(49) 9132 82-4950 info.de@schaeffler.com

Schaeffler Technologies AG & Co. KG Georg-Schäfer-Str. 30 97421 Schweinfurt Tel. +(49) (9721) 91-0 Fax +(49) (9721) 91-3435 faginfo@schaeffler.com

FAG Aerospace GmbH & Co. KG Georg-Schäfer-Str. 30 97421 Schweinfurt Tel. +49 (0 97 21) 91-33 72 astrid.hofmann@schaeffler.com

Hungary

Schaeffler Magyarország Ipari Kft. Rétköz u.5 1118 Budapest Tel. +(36) 1 4 81 30 50 Fax +(36) 1 4 81 30 53 budapest@schaeffler.com

India

FAG Bearings India Limited Maneja Vadodara 390 013 Tel. +(91) 26 52 6426-51 Fax +(91) 26 52 6388-04/-10 info.fag.in@schaeffler.com

Italy

Schaeffler Italia S.r.l. Via Dr. Georg Schaeffler, 7 28015 Momo (Novara) Tel. +(39) 3 21 92 92 11 Fax +(39) 3 21 92 93 00 info.it@schaeffler.com

Japan

Schaeffler Japan Co., Ltd. Schaeffler R&D Center Building, Yokohama Business Park, 134 Godo-cho, Hodogaya-ku, 240-0005 Yokohama Tel. +(81) 45 287 9001 Fax +(81) 45 287 9011 info-japan@schaeffler.com

Korea

Schaeffler Korea Corporation – Guro Office A-501, 1258, Guro-dong, Guro-gu, Seoul, 152-721 Tel. +(82) 2 2625-8572 Fax +(82) 2 2611-6075

Latvia

Schaeffler Technologies Repräsentanz Baltikum Duntes iela 23a 1005 Riga Tel. +(371) 7 06 37 95 Fax +(371) 7 06 37 96 info.lv@schaeffler.com

Lithuania

Schaeffler Technologies Repräsentanz Baltikum Duntes iela 23a 1005 Riga Latvia Tel. +(371) 7 06 37 95 Fax +(371) 7 06 37 96 info.lv@schaeffler.com

Luxembourg

Schaeffler Belgium S.P.R.L./B.V.B.A. Avenue du Commerce, 38 1420 Braine L'Alleud Belgium Tel. +(32) 2 3 89 13 89 Fax +(32) 2 3 89 13 99 info.be@schaeffler.com

Malaysia

Schaeffler Bearings (Malaysia) Sdn. Bhd. 5-2 Wisma Fiamma, No. 20 Jalan 7A/62A Bandar Menjalara 52200 Kuala Lumpur Tel. +(60) 3-6275 0620 Fax +(60) 3 6275 6421 marketing_my@schaeffler.com

Schaeffler Bearings (Malaysia) Sdn. Bhd. (Penang Branch) No. B-02-28, 2nd Floor, Krystal Point 303, Jalan Sultan Azlan Shah 11900 Sungai Nibong Tel. +(60) 4 642 3708/3781 Fax +(60) 4 642 3724

Mexico

INA México, S.A. de C.V. -Rodamientos FAG, S.A. de C.V. Henry Ford #141 Col. Bondojito Deleg. Gustavo A. Madero 07850 Mexico D.F. Tel. +(52) 55 5062 6085 Fax +(52) 55 5739 5850 distr.indl.mx@schaeffler.com

Netherlands

Schaeffler Nederland B.V. Gildeweg 31 3771 NB Barneveld Tel. +(31) 342 40 30 00 Fax +(31) 342 40 32 80 info.nl@schaeffler.com

New Zealand

Schaeffler New Zealand (Unit R, Cain Commercial Centre) 20 Cain Road 1135 Penrose Tel. +(64) 9 583 1280 Fax +(64) 9 583 1288 sales.nz@schaeffler.com

Norway

Schaeffler Norge AS Grenseveien 107B 0663 Oslo Tel. +(47) 23 24 93 30 Fax +(47) 23 24 93 31 info.no@schaeffler.com

Philippines

Schaeffler Philippines Inc 5th Floor, Optima Building 221 Salcedo Street, Legaspi Village 1229 Makati City Tel. +(63) 2 759 3583 Fax +(63) 2 779 8703 marketing_ph@schaeffler.com

Poland

Schaeffler Polska Sp. z o.o. Budynek E ul. Szyszkowa 35/37 02-285 Warszawa Tel. +(48) 22 8 78 41 20 Fax +(48) 22 8 78 41 22 info.pl@schaeffler.com

Portugal

INA Rolamentos Lda. Arrábida Lake Towers Rua Daciano Baptista Marques Torre C, 181, 2º piso 4400-617 Vila Nova de Gaia Tel. +(351) 22 5 32 08 00 Fax +(351) 22 5 32 08 60 info.pt@schaeffler.com

Schaeffler Portugal S.A. Rua Estrada do Lavradio 25 2500-294 Caldas da Rainha Tel. +(351) 262 837000 Fax +(351) 262 837011

Romania

S.C. Schaeffler Romania S.R.L. Aleea Schaeffler Nr. 3 507055 Cristian/Brasov Tel. +(40) 268 504816 Fax +(40) 268 505848 info.ro@schaeffler.com

Russia

Schaeffler Russland GmbH Leningradsky Prospekt 47, Bau 3 Business-Center Avion 125167 Moscow Tel. +(7) 495 7 37 76 60 Fax +(7) 495 7 37 76 61 info.ru@schaeffler.com

Saudi Arabia

Schaeffler Middle East FZE
Road SE101, Schaeffler Building
Jebel Ali Free Zone – Southside
Postbox 261808
Dubai UAE
United Arab Emirates
Tel. +(971) 4 81 44 500
Fax +(971) 4 81 44 601
info.ae@schaeffler.com

Serbia

Schaeffler Technologies Repräsentanz Serbien Branka Krsmanovica 12 11118 Beograd Tel. +(381) 11 308 87 82 Fax +(381) 11 308 87 75 fagbgdyu@orion.rs

Singapore

Schaeffler (Singapore) Pte. Ltd. 151 Lorong Chuan, #06-01 New Tech Park, Lobby A 556741 Singapore Tel. +(65) 6540 8600 Fax +(65) 6540 8668 info.sg@schaeffler.com

Slovak Republic

Schaeffler Slovensko, spol. s r.o. Ulica Dr. G. Schaefflera 1 02401 Kysucké Nové Mesto Tel. +(421) 41 4 20 51 11 Fax +(421) 41 4 20 59 18 info.sk@schaeffler.com

Schaeffler Slovensko, spol. s r.o. Nevädzova 5 821 01 Bratislava Tel. +(421) 2 43 294 260 Fax +(421) 2 48 287 820 info.sk@schaeffler.com

Slovenia

Schaeffler Slovenija d.o.o. Glavni trg 17/b 2000 Maribor Tel. +(386) 2 22 82 070 Fax +(386) 2 22 82 075 info@schaeffler.si

South Africa

Schaeffler South Africa (Pty.) Ltd. 1 End Street Ext. Corner Heidelberg Road 2000 Johannesburg Tel. +(27) 11 225 3000 Fax +(27) 11 334 1755 info.co.za@schaeffler.com

Spain

Schaeffler Iberia, S.L.U. – División Industria C/ Foment, 2 Polígono Ind. Pont Reixat 08960 Sant Just Desvern – Barcelona Tel. +(34) 93 4 80 34 10 Fax +(34) 93 3 72 92 50 info.es@schaeffler.com

Sweden

Schaeffler Sverige AB Charles gata 10 195 61 Arlandastad Tel. +(46) 8 59 51 09 00 Fax +(46) 8 59 51 09 60 info.se@schaeffler.com

Switzerland

HYDREL GmbH Badstrasse 14 8590 Romanshorn Tel. +(41) 71 4 66 66 66 Fax +(41) 71 4 66 63 33 info.ch@schaeffler.com

Taiwan

Schaeffler Taiwan Co. Ltd. 23F, No.76, Sec. 2, Dunhua S. Rd., Da'an Dist. Taipei 106 Tel. +886 2 7730 1911 Fax +886 2 2707 9964 info.tw@schaeffler.com

Thailand

Schaeffler (Thailand) Co., Ltd. 388 Exchange Tower 31st, 34th Floor, Unit 3103, 3403-3404 Sukhumvit Rd., Klongtoey Bangkok, 10110 Tel. +(66) 2697 0000 Fax +(66) 2697 0001 info.th@schaeffler.com

Addresses

Turkey

Schaeffler Rulmanlari Ticaret Limited Sirketi Ömer Faik Atakan cad. Saray Mah. Yılmaz Plaza No:3 34768 Istanbul Tel. +(90) 212 2 79 27 41 Fax +(90) 212 2 81 66 45 info.tr@schaeffler.com

Ukraine

Schaeffler Ukraine GmbH Zhylyanskaya Str. 75, 5. Stock, Businesscenter «Eurasia» 01032 Kiew Ukraine Tel. +(380) 44 520 13 80 Fax +(380) 44 520 13 81 info.ua@schaeffler.com

United Kingdom

Schaeffler (UK) Ltd Forge Lane, Minworth Sutton Coldfield B76 1AP Tel. +(44) 121 3 13 58 70 Fax +(44) 121 3 13 00 80 info.uk@schaeffler.com

The Barden Corporation (UK) Limited Plymbridge Road, Estover Plymouth, PL6 7LH Tel. +(44) 1752 73 55 55 Fax +(44) 1752 73 34 81 bardenbearings@schaeffler.com

United Arab Emirates

Schaeffler Middle East FZE Road SE101, Schaeffler Building Jebel Ali Free Zone - Southside Postbox 261808 Dubai UAE Tel. +971 4 81 44 500 Fax +971 4 81 44 601 info.ae@schaeffler.com

USA

Schaeffler Group USA Inc. 200 Park Avenue P.O. Box 1933 Danbury, CT 06813-1933 Tel. +(1) 203 790 5474 Fax +(1) 203 830 8171 Walter.Newton@schaeffler.com

The Barden Corporation 200 Park Avenue P.O. Box 2449 Danbury, CT 06813-2449 Tel. +(1) 203 744 2211 Fax +(1) 203 744 3756 sales@bardenbearings.com

Schaeffler Group USA Inc. 308 Springhill Farm Road Corporate Offices Fort Mill, SC 29715 Tel. +(1) 803 548 8500 Fax +(1) 803 548 8599 info.us@schaeffler.com

Schaeffler Group USA Inc. 5370 Wegman Drive Valley City, OH 44280-9700 Tel. +(1) 800 274 5001 Fax +(1) 330 273 3522 luk-ina-fag-as.us@schaeffler.com

Venezuela

Schaeffler Venezuela C.A. Urbanización San José de Tarbes Torre BOD, Piso 14, Oficina 14-1 Valencia Tel. +(58) 58 241 825 9250 Fax +(58) 58 241 825 9705 ana.acevedo@schaeffler.com

Vietnam

Schaeffler Vietnam Co., Ltd – Ho Chi Minh Sales Office 6th Floor, TMS Building. 172 Hai Ba Trung street, Da Kao Ward, District 1. Ho Chi Minh City Tel. +(84) 8 22 20 2777 Fax +(84) 8 22 20 2776 marketing_vn@schaeffler.com

Schaeffler Vietnam Co., Ltd – Hanoi Sales Office MIPEC Tower, 12th Floor No. 229 Tay Son Street, Dong Da District Ha Noi Tel. +(84) 4 3556 0930 Fax +(84) 4 3556 0931 marketing_vn@schaeffler.com





Schaeffler Technologies AG & Co. KG

Industriestraße 1 – 3

91074 Herzogenaurach Germany Internet www.ina.com E-mail info.de@schaeffler.com In Germany:

in Germany:

Phone 0180 5003872 Fax 0180 5003873

From other countries:

Phone +49 9132 82-0 Fax +49 9132 82-4950

FAG

Schaeffler Technologies AG & Co. KG

Georg-Schäfer-Straße 30 97421 Schweinfurt Germany Internet www.fag.com E-mail faginfo@schaeffler.com

In Germany:

Phone 0180 5003872 Fax 0180 5003873 From other countries: Phone +49 9721 91-0 Fax +49 9721 91-3435 Every care has been taken to ensure the correctness of the information contained in this publication but no liability can be accepted for any errors or omissions.

We reserve the right to make technical changes.

© Schaeffler Technologies AG & Co. KG Issued: 2015, September

This publication or parts thereof may not be reproduced without our permission.